
**Road vehicles — Aerosol separator
performance test for internal
combustion engines —**

**Part 2:
Laboratory test method**

*Véhicules routiers — Essai de performance du séparateur d'aérosols
pour les moteurs à combustion interne —*

Partie 2: Méthode d'essai de laboratoire

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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see www.iso.org/directives).

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For an explanation on the voluntary nature of standards, the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the World Trade Organization (WTO) principles in the Technical Barriers to Trade (TBT) see the following URL: www.iso.org/iso/foreword.html.

This document was prepared by Technical Committee ISO/TC 22, *Road vehicles*, Subcommittee SC 34, *Propulsion, powertrain and powertrain fluids*.

A list of all parts in the ISO 17536 series can be found on the ISO website.

Introduction

The engine crankcase blowby is composed of combustion exhaust gases which have escaped to the crankcase via piston ring seals and lube oil aerosols generated by thermal and mechanical action within the engine. These gases are vented from the crankcase to prevent a build-up of high pressure. The constituents of vented engine blowby gases are recognized as an undesirable contaminant and technology for their containment is therefore evolving.

The device, used to separate oil aerosols from the blowby, typically releases cleaned gases to the atmosphere or alternatively returns the cleaned product to the combustion process by feeding into the engine air intake prior to the turbo compressor (if present). The latter has led to the requirement for a pressure control device to isolate the engine crankcase from air intake pressure.

It is the purpose of this document to define standardized and repeatable test procedures for the evaluation of blowby oil aerosol separators and filtering devices using this laboratory gravimetric test method.

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Road vehicles — Aerosol separator performance test for internal combustion engines —

Part 2: Laboratory test method

1 Scope

This document defines standardized and repeatable test procedures for the evaluation of blowby oil aerosol separators and filtering devices and specifies laboratory gravimetric separation efficiency and system pressure tests in both open and closed crankcase ventilation systems. This document has a limitation of 0 % to 99 % for aerosol gravimetric efficiency.

NOTE Gravimetric efficiencies >99 % may be difficult to measure due to long test durations and absolute filter weight measurements.

Filter life is not evaluated in this document.

This test method only applies to devices that have a defined tubular inlet, outlet and drain that can be connected to the test equipment. For devices that lack such connections, for example, one that is built into a valve cover, see [Annex A](#).

2 Normative references

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 17536-1:2015, *Road vehicles — Aerosol separator performance test for internal combustion engines — Part 1: General*

3 Terms and definitions

For the purposes of this document, the following terms and definitions apply.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- IEC Electropedia: available at <http://www.electropedia.org/>
- ISO Online browsing platform: available at <http://www.iso.org/obp>

3.1

standard flow

flow rate corrected to standard conditions

Note 1 to entry: See [5.3](#) for details.

4 Measurement accuracy

The measurement accuracy of this document shall be in accordance with ISO 17536-1:2015, Clause 3.

5 Test materials and test conditions

5.1 Test oil

The test oil to be used shall be an oil of such appropriate viscosity and surface tension that the particle size of 50 % cumulative mass of the generated aerosol exhibits more than 0,85 µm and less than 0,90 µm. The test oil shall meet the aerosol distribution by mass given in [Annex B](#). The challenge aerosol size distribution shall be plotted in [Figure C.1](#).

5.2 Absolute filter, wall flow trap and leakage

The provisions related to the absolute filter, the downstream wall flow trap and leakage shall be in accordance with ISO 17536-1:2015, Clause 4.

5.3 Standard conditions

The standard condition for temperature, humidity and pressure is 20 °C, 0 % RH and 101,3 kPa (1 013 mbar). Airflow differential pressure, inlet and outlet pressure and pressure loss shall be corrected to that standard condition.

5.4 Test temperature

5.4.1 Efficiency tests

The volume directly outside of the UUT and internal temperature of the efficiency test shall be either:

Condition A: 80 °C ± 3 °C

Condition B: 23 °C ± 5 °C

The condition that is run shall be documented in the test report (see [Table C.1](#)).

5.4.2 Differential pressure, pressure loss and crankcase pressure control tests

The flow rate for pressure loss and crankcase pressure control tests shall be corrected to standard flow. The pressure loss and crankcase pressure control tests shall be conducted with air entering the aerosol separator at a temperature of 23 °C ± 5 °C.

6 Test procedure

6.1 General

Performance tests shall be performed on a complete aerosol separator assembly. The tests shall consist of a pressure loss test, gravimetric efficiency test, conditioned gravimetric efficiency test, crankcase pressure control test (when a pressure regulator is present) and drain interval test (when applicable).

6.2 Test equipment

NOTE The definitions of the following terms related to the test equipment are defined in ISO 17536-1:2015, Clause 2: upstream particle counter, particle counter calibration, maximum particle concentration and particle counter flow.

6.2.1 Typical arrangements to determine the differential pressure or pressure loss to air flow, efficiency and crankcase pressure control are shown in [Annex D](#).

Use an aerosol generator which is capable of dosing oil mist over the range of delivery rates required according to the customers' specification.

The aerosol generator shall be validated as follows.

