

# ISO

INTERNATIONAL ORGANIZATION FOR STANDARDIZATION

## ISO RECOMMENDATION R 1604

ENDLESS WIDE V-BELTS FOR INDUSTRIAL SPEED-CHANGERS  
AND GROOVE PROFILES FOR CORRESPONDING PULLEYS

1st EDITION

October 1970

COPYRIGHT RESERVED

The copyright of ISO Recommendations and ISO Standards belongs to ISO Member Bodies. Reproduction of these documents, in any country, may be authorized therefore only by the national standards organization of that country, being a member of ISO.

For each individual country the only valid standard is the national standard of that country.

Printed in Switzerland

Also issued in French and Russian. Copies to be obtained through the national standards organizations.

## BRIEF HISTORY

The ISO Recommendation R 1604, *Endless wide V-belts for industrial speed-changers and groove profiles for corresponding pulleys*, was drawn up by Technical Committee ISO/TC 41, *Pulleys and belts (including vee-belts)*, the Secretariat of which is held by the Association Française de Normalisation (AFNOR).

Work on this question led to the adoption of Draft ISO Recommendation No. 1604 which was circulated to all the ISO Member Bodies for enquiry in May 1968. It was approved, subject to a few modifications of an editorial nature, by the following Member Bodies :

Austria	Israel	Sweden
Belgium	Italy	Switzerland
Brazil	Netherlands	Thailand
Czechoslovakia	New Zealand	Turkey
Denmark	Peru	U.A.R.
Finland	Portugal	United Kingdom
France	South Africa, Rep. of	U.S.S.R.
India	Spain	Yugoslavia

The following Member Body opposed the approval of the Draft :

Norway

This Draft ISO Recommendation was then submitted by correspondence to the ISO Council, which decided to accept it as an ISO RECOMMENDATION.

## ENDLESS WIDE V-BELTS FOR INDUSTRIAL SPEED-CHANGERS AND GROOVE PROFILES FOR CORRESPONDING PULLEYS

### 1. SCOPE

This ISO Recommendation specifies the principal dimensions of endless wide V-belts for *industrial speed-changers\**, as well as the groove profile of corresponding fixed or variable diameter pulleys.

It does not concern speed-changer belts used either for the propulsion of self-propelled vehicles (motor-cycles, scooters, cars) or as parts of farm-machines (chiefly harvesters).

The wide V-belts are characterized by a "relative height" (ratio of the nominal height of the theoretical profile to its pitch width) of about 0.32.

### 2. DIMENSIONS AND TOLERANCES

#### 2.1 Belt dimensions

The dimensions and the tolerances of the belts are given in millimetres in Table 1 and in inches in Table 2.

2.1.1 *Designation*. The proposed nine cross-sections are designated as follows :

— W 16 — W 20 — W 25 — W 31.5 — W 40 — W 50 — W 63 — W 80 — W 100

2.1.2 *Width*. Values for the *pitch width*  $l_p$  (expressed in millimetres) are nine consecutive terms from the R 10 series of preferred numbers.

Should this series of values be considered insufficient, it may be completed, outside the limits, with other terms from the R 10 series of preferred numbers.

\* Due to the need for compactness of such types of speed-changers, it is necessary to minimize pitch diameters. At the present level of techniques, the values given for the minimum pitch diameter should be used on the notched V-belts.

2.1.3 *Belt height.* The nominal belt height  $T$ , equal to  $0.32 l_p$ , is divided as follows on both sides of the pitch line :

- above ( $B$ ) =  $0.08 l_p$
- below ( $H$ ) =  $0.24 l_p$

TOLERANCES

(a) Belt ride out, in the conditions specified in section 3, should not exceed the value shown in the following table (round value of the expression  $0.36 \sqrt{l_p}$ , where  $l_p$  is expressed in millimetres).

Designation of the sections	W 16	W 20	W 25	W 31.5	W 40	W 50	W 63	W 80	W 100
mm	1.2	1.8	1.8	1.8	2.4	2.4	3.0	3.0	3.6
in	0.047	0.071	0.071	0.071	0.094	0.094	0.118	0.118	0.142

(b) Height  $H$  should not exceed the value shown in the following table (round value of the expression  $0.24 l_p + 0.06 \sqrt{l_p}$ , where  $l_p$  is expressed in millimetres).

