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# **Performance Criteria for Rollover Protective Structures (ROPS) for Construction, Earthmoving, Forestry, and Mining Machines—SAE J1040a**

**SAE Recommended Practice  
Editorial Change April 1976**

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**Society of Automotive Engineers, Inc.**  
400 COMMONWEALTH DRIVE, WARRENDALE, PA. 15096



**PREPRINT**

# PERFORMANCE CRITERIA FOR ROLLOVER PROTECTIVE STRUCTURES (ROPS) FOR CONSTRUCTION, EARTHMOVING, FORESTRY, AND MINING MACHINES—SAE J1040a

SAE Recommended Practice

Report of Construction and Industrial Machinery Technical Committee approved April 1974 and last revised February 1975. Editorial change April 1976. This document incorporates material formerly published as SAE J120, J194, J195, J196, and J1011.

**1. Objective** To establish a consistent, repeatable means of evaluating Force-Deflection and Vertical load carrying characteristics of Rollover protective structures under static loading and to prescribe performance requirements of a representative specimen under such loading. Rollover protective structures (ROPS) are structures whose primary purpose is to reduce the possibility of operators, who are wearing seat belts, being crushed if the machine they are operating should roll over.

## 2. Scope

**2.1** The following off-highway machines commonly used in earthmoving, construction, logging, and mining applications are included: pneumatic-tired agricultural machines and machines whose use is predominantly, or entirely, in manufacturing plants and/or warehouses are specifically excluded:

- (a) Track-type tractors of mass greater than 700 kg (1540 lb).
- (b) Pneumatic-tired, self-propelled motor graders of mass greater than 700 kg (1540 lb).
- (c) Pneumatic-tired front-end loaders, dozers, and their modifications used for rolling or compacting, and log skidders of mass greater than 700 kg (1540 lb).
- (d) Pneumatic-tired prime-movers of mass greater than 700 kg (1540 lb).
- (e) Pneumatic-tired, off-highway, nontrailed hauling units, with rear or side dump bodies, of mass greater than 700 kg (1540 lb).
- (f) Four-wheel drive, skid steer, front-end loaders of mass greater than 700 kg (1540 lb).
- (g) Industrial-type, wheeled front-end loaders and dozers of mass greater than 700 kg (1540 lb).
- (h) Rollers and compactors of mass greater than 2700 kg (5950 lb).

**2.2** The requirements are force resistance and energy absorption under horizontal loading, and vertical load carrying capability after removal of the horizontal load. (There are limitations on deflections under both horizontal and vertical loading.) Specific requirements derive from investigations on ROPS that have performed the intended function in a variety of actual rollovers, as well as analytical considerations based upon the compatibility of ROPS and the machine frame to which it attaches.

In addition to the loading requirements, there is a Temperature-Material requirement. This requires either making the static loadings with all structural members at, or below, a given temperature or making the loadings above the given temperature and constructing the ROPS of material that meets certain mechanical requirements.

It is intended that all portions of this Recommended Practice will be reviewed and revised as knowledge of ROPS performance, design, evaluation, and manufacture increases.

**2.3** Because this recommended practice presents both procedure and criteria that are intended for consideration as worldwide standards for ROPS, the following points are explicitly stated to aid in understanding its underlying principles, intention, and application.

**2.3.1** This evaluation procedure will not necessarily duplicate structural deformations due to a given actual roll.

**2.3.2** This evaluation procedure is generally destructive of the ROPS-machine assembly, as permanent deformation is apt to be induced in either or both.

**2.3.3** Although ROPS meeting these criteria may not give crush protection under all conceivable circumstances in which a machine could overturn, it is expected that crush protection will be assured under at least the following condition: An initial forward velocity of 0-16 km/h (0-10 mph) on a hard clay surface of 30 deg maximum slope, 360 deg of roll about the machine's longitudinal axis without losing contact with the slope.

**2.3.4** The horizontal force requirement and limitation on deflection (deflection limiting volume (DLV), see SAE J397a) are intended to assure that the ROPS will penetrate unfrozen soil, thereby giving a braking action to a roll.

**2.3.5** The energy requirement and limitations on deflection (DLV, see SAE J397a) under horizontal loading are intended to assure that the ROPS will deflect when it impacts a surface which will not significantly deform (frozen ground, concrete, rock), while retaining significant capability to withstand any subsequent impacts in an overturn.

**2.3.6** The vertical loading requirement is intended to assure that a deformed ROPS will be able to support the machine in an upside-down attitude.

**2.3.7** The Temperature-Material requirement is intended to assure that the ROPS will have meaningful resistance to brittle fracture. The material re-

quirement is the conventional Charpy V-notch evaluation; it is primarily a quality control check and the indicated temperature does not directly relate to operating conditions.

**2.3.8** The specific force, energy, and vertical load carrying requirements, including the general requirement of 8.2, are to be met, or exceeded, in the testing of a representative specimen. The Force and Energy requirements do not need to be attainable simultaneously; accordingly, one may be significantly exceeded before the other is attained in the horizontal loading. The limitation on deflections (DLV, J397a) is absolute; it cannot be exceeded in attaining the Force-Energy-Vertical loading requirements.

**2.3.9** Because, in an actual roll, loading will be dynamic (possibly impact), the use of "safety factors" which are based on elastic deformation, or ratio of the force attained to the force required, is not applicable. The "safety factor" of a ROPS is related more to energy absorption capability and details of weldment design and welding procedure than it is to either static force resistance or to avoiding permanent deformation. The static evaluation procedure of this Recommended Practice is itself a conservative element as it requires the representative specimen (ROPS, machine frame, and connecting members) to withstand a sustained loading that must be resisted by all components including those of the machine frame.

**2.3.10** A representative specimen is one that is within the manufacturer's specifications.

**3. Facilities and Instruments**—Facilities to secure the ROPS-machine frame assembly to the bedplate, as described below, and to apply the side and vertical loads are required. Typical, but not mandatory, installations are given for specific types of machines in accompanying illustrations.