- Fill the aerosol generator to a pre-determined level.
- Simultaneously start the aerosol generator and timer.
- At a time interval relative to a mass oil flow of >1 g, determine the amount of aerosol dispersed and particle size distribution. Continue mass oil flow determinations of the aerosol until the desired oil flow deviates by <5 % and shall be >30 min. Continue feeding aerosol until the particle distribution does not meet the [Annex B](#) specification (to understand time capability to deliver a distribution as specified in [Annex B](#)).
- Adjust the aerosol generator until the average delivery rate is within ± 5 % of the desired rate and deviation in the delivery rate from the average is not more than ± 5 % for the entire designated test duration.
- After verifying the delivery rate, verify the aerosol delivered from the aerosol generator for the entire test duration is within the [Annex B](#) specifications.

6.2.2 An upstream wall flow trap should be used between the oil mist generator and the inlet tube to eliminate any oil wall flow to the inlet tube. Use a wall flow trap conforming to ISO 17536-1:2015, Annex I.

6.2.3 Use an inlet piezometer tube conforming to ISO 17536-1:2015, Figure B.2. The cross-section shall be the same as the aerosol separator inlet. In the case of non-uniform flow conditions caused by special inlet tubes, special precautions may be required.

6.2.4 Use a manometer or other differential pressure measuring device with the specified accuracy described in ISO 17536-1:2015, Clause 3.

6.2.5 Setup test with no UUT present, e.g. straight pipe.

6.2.6 A downstream wall flow trap should be used between the unit under test and the outlet piezometer tube described in [6.2.3](#) to eliminate any oil wall flow. Use a wall flow trap conforming to ISO 17536-1:2015, Annex I.

6.2.7 Use an outlet tube conforming to ISO 17536-1:2015, Figure B.2. The cross-section shall be the same as the aerosol separator outlet. In the case of non-uniform flow conditions caused by special inlet tubes, special precautions may be required.

6.2.8 Use an air flow rate measuring system having the accuracy described in ISO 17536-1:2015, Clause 3. The flow rate for differential pressure and crankcase pressure control tests shall be standard flow, which is the volume flow rate corrected to standard conditions, as specified in [5.3](#).

6.2.9 Use an air flow rate control system with a refresh rate greater than 2 Hz capable of maintaining the indicated flow rate to within 5 % of the selected value at a minimum data record frequency of 2 Hz during a steady-state and variable air flow operation.

6.2.10 Use a compressed air/blower/exhauster for controlling the air flow through the system, which has adequate flow rate and pressure characteristics for the oil separators to be tested.

6.2.11 If the unit under test has a pressure regulator or bypass, the use of a blower/exhauster on the downstream of the system can be used to regulate the pressure on the outlet of the unit under test. Devices with pressure regulators shall have air pushed through the inlet because the pressure regulator device regulates the amount of negative vacuum allowed on the system.

6.2.12 Grounding is required for all test apparatus to reduce the effects of static charges and to improve the consistency of the test results. Grounding of metallic and non-metallic surfaces, housings, transport tubes, injectors and associated hardware is recommended.

6.3 Pressure loss test

6.3.1 The purpose of this test is to determine the pressure loss across the unit under test which will result when air is passed through under predetermined conditions. Airflow differential pressure is measured with a clean aerosol separator at least four equally spaced air flows or agreed upon between the customer and supplier.

6.3.2 Set up the UUT as shown in ISO 17536-1:2015, Figure B.1, Figure D.1 or Figure D.3. Seal all joints to prevent air leaks. Connect the piezometer tubes to the inlet and outlet of the unit under test. The piezometer tube shall be sized to the size of the inlet and outlet of the UUT.

Care should be taken to understand the product components that may affect the flow path during a pressure loss test, e.g. pressure regulators.

6.3.3 Record the inlet temperature, barometric pressure and relative humidity.

6.3.4 Measure and record the differential pressure and upstream absolute pressure of the unit under test versus the air flow rate at a minimum of four equally spaced air flows or flow rates agreed upon between the customer and supplier.

6.3.5 Record the inlet temperature, barometric pressure and relative humidity.

6.3.6 Recorded differential pressure readings shall be corrected to standard conditions in accordance with [Annex E](#). See [Formulae \(E.4\)](#) and [\(E.5\)](#).

6.3.7 For pressure loss determination, use the formula given in ISO 17536-1:2015, Annex A.

6.3.8 Plot the pressure loss as shown in [Figure C.2](#) or equivalent.

6.4 Gravimetric efficiency test

6.4.1 The purpose of the gravimetric efficiency test is to determine the gravimetric separation efficiency of a device in two conditions:

- a) new state;
- b) conditioned state, as specified in [6.5](#).

The test duration for a gravimetric efficiency test shall be a minimum of 30 min and the minimum mass gained on the absolute filter shall be 0,1 g. Additional time may be needed to achieve the absolute filter weight gain requirement. The weight changes of the component parts and the absolute filter during the test duration are used to calculate the new and conditioned state gravimetric efficiency.

High efficiency separators shall not exceed 3 h for [6.4.1, a\)](#) as the new state is no longer maintained. For such separators, [6.5](#) and [6.6](#) shall be performed to complete an efficiency evaluation on the product and shall meet the above minimum requirements of 30 min and 0,1 g on the absolute filter.

NOTE The higher efficiency separators can require additional time to achieve the specified absolute filter weight gain requirement.

6.4.2 The mass oil flow is agreed upon by the user and manufacturer.

Care should be taken to understand that mass oil flow may affect the challenge aerosol size distribution.