Designation of the sections	W 16	W 20	W 25	W 31.5	W 40	W 50	W 63	W 80	W 100
mm	4.0	5.1	6.3	7.8	10.0	12.4	15.5	19.7	24.6
in	0.157	0.201	0.248	0.307	0.394	0.488	0.610	0.776	0.969

NOTE. - The profile of the cross section of the free strand of the belt under tension is not imposed; in particular the angle of the sidewalls and their profile are left to the initiative of the manufacturers.

2.1.4 *Length.* Values for the *pitch length*  $l_p$  are multiples of the pitch width figures, the multiplier coefficients being eight consecutive terms (28 to 63 inclusive) from the R 20 series of preferred numbers.

Should the above range of lengths be considered insufficient, it may be completed

- upwards or downwards, with other terms from the R 20 series of preferred numbers;
- exceptionally, between two consecutive lengths from the table, with terms from the R 40 series of preferred numbers (especially for box-type speed-changers).

TABLE 1 - Belt dimensions in millimetres

Designation of the sections	W 16		W 20		W 25		W 31.5		W 40		W 50		W 63		W 80		W 100	
	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$
Approximate topwidth	16.6	20.7	20.7	25.9	32.6	41.5	51.8	65.3	82.9	103.7								
Pitchwidth $l_p$	16	20	25	31.5	40	50	63	80	100									
Belt height	Above the pitch line	1.3	1.6	2	2.5	3.2	4	5	6.3	8								
	Below the pitch line	3.8	4.8	6	7.5	9.6	12	15	19.2	24								
Total	5.1	6.4	8	10	12.8	16	20	25.5	32									
Belt length ( $L_e$ = Approximate outside length $L_p$ = Pitch length)	$L_e$	458	570	722	916	1140	1425	1832	2280	2850	3550	4550	5050	5650	6350			
	$L_p$	450 ± 10	560 ± 12	710 ± 14	900 ± 18	1120 ± 22	1400 ± 28	1800 ± 36	2240 ± 44	2850 ± 56	3550 ± 70	4550 ± 90	5050 ± 100	5650 ± 110	6350 ± 120			
	$L_e$	508	640	812	1016	1270	1625	2032	2540	3200	4000	5000	6000	7000	8000			
	$L_p$	560 ± 12	710 ± 14	900 ± 18	1120 ± 22	1420	1825	2272	2840	3600	4500	5500	6500	7500	8500			
	$L_e$	638	810	1012	1266	1620	2025	2532	3190	4050	5050	6050	7050	8050				
	$L_p$	710 ± 14	900 ± 18	1120 ± 22	1416	1820	2240 ± 44	2832	3590	4550	5550	6550	7550	8550				
	$L_e$	808	1010	1262	1616	2020	2525	3182	4040	5050	6050	7050	8050	9050				
	$L_p$	900 ± 18	1120 ± 22	1412	1816	2260	2825	3582	4540	5550	6550	7550	8550	9550				
$L_e$	1008	1260	1612	2016	2520	3175	4032	5040	6050	7050	8050	9050	10050					
$L_p$	1000 ± 20	1250 ± 24	1600 ± 32	2016	2520 ± 50	3175 ± 62	4032 ± 70	5040 ± 80	6050 ± 90	7050 ± 100	8050 ± 110	9050 ± 120	10050 ± 130					