Instrument systems used to measure mass, force, and deflection as follows:

Means to Measure	Accuracy
Vehicle mass	± 5% of max mass
Deflection of ROPS	± 5% of max deflection
Force on ROPS	± 5% of max force

The above percentages are nominal ratings of the accuracy of the instrumentation and should not be taken to indicate that compensating overtest is required.

## 4. Mass-Weight

**4.1** In keeping with the directive to include SI units in SAE technical reports (SAE J916), the distinction between mass and weight is emphasized by use of the mass unit (kilograms) in SI equations. In the gravimetric equations, the units lbf (pounds-force) and lb (pounds) are used. As common usage of the word "weight" to denote "mass" is not likely to cease with formal adoption of SI units, it seems desirable to emphasize that mass (as pounds or as kilograms) will continue to be mistakenly called "weight" for the foreseeable future. In fact, the greatest error possible in the weight-mass issue is 0.5%, which is small in relation to the ±5% allowable and any issue is one of semantics rather than engineering consequence.

**4.2** In section 8, the phrase "maximum recommended mass" is used. This refers to "clean machine" values. Soil, mud, rocks, limbs, debris, etc., that commonly adhere to or lay on machines in actual use should not be considered in determining the maximum recommended values.

**5. ROPS-Machine Assembly and Attachment to Bedplate**—The ROPS shall be attached to the machine frame as it would be on an operating machine. A complete machine is not required for the evaluation; however, the machine frame and ROPS mounting must represent an operating installation. All normally detachable windows, panels, doors, and other nonstructural elements shall be removed so they do not contribute to or detract from the structural evaluation. For horizontal loading, the ROPS-machine frame assembly shall be secured to the bedplate so that the members connecting the assembly and bedplate experience minimal deflection when the ROPS is side loaded. During side loading, the ROPS-machine frame assembly shall not receive any support from the bedplate, other than that due to the initial attachment.

The assembly shall be secured and/or modified so that any machine element that might be considered a suspension (rubber, gas, gas-oil, or mechanical spring) shall be effectively eliminated as an energy absorber. However, ROPS-to-machine frame mounts must not be altered; at the start of the evaluations, they shall be as they would be on an operating machine.

For the vertical loading, there is no limitation on securing or supporting the ROPS-machine frame assembly other than no repair or straightening of the assembly is permissible during or after side loading and during the vertical loading.

For editorial and printing convenience, specifics of attachment to bedplates are covered along with machine descriptions and ROPS requirements in section 8.

**6. Loading Procedure**

**6.1 Side Loading**

6.1.1 Load distribution devices may be used to prevent localized penetration.

6.1.2 Load shall be applied to major, upper, and longitudinal members except when a one- or two-post structure without cantilevered overhead shield is used. For that type of structure, load shall be applied in line with the upper cross member.

6.1.3 Initial loading of a one- or two-post frame with an overhead shield shall be dictated by the total longitudinal distance between major, upper ROPS members ( $L$ ) and the vertical projection of the front and rear planes of the DLV. The load point may not be within  $L/3$  from the one- or two-post frame. Should the  $L/3$  point be between the vertical projection of the DLV and the one- or two-post frame, the load point shall move away from the frame until it enters the vertical projection of the DLV (Fig. 1). Any load distribution plate used must not impede rotation of the ROPS around a vertical axis during the loading and may not distribute the load over a distance greater than 80% of  $L$ —Fig. 1.

6.1.4 For ROPS of more than two posts, loading shall not be distributed over more than 80% of the horizontal distance between the front and rear posts. Loading shall be within the 1000 mm (39.4 in.) that is defined by vertical projections of planes 80 mm (3.2 in.) outside of the front and rear planes of the DLV (Figs. 2-4).

6.1.5 The initial direction of loading shall be horizontal and perpendicular to a vertical plane through the machine's longitudinal centerline. As loading continues, the ROPS machine frame deformations may cause the direction of loading to change; this is permissible.

6.1.6 Should the operator's seat be off of the machine's longitudinal centerline, the loading shall be against the outermost side nearest the seat. For on-centerline seats, if mounting of the ROPS is such that different force-deflection relations are obtained from loading from left or right sides, the side loaded shall be that which will place the most severe requirements on the ROPS-machine assembly.

6.1.7 The rate of application of deflection (load) shall be such that it can be considered static. At deflection increments no greater than 1.3 mm (0.1 in.), at the point of application of the resultant load, force and deflection are to be recorded. This loading is to continue until the ROPS has achieved both the force and energy requirements. (See Fig. 5 for method of calculating energy.) The deflection(s) used in calculating energy is (are) to be that of the ROPS along the line(s) of action of the force(s). Any deflection of members used to support load application devices shall be deducted from total deflection.

6.2 Vertical Loading—After removal of the horizontal load, a vertical load shall be applied to the top of the ROPS. There are no limitations on the manner of distributing this load on the ROPS unless specifically indicated in