6.4.3 Weigh and record the unit under test.

6.4.4 Weigh and record the drainage vessel (if present).

6.4.5 Weigh the absolute filter as specified in ISO 17536-1:2015, 4.1.2 and record the mass before assembly within the absolute filter housing.

6.4.6 Weigh the downstream wall flow trap of the unit under test as specified in ISO 17536-1:2015, 4.2.1.

6.4.7 Setup the test stand as shown in [Figure D.2](#) or [Figure D.4](#) for all aerosol separators. Seal all joints to prevent air leakage. The orientation of the unit under test shall be as in application.

Care should be taken to understand the product components that may affect the flow path during a pressure loss test, e.g. pressure regulators.

6.4.8 Record the UUT external air temperature, pressure and relative humidity.

6.4.9 Start the air flow through the test stand as specified in [5.4.1](#) and stabilize at the test flow as specified in [6.2.8](#). Record the differential pressure.

6.4.10 Set the feed rate to the pre-determined oil flow. Start the aerosol generator.

6.4.11 The differential pressure shall be compensated for the increased differential pressure that the tubing and downstream wall flow trap between the unit under test and the piezometer introduces, since the downstream wall flow trap will be in this area. The downstream wall flow trap is present to protect the downstream piezometer from contamination of liquid oil wall flow. The pressure loss of the downstream wall flow trap shall be subtracted from the overall pressure loss.

6.4.12 Every 10 min, record the differential pressure at the air test flow and the elapsed test time.

6.4.13 Record the differential pressure at the end of the test before interrupting either the air flow rate or mass oil flow to remove the absolute filter.

6.4.14 Stop the aerosol generator and continue to run the air flow rate for 15 s to 30 s as this will evacuate the test stand of aerosol.

6.4.15 Stop the air flow rate.

6.4.16 Record the UUT external air temperature, pressure and relative humidity.

6.4.17 Weigh the unit under test. Note any evidence of seal leakage or unusual conditions. The increase in mass of the unit under test is the mass measured minus the mass recorded in [6.4.3](#).

NOTE Carefully weigh all components so as not to lose any oil or mass.

6.4.18 Remove the absolute filter. Repeat [6.4.5](#) and determine the difference in mass. The change is the increase in mass of the absolute filter. The difference is the aerosol penetration of the unit under test. The minimum mass gained on the absolute filter shall be 0,1 g.

6.4.19 Reweigh the downstream wall flow trap. The increase in mass of the downstream wall flow trap is the mass minus the mass recorded in [6.4.6](#).

6.4.20 Reweigh the drainage vessel. The increase in mass of the drainage vessel is the mass minus the mass recorded in [6.4.4](#).

6.4.21 Calculate the aerosol efficiency, E_a , E_T and P_L , by using ISO 17536-1:2015, 5.3.2, Formulae (2), (3) and (4), respectively.

6.4.22 Report the results as shown in [Table C.2](#) or equivalent.

6.5 Conditioning of the separation device before a conditioned gravimetric efficiency test

6.5.1 After the gravimetric efficiency as specified in [6.4](#), condition the oil separator prior to running [6.6](#). The conditioning portion shall use aerosol as specified in [Annex B](#). The goal of this test is to condition the oil separator to a condition that is representing the majority of the time on an engine.

6.5.2 Perform the steps prescribed in [6.4.2](#) to [6.4.20](#) to run this test.

6.5.3 After introducing a total oil mass numerically in grams equal to at least 50 % of the volume of the media, verify that the unit under test is continuously draining. Once the unit under test is continuously draining, the pressure loss shall not change more than 100 Pa over a 3 h period.

6.5.4 Once the separation device has reached a conditioned state according to [6.5.3](#). Calculate the total mass challenged, Δm_T , found respectively in ISO 17536-1:2015, 5.3.2. Document the total amount of aerosol challenged as specified in ISO 17536-1 to the UUT during conditioning, both [6.4](#) and [6.5](#) in [Table C.3](#). Proceed to [6.6](#).

6.5.5 If the unit under test is an inertial separator, test using [6.4](#) until the oil drainage is observed. This efficiency shall be used to document the [6.6](#) condition for an inertial separator.

6.5.6 If the unit under test is a combination of separator technologies or methods, [6.5.1](#) to [6.5.4](#) shall be used to condition a combination separator.

6.6 Conditioned gravimetric efficiency test

6.6.1 The purpose of the gravimetric efficiency test is to determine the gravimetric separation efficiency of a device in two conditions as stated in [6.4.1](#).

The test duration for a gravimetric efficiency test shall be a minimum of 30 min and the minimum mass gained on the absolute filter shall be 0,1 g. Additional time may be needed to achieve the absolute filter weight gain requirement. The weight changes of the component parts and the absolute filter during the test duration are used to calculate the new and conditioned state gravimetric efficiency.

NOTE The higher efficiency separators may require additional time to achieve the specified absolute filter weight gain requirement.

6.6.2 Perform the steps prescribed in [6.4.2](#) to [6.4.20](#) to run this test

6.6.3 Calculate the aerosol efficiency, E_a , E_T and P_L , by using ISO 17536-1:2015, 5.3.2, Formulae (2), (3) and (4), respectively.