STANDARDSISO.COM · Click to view the full PDF of ISO/R 1604:1970

TABLE 2 — Belt dimensions in inches

Designation of the sections	W 16		W 20		W 25		W 31.5		W 40		W 50		W 63		W 80		W 100	
	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$
Approximate topwidth	0.654	0.815	0.787	0.984	1.020	1.283	1.634	2.039	2.571	3.264	4.083							
Pitchwidth $L_p$	0.630	0.787	0.063	0.079	0.098	0.126	0.157	0.197	0.248	0.315	0.397							
Belt height	Above the pitch line		0.189		0.236		0.295		0.378		0.472		0.591		0.756		0.945	
	Below the pitch line		0.150		0.236		0.295		0.378		0.472		0.591		0.756		0.945	
Total $T$	0.201	0.252	0.315	0.394	0.504	0.630	0.787	1.004	1.26									
Belt length  ( $L_e$ = Approximate outside length $L_p$ = Pitch length)	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$	$L_e$	$L_p$
	18.03	17.72 ± 0.394	22.44	22.05 ± 0.472	28.43	27.95 ± 0.051	36.06	35.43 ± 0.709	44.88	44.09 ± 0.866	56.10	55.12 ± 1.102	72.13	70.87 ± 1.417	89.76	88.19 ± 1.732	112.20	110.24 ± 2.205
	20	19.69 ± 0.394	25.20	24.80 ± 0.472	31.97	31.50 ± 0.630	40.00	39.37 ± 0.787	50	49.21 ± 0.945	63.98	62.99 ± 1.260	80	78.74 ± 1.575	100	98.43 ± 1.969	125.98	124.02 ± 2.441
	22.36	22.05 ± 0.472	28.35	27.95 ± 0.551	35.91	35.43 ± 0.709	44.72	44.09 ± 0.866	55.91	55.12 ± 1.102	71.85	70.87 ± 1.417	89.45	88.19 ± 1.732	111.81	110.24 ± 2.205	141.73	139.76 ± 2.756
	25.12	24.80 ± 0.472	31.89	31.50 ± 0.630	39.84	39.37 ± 0.787	49.84	49.21 ± 0.945	63.78	62.99 ± 1.260	79.72	78.74 ± 1.575	99.69	98.43 ± 1.969	125.59	124.02 ± 2.441	159.45	157.48 ± 3.150
	28.27	27.95 ± 0.551	35.83	35.43 ± 0.709	44.57	44.09 ± 0.866	55.75	55.12 ± 1.102	71.65	70.87 ± 1.417	89.07	88.19 ± 1.732	111.50	110.24 ± 2.205	141.34	139.76 ± 2.756	179.13	177.17 ± 3.543
	31.81	31.50 ± 0.630	39.76	39.37 ± 0.787	49.69	49.21 ± 0.945	63.62	62.99 ± 1.260	79.53	78.74 ± 1.575	99.41	98.43 ± 1.969	125.28	124.02 ± 2.441	159.06	157.48 ± 3.150	198.82	196.85 ± 3.937
	35.75	35.43 ± 0.709	44.49	44.09 ± 0.866	55.59	55.12 ± 1.102	71.50	70.87 ± 1.417	88.98	88.19 ± 1.732	111.22	110.24 ± 2.205	141.02	139.76 ± 2.756	178.74	177.17 ± 3.543	222.44	220.47 ± 4.331
	39.69	39.37 ± 0.787	49.61	49.21 ± 0.945	63.46	62.99 ± 1.260	79.37	78.74 ± 1.575	99.21	98.43 ± 1.969	125.00	124.02 ± 2.441	158.74	157.48 ± 3.150	198.43	196.85 ± 3.937	250	248.03 ± 4.724

STANDARDSISO.COM: Click to view PDF file: ISO/R 1604:1970

Approximate outside lengths  $L_e$  are computed by adding to pitch lengths the following values :

Designation of the sections	W 16	W 20	W 25	W 31.5	W 40	W 50	W 63	W 80	W 100
mm	8	10	12	16	20	25	32	40	50
in	0.31	0.39	0.47	0.63	0.79	0.98	1.26	1.57	1.97

TOLERANCE. The permissible deviation on the length shown in Table 1 is about  $\pm 2\%$ .

Box-type speed changers need a closer tolerance, which can be complied with by printing a conventional length symbol on the external face of the belt after checking.

## 2.2 Profile dimensions of the pulley grooves

2.2.1 *Groove angle.* The groove angle  $\alpha$  of pulleys should be  $26 \pm 1^\circ$ .

NOTE. - Practice shows that most wide V-belts designed for a  $26^\circ$  groove angle can also be used with pulleys with groove angles between  $22^\circ$  and  $30^\circ$ ; however, the value of this angle affects the maximum allowable tension in the driving side of the belt, and consequently the transmissible horsepower, as well as the characteristics of the tension device of the belt (for example the axial spring in a variable diameter pulley).

2.2.2 *Profile dimensions.* Radial dimensions shown in Figures 1 and 2, namely

- $b$  : groove height above the pitch line,
- $h$  : groove depth below the pitch-line,
- $d_p$  : pitch diameter (minimum in the case of Figure 1),

should not be less, for any setting of the drive, than the corresponding values for  $b$  min.,  $h$  min.,  $d_p$  min. given in Tables 3 and 4.