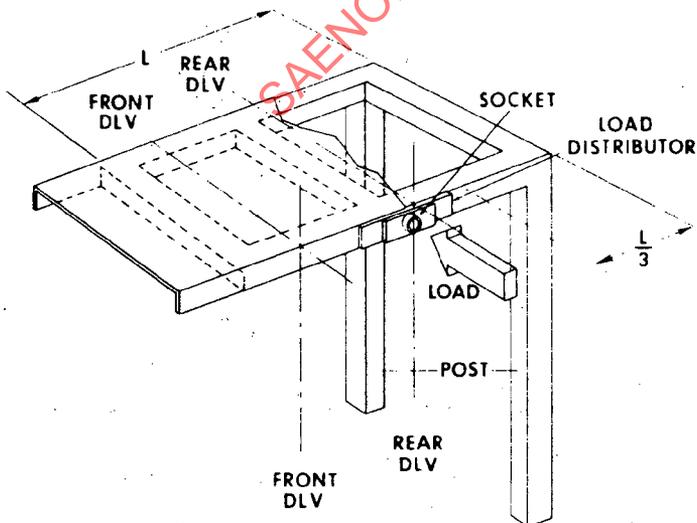


FIG. 1—LOAD DISTRIBUTOR AND SOCKET ARE TO PREVENT LOCAL PENETRATION AND TO HOLD END OF LOAD GENERATING DEVICE

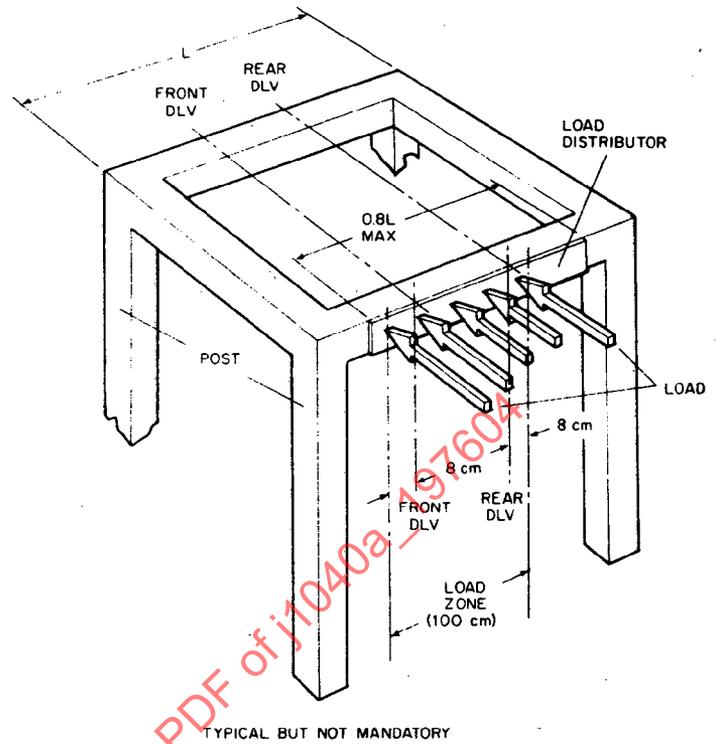


FIG. 2

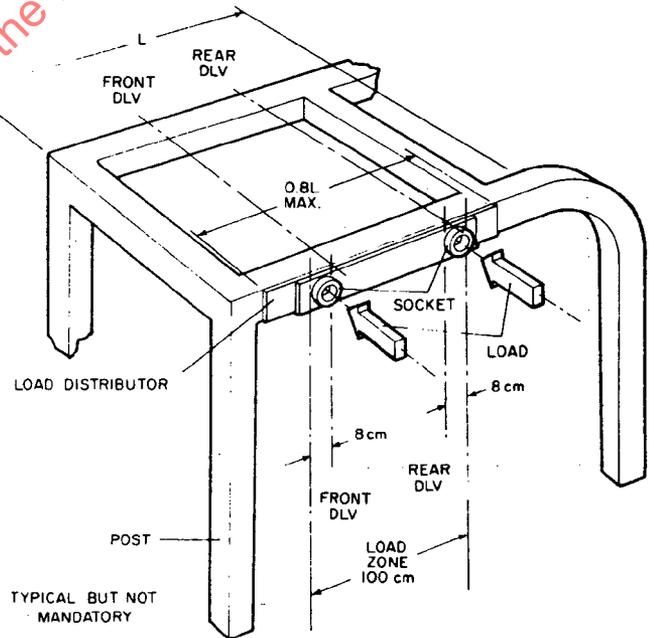


FIG. 3—LOAD DISTRIBUTOR AND SOCKET ARE TO PREVENT LOCAL PENETRATION AND TO HOLD END OF LOAD GENERATING DEVICE

section 8. Figs. 6-10 show typical vertical loadings. Note that Fig. 6 is loading by mass and the others loading by force. As requirements are mass based, it is necessary to relate force and mass. The correct relation is newtons = 9.80665 kg, 9.80665  $m/s^2$  being the gravitational acceleration at the standard conditions of sea level, 45 deg latitude (see SAE J916).

7. Temperature-Material Requirements—These requirements shall be met by either performing the horizontal and vertical loadings with all ROPS and machine frame members at  $-18^{\circ}C$  ( $0^{\circ}F$ ) or below, or performing the horizontal and vertical loadings at any higher temperature and meeting the following material requirements:

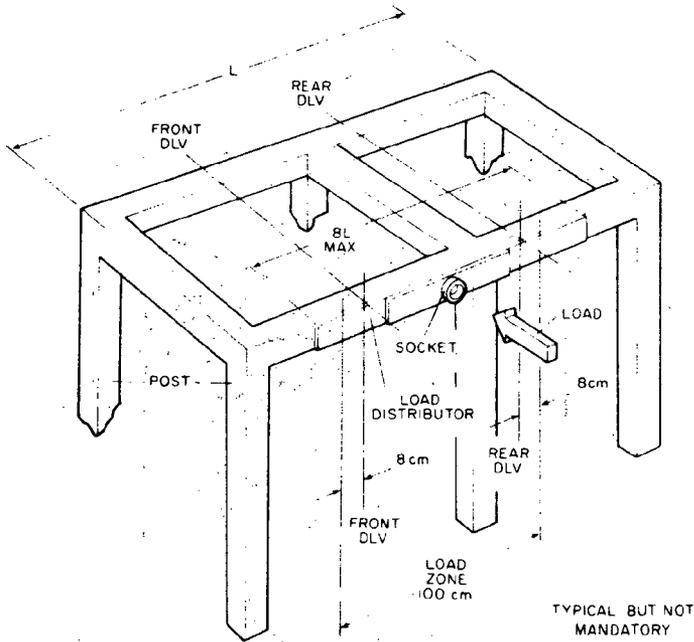


FIG. 4—LOAD DISTRIBUTOR AND SOCKET ARE TO PREVENT LOCAL PENETRATION AND TO HOLD END OF LOAD GENERATING DEVICE

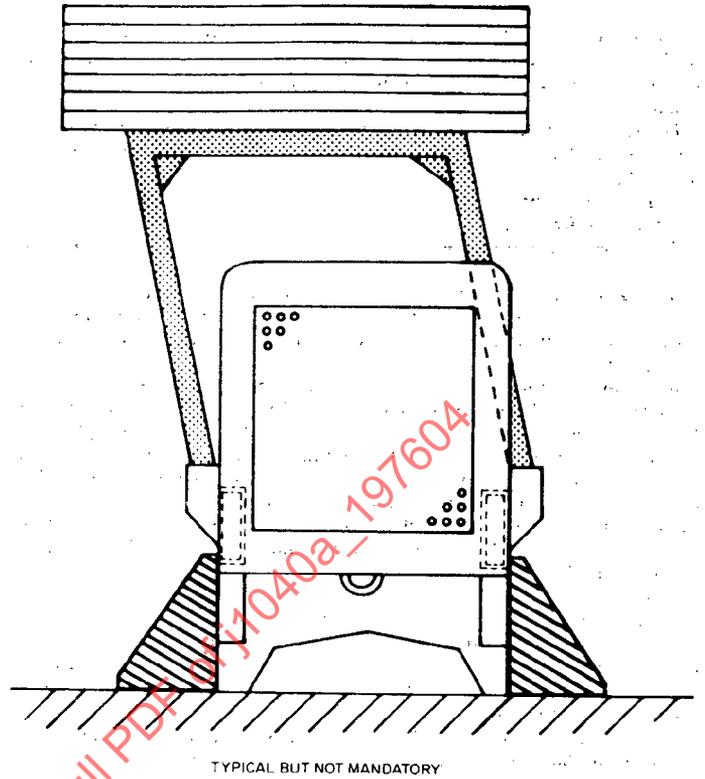
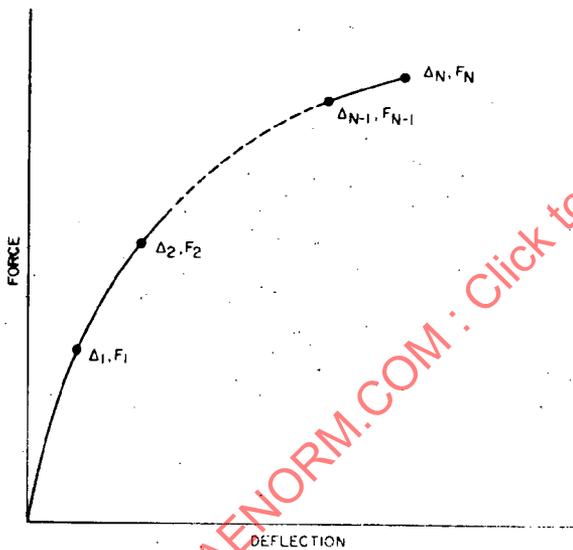


FIG. 6



$$\text{AREA} = \frac{\Delta_1 F_1}{2} + \left\{ \frac{\Delta_2 - \Delta_1}{2} \right\} \left[ \frac{F_1 + F_2}{2} \right] + \dots + \left\{ \frac{\Delta_N - \Delta_{N-1}}{2} \right\} \left[ \frac{F_{N-1} + F_N}{2} \right]$$

FIG. 5

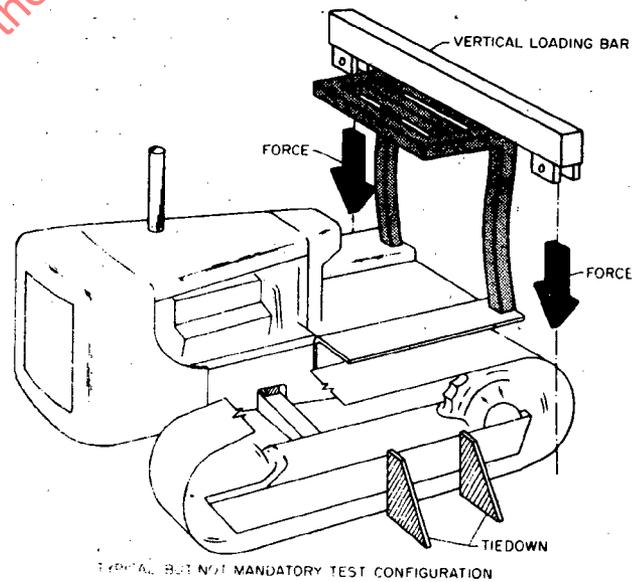


FIG. 7

7.1 Bolts and nuts used to attach the ROPS to the machine frame and to connect structural parts of the ROPS shall be SAE Grade 5 or B (SAE J129 and J995).

7.2 Structural members of the ROPS and the mounts which attach it to the machine frame shall be made of steels that have one of the Charpy V-notch impact strengths at  $-30^{\circ}\text{C}$  ( $-20^{\circ}\text{F}$ ) shown in Table 1.

Specimens are to be "longitudinal" and taken from flat stock, tubular, or structural sections before forming or welding for use in the ROPS. Specimens from tubular or structural sections are to be taken from the middle of the side of greatest dimensions, not to include welds.

SAE Information Report J1119 presents more data on specifics of CVN specimen size-test temperature interaction that meets the intent of the basic

requirements of Table 1. SAE J1119 also gives information on chemistries, manufacturing processes, and tensile properties of steel suitable for use in ROPS but only the CVN impact strengths are the mandatory requirement for the ROPS material option.

**8. Machine Description—Attachment to Bedplate—Force/Energy/Vertical Load Requirements**

8.1 Force-energy and vertical load requirements shall be met within the deflection(s) permitted by the Deflection Limiting Volume (DLV, SAE J397a) article. ( $M$  in the various equations denotes mass.)