6.6.4 Report the results as shown in [Table C.3](#) or equivalent.

6.7 Crankcase pressure control test

6.7.1 The purpose of the test is to determine the relationship between the crankcase (inlet) and air intake (outlet) during a sweep of the outlet vacuum pressures. This test is to document the change in pressures as a function of the pressure regulator device. See [Table C.5](#).

6.7.2 Mount the aerosol separator system with a built-in pressure regulator or standalone pressure regulator to the test stand as specified in [Figure D.1](#) or [Figure D.3](#).

6.7.3 If the drain is external, plug the drain.

6.7.4 Record the inlet temperature, pressure and humidity.

6.7.5 Map the inlet and outlet pressure as a function of air flow rate as follows.

- a) Start the air flow through the test stand and stabilize at the test flow as specified in [6.2.8](#).
- b) Record the inlet pressure at the following outlet pressures: 0 to -8 kPa in 1 kPa increments. Pressure readings shall be corrected as specified in [5.3](#).

6.7.6 Repeat [6.7.5](#) for the flow rates at approximately 50 %, 100 %, 150 % and 200 % of rated air flow.

6.7.7 Record the results as shown in [Figure C.3](#) or equivalent.

6.8 Drain interval test

6.8.1 When operating closed crankcase ventilation system, the separator outlet is exposed to variable levels of vacuum resulting from air intake filter restriction. To prevent the reverse flow of oil through the separator devices oil return connection, the CCV system pressure differential, plus the head of collected oil, shall always exceed this outlet depression. In some cases, the solution to this situation is to fit a non-return valve to the oil return connection and allow only the intermittent drainage of the separated oil during low vacuum conditions.

It is the purpose of this procedure to determine and report the oil storage capacity of the type of separator device described above in order that a maximum oil drainage interval can be calculated for any given oil mass separation rate. The determined drainage interval shall be respected by allowing the CCV device to drain periodically while performing any validation tests within this document which include a challenge aerosol mass flow.

For tests performed in the laboratory method, the turbo and air intake interaction is replicated by a suction generated by the flow pump.

For this procedure, the oil storage capacity will be defined as the volume of oil which can be collected before the measurable separation performance of the device is reduced.

6.8.2 The oil flow rate is agreed upon by the user and manufacturer.

6.8.3 Weigh and record the unit under test.

6.8.4 Weigh and record the drainage vessel.

6.8.5 Setup the test stand as shown in [Figure D.2](#) and [Figure D.4](#) for all aerosol separators. Seal all joints to prevent air leakage. The aerosol separation device shown in [Figure D.2](#) and [Figure D.4](#) shall have all the drain components which are present on the engine, including check valves, orifices, actual lengths of tubing, etc. to accurately represent the product mounted drain configurations. The drainage vessel

shall only be used only to quantify the amount of oil contained in this test if the unit under test cannot be weighed accurately.

6.8.6 Record the inlet temperature, pressure and humidity.

6.8.7 Start the air flow through the test stand as specified in [5.4.1](#) and stabilize at the test flow as specified in [6.2.8](#). Record the differential pressure.

6.8.8 Set the feed rate to the pre-determined oil flow. Start the aerosol generator.

6.8.9 Continue feeding the challenge aerosol until the oil carryover measurement has reached the customer-specified limits by mass or visual signs of oil carryover are observed.

NOTE Clear tubing can be used to help facilitate the visual confirmation of oil carryover.

6.8.10 Stop the aerosol generator and continue to run the air flow rate for 15 s to 30 s as this will evacuate the test stand of aerosol.

6.8.11 Stop the air flow rate.

6.8.12 Record the inlet temperature, pressure and relative humidity.

6.8.13 Weigh the unit under test without losing any oil. Note any evidence of seal leakage or unusual conditions. Weigh the unit. The increase in mass of the unit under test is the mass minus the mass recorded in [6.8.3](#).

NOTE Carefully weigh all components so as not to lose any oil or mass.

6.8.14 If the unit under test cannot be weighed, drain the oil in the unit under test into the drainage vessel. Reweigh the drainage vessel. The increase in mass of the drainage vessel is the mass minus the mass recorded in [6.8.4](#).

6.8.15 Calculate the mass of oil storage capacity, M_s , by [Formula \(1\)](#):

$$M_s = \Delta m_u + \Delta m_d \quad (1)$$

Calculate the volume of oil storage capacity, V_s , by [Formula \(2\)](#):

$$V_s = \frac{M_s}{\rho} \quad (2)$$

Calculate the maximum oil drainage interval, I_s , by [Formula \(3\)](#):

$$I_s = \frac{M_s}{O_m} \times \rho \quad (3)$$

where

M_s is the oil storage capacity in mass (grams) of the unit under test;

V_s is the oil storage capacity in volume (millilitres) of the unit under test;

Δm_u is the mass increase of the unit under test;

- Δm_d is the mass increase of the drainage vessel;
- ρ is the density of the oil specified in [5.1](#);
- O_m is the mass oil flow of the unit under test per unit time.

6.8.16 Record the results as shown in [Table C.4](#) or equivalent.

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Annex A **(informative)**

Inlet geometry for equal oil flow challenge

History has seen crankcase ventilation systems involved in standalone systems, which have a separate inlet and outlet. These types of crankcase ventilation systems are easily tested as specified in this document since it has an inlet to connect the piezometers and system tubing. There is another style of crankcase ventilation systems, which are integrated into valve covers, gear covers or other locations on an engine which do not have a separate inlet. The inlet on these styles is in various shapes and sizes, typically not a shape that adheres to a standard round inlet and makes it hard to connect to a test stand.