Profile dimensions are given in millimetres in Table 3 and in inches in Table 4.

TABLE 3 - Pulley dimensions in millimetres

Designation of the sections		W 16	W 20	W 25	W 31.5	W 40	W 50	W 63	W 80	W 100
Symbol	Approximate formula									
$l_p$		16	20	25	31.5	40	50	63	80	100
$b$ min.	$0.080 l_p$	1.3	1.6	2	2.5	3.2	4	5	6.3	8
$h$ min.	$0.335 l_p$	5.3	6.7	8.5	10.6	13.2	17.0	21.2	26.5	33.5
$d_p$ min.*	$1.80 l_p$	28	36	45	56	71	90	112	140	180
$d_e$ min.	$d_p$ min. + $2 b$ min.	30.6	39.2	49	61	77.4	98	122	152.6	196

TABLE 4 - Pulley dimensions in inches

Designation of the sections		W 16	W 20	W 25	W 31.5	W 40	W 50	W 63	W 80	W 100
Symbol	Approximate formula									
$l_p$		0.630	0.787	0.984	1.240	1.575	1.969	2.480	3.150	3.937
$b$ min.	$0.080 l_p$	0.051	0.063	0.079	0.098	0.13	0.16	0.20	0.25	0.31
$h$ min.	$0.335 l_p$	0.21	0.26	0.33	0.42	0.52	0.67	0.83	1.04	1.32
$d_p$ min.*	$1.80 l_p$	1.10	1.42	1.77	2.20	2.80	3.54	4.41	5.51	7.09
$d_e$ min.	$d_p$ min. + $2 b$ min.	1.20	1.54	1.93	2.40	3.05	3.86	4.80	6.01	7.72

NOTE. - The value  $d_p$  min. =  $1.80 l_p$  is nearly equivalent to 5.6 times the nominal height of the belt sidewalls.

\* Due to the need for compactness of these types of speed-changers, it is necessary to minimize pitch diameters. At the present level of techniques, the values given for the minimum pitch diameter should be used on the notched V-belts.

### 3. MEASURING AND CHECKING PRACTICE

#### 3.1 Preliminary note

When a wide V-belt under tension passes round a grooved pulley, its cross-section undergoes considerable distortion along the arc of contact. For this reason, the dimensions of such a belt can only be validly defined, and consequently checked, if the belt is placed under conditions as similar as possible to the average conditions for normal use.

#### 3.2 Principle of checking

The recommended device, shown in Figure 3, consists basically of two pulleys of the same dimensions, one of which can be moved by translation in the plane of symmetry of the grooves, under the effect of the measuring force  $F$  (see Tables 5 and 6).

In addition, a gauge is provided in one of the pulleys to check the section.

The belt should be checked only after it has made three complete revolutions.

#### 3.3 Dimensions of the checking device

The dimensions of the pulleys and of the gauge are given in millimetres in Table 5 and in inches in Table 6.

#### 3.4 Checking belt cross-section

The method consists in checking the radial position of the belt in the gauge-pulley.

For this purpose (see Fig. 4), the rim of the gauge-pulley has two symmetrical radial slots, widening on the outside, which make it possible to locate the contact area between belt and groove.

When checking the radial position of the belt, the twin slots should first be located as shown in Figure 3.

It should then be checked

- that belt ride out is below the limit indicated in clause 2.1.3;
- that the theoretical edge of the internal base is outside the circumference  $H_2$ .

For this purpose, the intersection of this circumference with the slots is marked, on the level flanges, by a thin engraved line, parallel to the bottom of the slot; if necessary, these engraved lines may be filled with paint, so that they can be seen more easily.

#### 3.5 Checking belt length

Check the belt length as follows :

- (a) Calculate the pitch length  $L_p$  by the formula

$$L_p = 2E + C_p$$

where

- $E$  is the distance between axes, measured as specified in clauses 3.1 and 3.2;  
 $C_p$  is the pitch circumference of the checking pulleys.

- (b) Calculate the outside length  $L_e$  by the formula

$$L_e = 2E + C$$

where

- $E$  is the distance between axes, measured as specified in clauses 3.1 and 3.2;  
 $C$  is the sum of the pitch circumference ( $C_p$ ) of the checking pulleys and the relevant length given in the table in clause 2.1.4.