8.2 In the horizontal loadings, if the force is attained before the energy, the force may decrease but must again attain the required level when the energy requirement is met or exceeded.

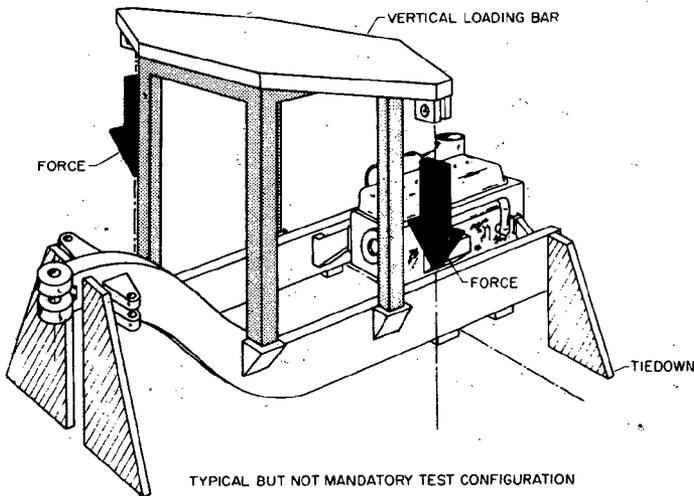


FIG. 8

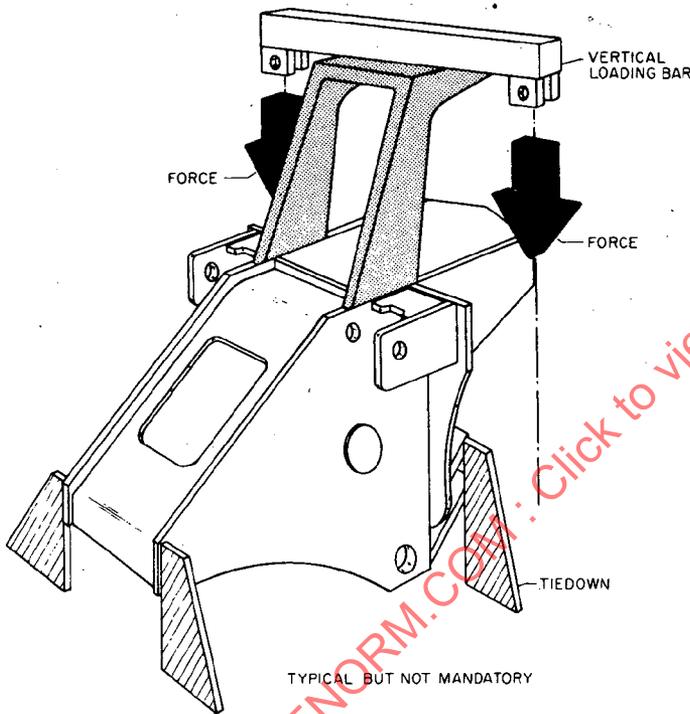


FIG. 9

**8.3 Track-Type Tractors, Without Sidebooms, Used Only for Bulldozing, Ripping, Pushloading, Front-End Loading, or Pulling a Drawbar Load**—See SAE J727b and J1057 (3.1 and 7.1) for description and nomenclature.

8.3.1 Connection to the bedplate shall be through the main housing and/or track frames (Figs. 11 and 12).

8.3.2 Requirements for machines whose mass is 60 000 kg (132 280 lb) or less are:

FORCE (side load):

$$F = 70\,000 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: N and kg}$$

$$F = 6090 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 13\,000 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: J and kg}$$

$$U = 42\,830 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: in-lbf and lb}$$

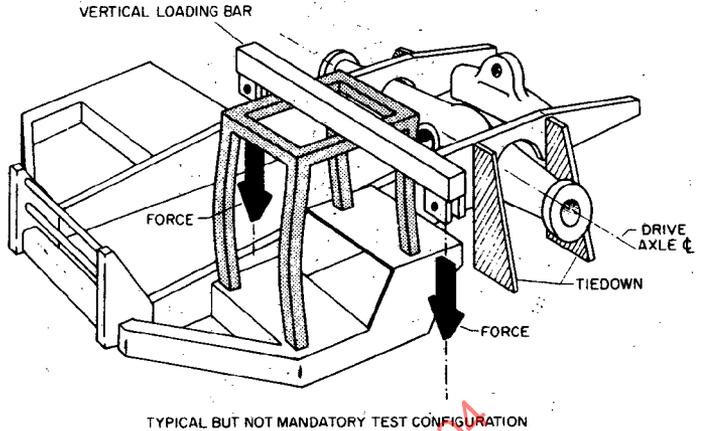


FIG. 10

TABLE 1—MINIMUM CHERRY V-NOTCH IMPACT STRENGTHS

Specimen Size, mm	J	ft-lbf
10 x 10 <sup>a</sup>	11.0	8.0
10 x 9	10.0	7.5
10 x 8	9.5	7.0
10 x 7.5 <sup>a</sup>	9.5	7.0
10 x 7	9.0	6.5
10 x 6.7	8.5	6.5
10 x 6	8.0	6.0
10 x 5 <sup>a</sup>	7.5	5.5
10 x 4	7.0	5.0
10 x 3.3	6.0	4.5
10 x 3	6.0	4.5
10 x 2.5 <sup>a</sup>	5.5	4.0

<sup>a</sup>Indicates preferred size. Specimen size shall be no less than the next preferred size that the material will permit. (Reference: ASTM A 370-68, Standard Methods and Definitions for Mechanical Testing of Steel Products.)