With this test setup issue, the goal in testing these products is to verify that the setup does not affect the test results of the unit under test. Test fixtures have been developed to minimize the distance between the piezometer and the inlet of the part.

Correct pressure corrections should be examined prior to documenting the pressure loss of the system.

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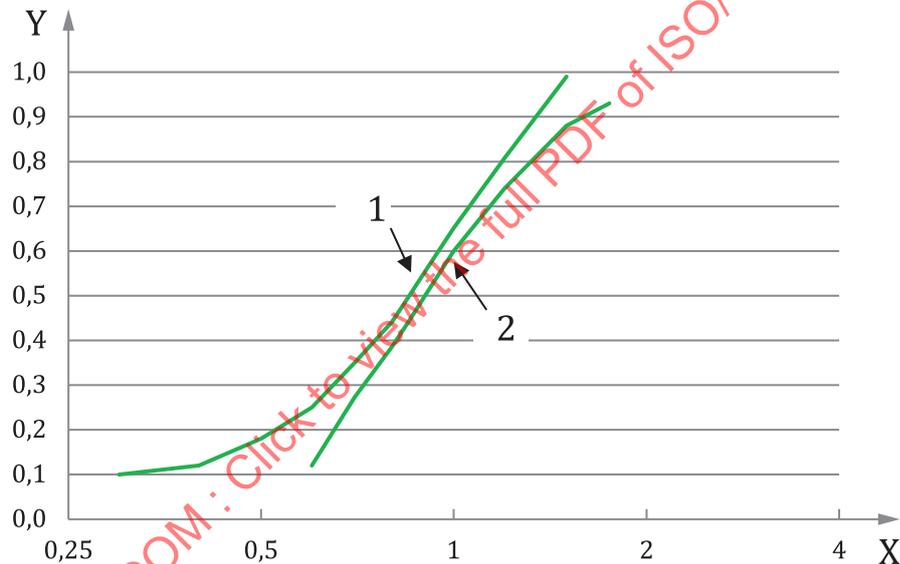
Annex B (normative)

Aerosol distribution by mass

The method to quantify the upstream distribution listed in [Annex B](#) shall be collected as specified in ISO 17536-1:2015, 5.2.2 at the test air flow, when measured using the light scattering technique calibrated according to the manufacturers' recommendations.

A graphical representation (see [Figure B.1](#)) and a table of values (see [Table B.1](#)) for the limiting curves are shown below.

NOTE The distribution curve shown in [Annex B](#) was developed using a sampling of actual engine distributions. The below distribution is near the middle of this sampling. The #1 and #2 limits were chosen to minimize the effect of the challenge aerosol's total mass on gravimetric efficiency numbers.



Key

- 1 upper limit curve
- 2 lower limit curve
- Y mass percent greater than the measured size ($\times 100$) (%)
- X particle size (micron)

Figure B.1 — Upper and lower limits on particle size distribution

Table B.1 — Figure B.1 data set

| Particle size (μm) | Cumulative size distribution (%) | |
|------------------------------------|-------------------------------------|-------------|
| | Lower limit | Upper limit |
| 0,3 | — | 6 |
| 0,4 | — | 9 |
| 0,5 | — | 16 |
| 0,6 | 15 | 24 |
| 0,7 | 27 | 34 |
| 0,8 | 39 | 44 |
| 0,85 | 44 | 50 |
| 0,9 | 50 | 55 |
| 1 | 60 | 65 |
| 1,2 | 74 | 81 |
| 1,5 | 88 | 99 |
| 1,75 | 93 | — |

NOTE 1 Cumulative size distribution in percentage is defined as the ratio of a fraction of aerosol underpass to the total amount of aerosol.

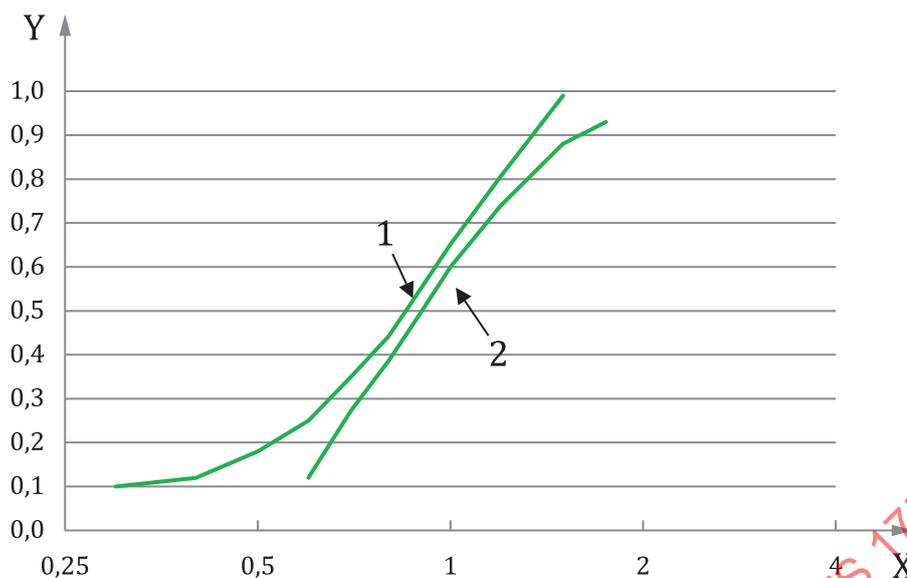
NOTE 2 When a cumulative distribution by mass in percentage is n %, the particle size is represented as D_n . D_{50} is the average diameter of the mass-based particle size distribution and 0,85 μm and 0,90 μm are given respectively for the lower limit and upper limit.