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.3.3 Requirements for machines whose mass is greater than 60 000 kg (132 280 lb) are:

FORCE (side load):

$$F = 420\,000 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: N and kg}$$

$$F = 80\,610 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 78\,000 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: J and kg}$$

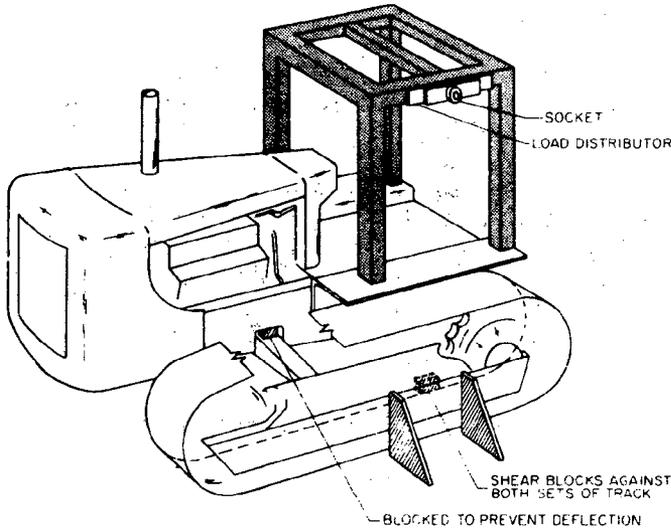
$$U = 566\,480 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: in-lbf and lb}$$

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.3.4 M in the above equations is the machine manufacturer's maximum recommended mass of the machine and its attachments, excluding towed equipment such as scrapers and rollers.

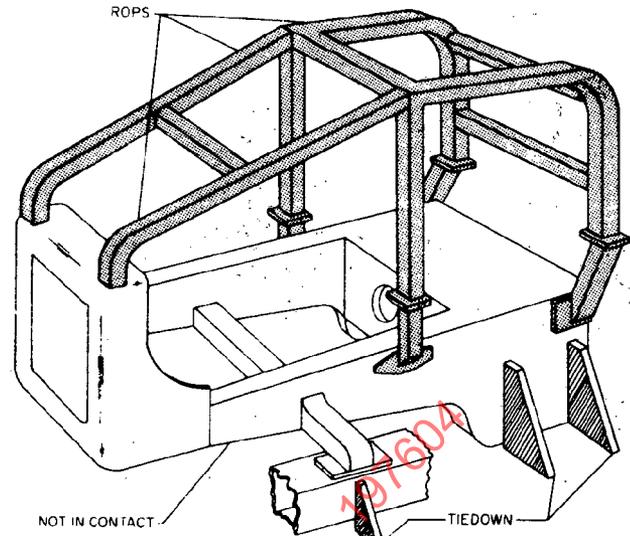
**8.4 Pneumatic-Tired, Self-Propelled Motor Graders**—See SAE J870 and J1057 (6) for description and nomenclature.

8.4.1 For nonarticulated machines and articulated machines using both frames, connections to the bedplate shall be directly from the machine frame at or near the front axle support and the rear drive support. For articulated machines, the hinge shall be locked if both frames are used in the evaluation; if only that frame to which the ROPS is mounted is used, the connections shall be at or near the extreme ends of the frame (Figs. 13-16).



TYPICAL-BUT NOT MANDATORY TEST CONFIGURATION

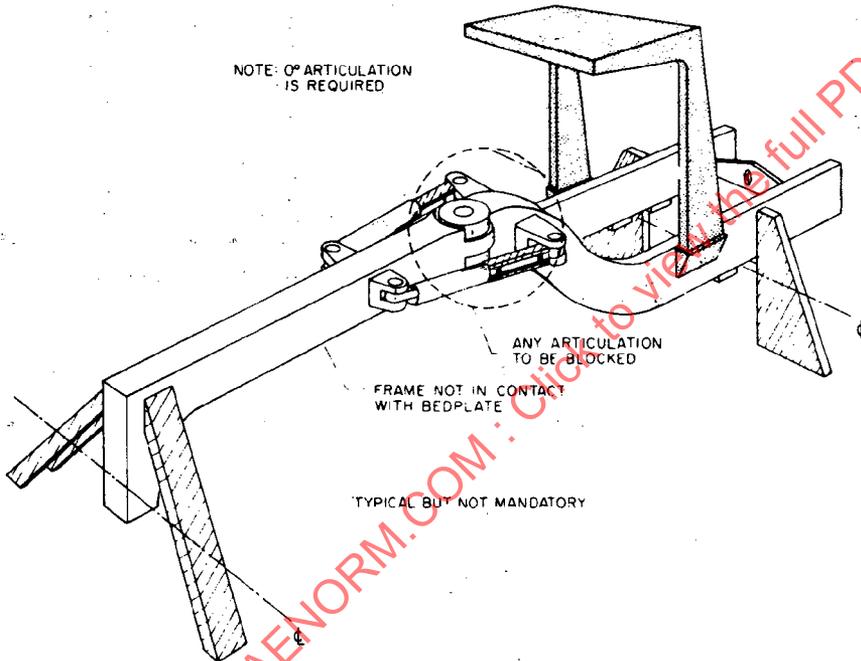
FIG. 11



TYPICAL BUT NOT MANDATORY CONFIGURATION

FIG. 12

NOTE: 0° ARTICULATION IS REQUIRED



TYPICAL BUT NOT MANDATORY

FIG. 13

8.4.2 Requirements for machines whose mass is 30 000 kg (66 140 lb) or less are:

FORCE (side load):

$$F = 70\,000 \left[ \frac{M}{10\,000} \right]^{1.10} \quad \text{units: N and kg}$$

$$F = 6600 \left[ \frac{M}{10\,000} \right]^{1.10} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 15\,000 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: J and kg}$$

$$U = 49\,410 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: in-lbf and lb}$$

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.4.3 Requirements for machines whose mass is greater than 30 000 kg (66 140 lb) are:

FORCE (side load):

$$F = 210\,000 \left[ \frac{M}{10\,000} \right]^{0.10} \quad \text{units: N and kg}$$

$$F = 43\,620 \left[ \frac{M}{10\,000} \right]^{0.10} \quad \text{units: lbf and lb}$$

ENERGY (side load):

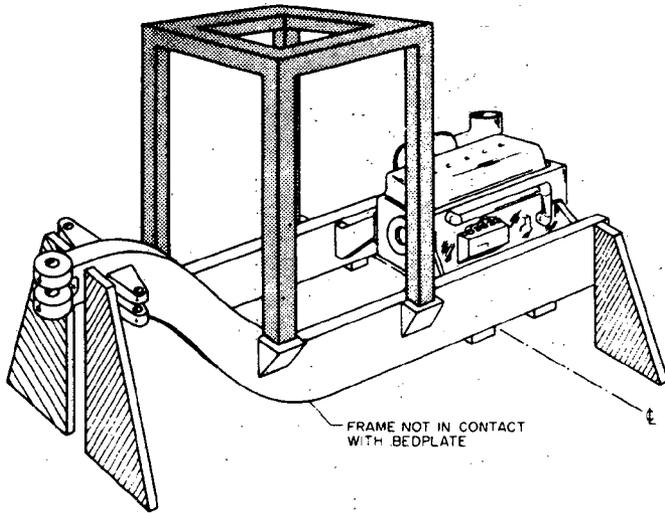
$$U = 45\,000 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: J and kg}$$

$$U = 326\,810 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: in-lbf and lb}$$

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.4.4 M in the above equations is the machine manufacturer's maximum recommended mass of the machine and its attachments excluding towed equipment.

8.5 Pneumatic-Tired Front-End Loaders, Dozers and Their Modifications Used for Rolling or Compacting, and Log Skidders—See SAE J1057 (3.2 and 7.2) for description and nomenclature.



TYPICAL BUT NOT MANDATORY TEST CONFIGURATION

FIG. 14

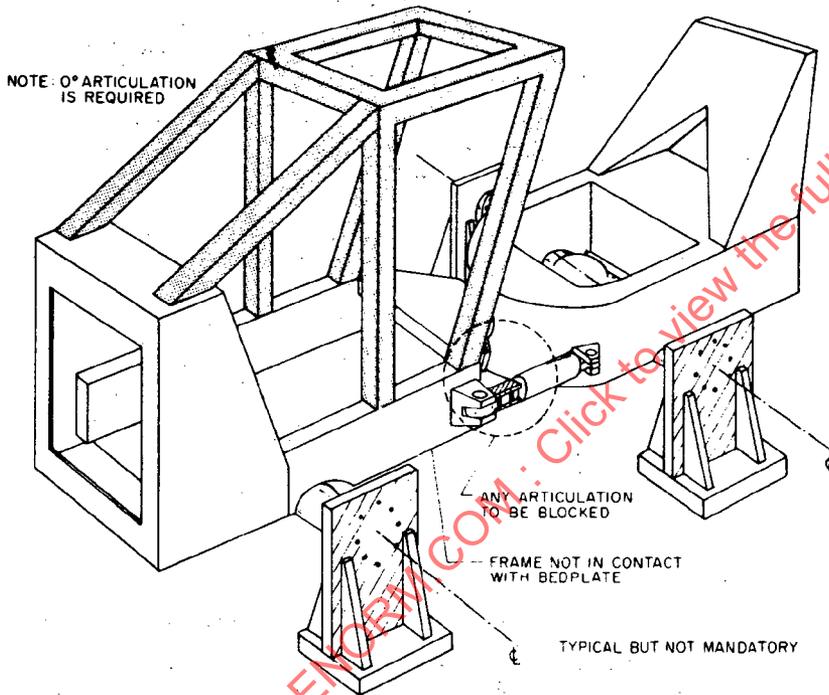


FIG. 15

8.5.1 For nonarticulated machines and articulated machines using both frames, connections to the bedplate shall be directly from the machine frame at or near the front axle support and the rear drive support. For articulated machines, the hinge shall be locked if both frames are used in the evaluation; if only that frame to which the ROPS is mounted is used, the connections shall be at or near the extreme ends of the frame (Figs. 13-16).

8.5.2 Requirements for machines whose mass is 60 000 kg (132 280 lb) or less are:

FORCE (side load):

$$F = 60\,000 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: N and kg}$$

$$F = 5220 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 12\,500 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: J and kg}$$

$$U = 41\,180 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: in-lbf and lb}$$

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.5.3 Requirements for machines whose mass is greater than 60 000 kg (132 280 lb) are:

FORCE (side load):

$$F = 360\,000 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: N and kg}$$

$$F = 69\,100 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 75\,000 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: J and kg}$$

$$U = 544\,690 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: in-lbf and lb}$$

VERTICAL: The ROPS-machine assembly shall support a vertical load of 1M after the side load is removed.

8.5.4 M in the above equations is the machine manufacturer's maximum recommended mass of the machine and its attachments, excluding towed equipment.

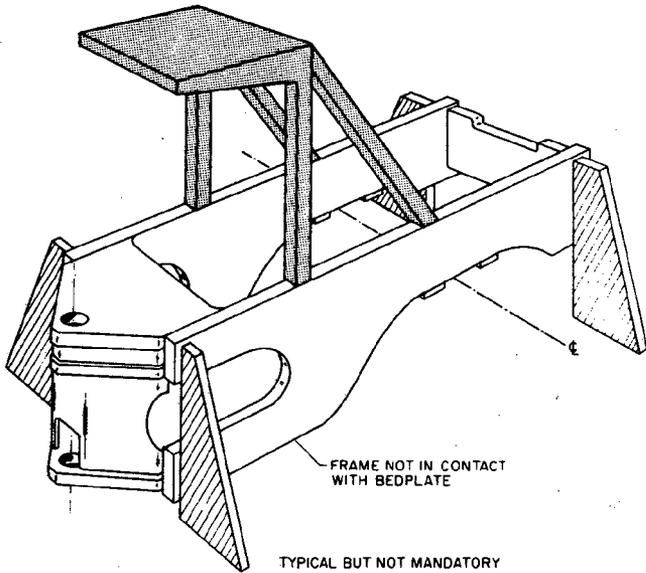


FIG. 16

**8.6 Pneumatic-Tired Prime Movers** of scrapers, water wagons, bottom dump wagons, side dump wagons, rear dump wagons, and towed fifth-wheel attachments—See SAE J869, J728a, and J1057 (4.1.1.4, 4.1.2.1, 4.2.1.1, 4.3.1.2, 4.3.1.3, 4.3.2.1, and 5) for description and nomenclature.