Annex C (informative)

Aerosol separator laboratory gravimetric test report

Table C.1 — Test unit, test materials and test equipment and test conditions

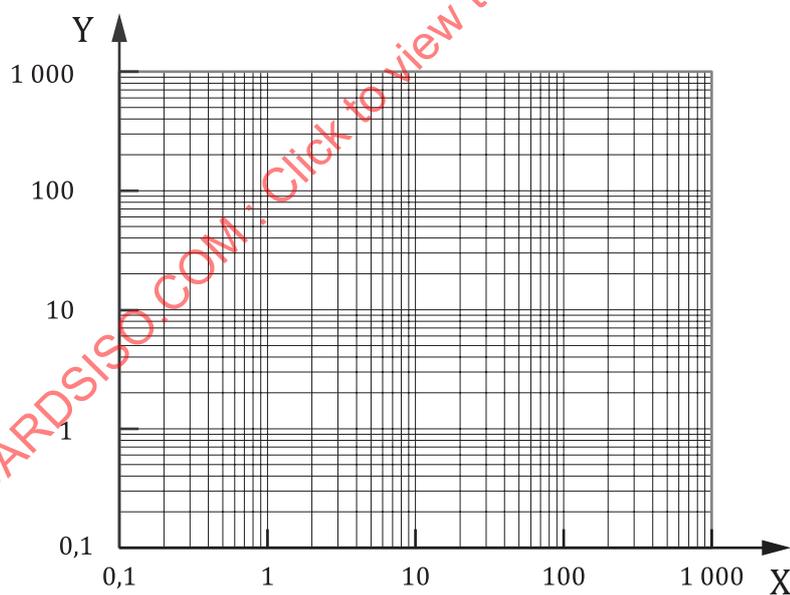
| Test report on aerosol separator laboratory gravimetric test according to ISO 17536-2 | | | | | | | |
|--|--|-----|--|------------------------------------|--|------|----|
| 1 Test unit | | | | | | | |
| Manufacturer: | | | | | | | |
| Model/type no.: | | | | | | | |
| Aerosol separator description: | | | | | | | |
| Internal volume of assembly or volume of filter elements: | | | | | | | |
| Round inlet diameter: | | mm | Non-round inlet: Dimension: | | | mm | |
| For non-round inlets; piezometer diameter: | | mm | Transition length: | | | mm | |
| Round outlet diameter: | | mm | Non-round outlet: Dimension: | | | mm | |
| For non-round outlets; piezom- eter diameter: | | mm | Transition length: | | | mm | |
| 2 Test materials and test equipment | | | | | | | |
| Test oil: | | | | | | | |
| Oil type: | | | | Batch no.: | | | |
| Oil viscosity at temperature: | | cSt | °C | Surface tension at temperature: | | mN/m | °C |
| Aerosol generator | | | | | | | |
| Make: | | | | Model: | | | |
| Dilution ratio: | | | | | | | |
| Particle counter: | | | | | | | |
| Make: | | | | Model: | | | |
| Flow rate: | | | | | | | |
| 3 Test conditions | | | | | | | |
| Test temperature for efficiency tests: | | | Condition A (80 °C) <input type="checkbox"/> | | Condition B (23 °C) <input type="checkbox"/> | | |
| Test terminal condition: | | | | | | | |
| Size distribution verification: See graph attached. | | | | | | | |



Key

- 1 upper limit curve
- 2 lower limit curve
- Y mass percent greater than the measured size ($\times 100$) (%)
- X particle size (micron)

Figure C.1 — Challenge aerosol size distribution



Key

- Y pressure loss (kPa)
- X flow rate (l/min)

Figure C.2 — Differential pressure versus flow rate

Table C.2 — Gravitational test results

| New state | | | | | | | | | | | |
|---------------|------------|-----------------------|---------------------------|----------------------|-----------------------|----------|---------------------------|------------------|-----------------------|----------|---|
| Test (sample) | Flow (lpm) | Measurement time (hr) | Before | | | | After | | | | |
| | | | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) | Efficiency type (See ISO 17536-1:2015, 5.3.2) |
| | | | | | | | | | | | |
| | | Unit weights | Unit under test (UUT) | Downstream wall trap | | Drain | Absolute filter | | | | |
| | | Before Test (g) | | | | | | | | | |
| | | After Test (g) | | | | | | | | | |

Table C.3 — Conditioned gravitational test results

| Conditioned state | | | | | | | | | | | |
|--------------------------------|------------|---|---------------------------|----------------------|-----------------------|----------|---------------------------|------------------|-----------------------|----------|---|
| Volume of media in sample (CC) | | Mass of aerosol subjected to part combined from tests 6.4 and 6.5 (grams) | | | | | | | | | |
| Test (sample) | Flow (lpm) | Measurement time (hr) | Before | | | | After | | | | |
| | | | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) | Efficiency type (See ISO 17536-1:2015, 5.3.2) |
| | | | | | | | | | | | |
| | | Unit weights | Unit under test (UUT) | Downstream wall trap | | Drain | Absolute filter | | | | |
| | | Before Test (g) | | | | | | | | | |
| | | After Test (g) | | | | | | | | | |

Table C.4 — Drain interval test results

| Drain interval test | | | | | | | | | | |
|----------------------------|------------|-----------------------|---------------------------|--|-----------------------|----------|---------------------------|------------------|-----------------------|----------|
| Mass of oil storage (g) | | | | Maximum mass oil drainage interval (g) | | | | | | |
| Volume of oil storage (cc) | | | | | | | | | | |
| Test (sample) | Flow (lpm) | Measurement time (hr) | Before | | | | After | | | |
| | | | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) | Barometric pressure (kPa) | Temperature (°C) | Relative humidity (%) | Dp (kPa) |
| | | | | | | | | | | |
| | | Unit weights | Unit under test (UUT) | | | Drain | | | | |
| | | Before Test (g) | | | | | | | | |
| | | After Test (g) | | | | | | | | |

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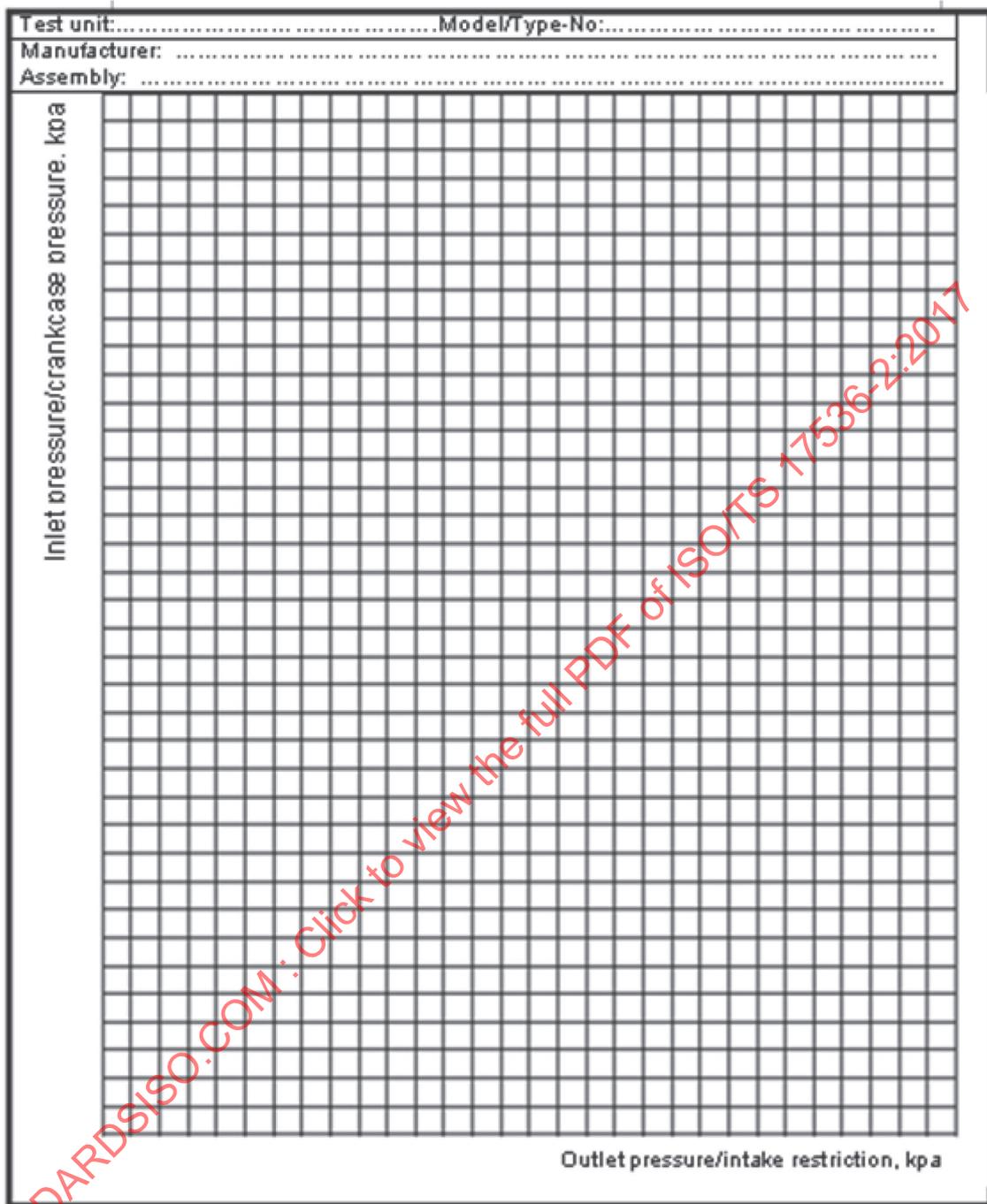