8.6.1 Connection to the bedplate shall be directly from the machine frame (or case) at or near the drive tire or axle connection. (For prime movers that are adaptations of another basic machine—that is, trucks, wheel tractors, etc.—the attachment shall be as specified under the ROPS criteria for the basic machine.) See Figs. 17 and 18.

8.6.2 Requirements for machines whose mass is 35 000 kg (77 160 lb) or less are:

FORCE (side load):

$$F = 95\,000 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: N and kg}$$

$$F = 8270 \left[ \frac{M}{10\,000} \right]^{1.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 20\,000 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: J and kg}$$

$$U = 65\,880 \left[ \frac{M}{10\,000} \right]^{1.25} \quad \text{units: in-lbf and lb}$$

**VERTICAL:** The ROPS-machine assembly shall support a vertical load of  $1M$  after the side load is removed.

8.6.3 Requirements for machines whose mass is greater than 35 000 kg (77 160 lb) are:

FORCE (side load):

$$F = 332\,500 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: N and kg}$$

$$F = 63\,820 \left[ \frac{M}{10\,000} \right]^{0.20} \quad \text{units: lbf and lb}$$

ENERGY (side load):

$$U = 70\,000 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: J and kg}$$

$$U = 508\,380 \left[ \frac{M}{10\,000} \right]^{0.25} \quad \text{units: in-lbf and lb}$$

**VERTICAL:** The ROPS-machine assembly shall support a vertical load of  $1M$  after the side load is removed.

8.6.4  $M$  in the above equations is the machine manufacturer's maximum recommended mass of the prime mover only—kingpins, hitches, and articu-

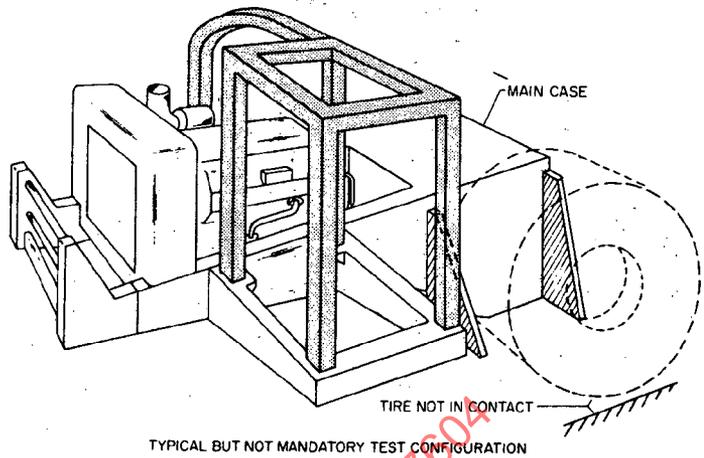


FIG. 17

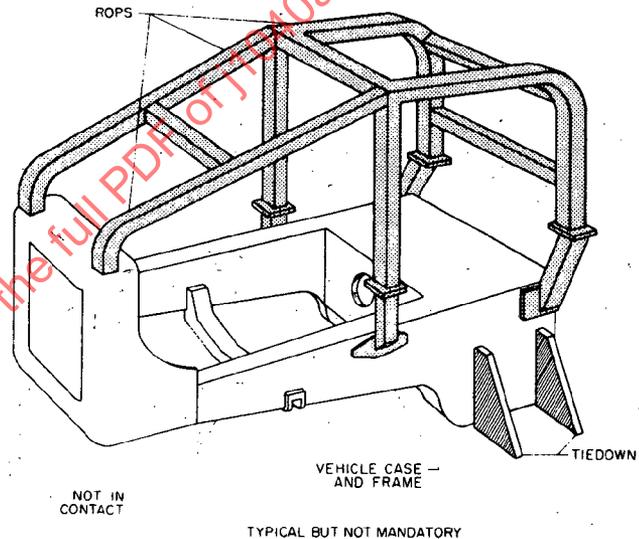


FIG. 18

lated steering components that attach to hitches or towed units are excluded from the mass of the prime mover.

**8.7 Pneumatic-Tired, Off-Highway, Nontrailed Hauling Units with Rear or Side Dump Bodies**—See SAE J1057 (4.1.1.1, 4.1.1.2, 4.1.1.3, 4.1.1.5, and 4.3.1.1) and J1016 for description and nomenclature.

8.7.1 Connections to the bedplate shall be at, or near, the front and rear axle locations. See Fig. 19.

8.7.2 Requirements can be met by any of the following options: ROPS only, body only, combination of ROPS and body.

8.7.3 For a ROPS only, side load force-energy requirements as indicated by the equations of paragraph 8.7.7 shall be met. After meeting these requirements, the ROPS shall support a vertical load of  $1M$  less the body tare mass.

8.7.4 For a body only, side load force-energy requirements indicated by the equations of paragraph 8.7.8 shall be met. After meeting these requirements, the body canopy shall support a vertical load of  $1M$ .

8.7.5 When both ROPS and body are used, the side loading Force and Energy requirements for each shall be 60% of those indicated by the equations of paragraphs 8.7.7 and 8.7.8, respectively. The vertical load requirements for both ROPS and body shall be as specified in paragraphs 8.7.3 and 8.7.4, respectively.

8.7.6 Side, or vertical, loading of the ROPS and/or body need not be applied simultaneously to both members of a combination. The only limitation on the order of the four loadings is that the vertical loading of members shall be after their side loading. Side load and vertical load on the truck body shall be applied within the length bounded by the vertical projections of the front and rear planes of the DLV (Fig. 20).