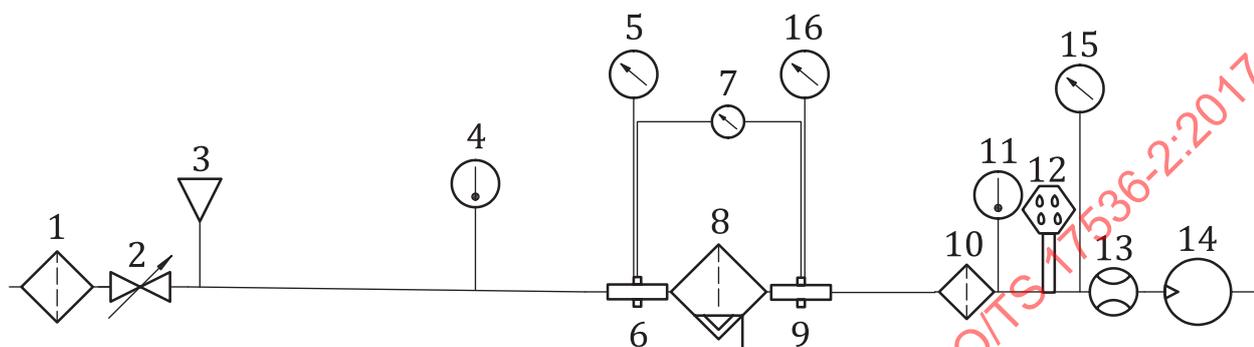
Figure C.3 — Crankcase inlet pressure versus intake outlet pressure graph

Table C.5 — Crankcase inlet pressure versus intake outlet pressure data grid

| Flow rate (l/min) | Outlet pressure (kPa) | | | | | | | | |
|----------------------|-----------------------|---|---|---|---|---|---|---|---|
| | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 |
| Flow #1 | | | | | | | | | |
| Flow #2 | | | | | | | | | |
| Flow #3 | | | | | | | | | |
| Flow #4 | | | | | | | | | |

Annex D (normative)

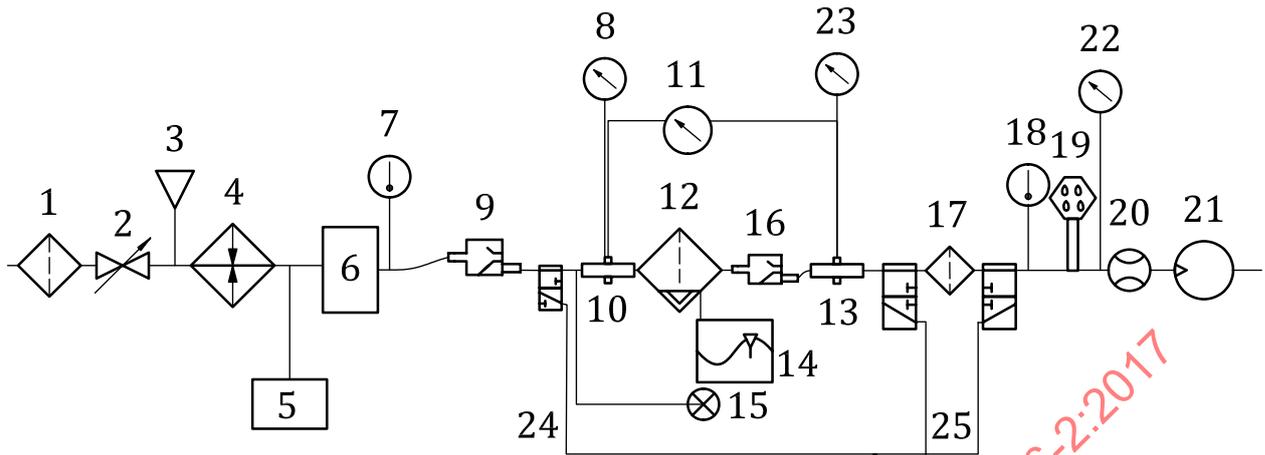
Test equipment



Key

- | | | | |
|---|------------------------------|----|------------------------------|
| 1 | HEPA inlet filter | 9 | downstream piezometer |
| 2 | HEPA inlet valve | 10 | absolute filter, if needed |
| 3 | pump/air supply | 11 | flow meter temperature |
| 4 | temperature probe, if needed | 12 | flow meter humidity |
| 5 | upstream pressure | 13 | flow meter |
| 6 | upstream piezometer | 14 | pump |
| 7 | differential pressure device | 15 | flow meter absolute pressure |
| 8 | aerosol separation device | 16 | downstream pressure |

Figure D.1 — Aerosol separator laboratory pressure loss and crankcase pressure control test set-up, flow meter downstream

**Key**

- | | | | |
|----|-----------------------------------|----|---|
| 1 | HEPA inlet filter | 14 | drainage vessel for mass analysis |
| 2 | HEPA inlet valve | 15 | particle counter |
| 3 | pump/air supply | 16 | downstream wall flow trap, optional |
| 4 | heater | 17 | absolute filter |
| 5 | aerosol generator | 18 | flow meter temperature |
| 6 | mixing area | 19 | flow meter humidity |
| 7 | temperature probe, if needed | 20 | flow meter |
| 8 | upstream pressure | 21 | pump |
| 9 | upstream wall flow trap, optional | 22 | flow meter absolute pressure |
| 10 | upstream piezometer | 23 | downstream pressure |
| 11 | differential pressure device | 24 | bypass around UUT, optional |
| 12 | aerosol separation device | 25 | bypass around absolute filter, optional |
| 13 | downstream piezometer | | |

Figure D.2 — Aerosol separator laboratory gravitational test set-up, flow meter downstream