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**Fire protection — Automatic sprinkler  
systems —**

Part 2:

**Requirements and test methods for wet  
alarm valves, retard chambers and water  
motor alarms**

*Protection contre l'incendie — Systèmes d'extinction automatiques du  
type sprinkler —*

*Partie 2: Exigences et méthodes d'essai des soupapes d'alarme  
hydrauliques, des limiteurs de surpression et des dispositifs d'alarme à  
moteur hydraulique*



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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 6182-2 was prepared by Technical Committee ISO/TC 21, *Equipment for fire protection and fire fighting*, Subcommittee SC 5, *Fixed firefighting systems using water*.

This second edition cancels and replaces the first edition (ISO 6182-2:1993) which has been technically revised.

ISO 6182 consists of the following parts, under the general title *Fire protection — Automatic sprinkler systems*:

- *Part 1: Requirements and test methods for sprinklers*
- *Part 2: Requirements and test methods for wet alarm valves, retard chambers and water motor alarms*
- *Part 3: Requirements and test methods for dry pipe valves*
- *Part 4: Requirements and test methods for quick-opening devices*
- *Part 5: Requirements and test methods for deluge valves*
- *Part 6: Requirements and test methods for check valves*
- *Part 7: Requirements and test methods for early suppression fast response (ESFR) sprinklers*
- *Part 8: Requirements and test methods for pre-action dry alarm valves*
- *Part 9: Requirements and test methods for water mist nozzles*
- *Part 10: Requirements and test methods for domestic sprinklers*
- *Part 11: Requirements and test methods for pipe hangers*
- *Part 12: Requirements and test methods for grooved end pipe couplings*
- *Part 13: Requirements and test methods for extended coverage sprinklers*

## Introduction

This part of ISO 6182 is one of a number of ISO International Standards prepared by ISO/TC 21 covering components for automatic sprinkler systems, including the following:

- a) carbon dioxide systems (ISO 6183);
- b) explosion suppression systems (ISO 6184);

An International Standard covering foam systems is planned.

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# Fire protection — Automatic sprinkler systems —

## Part 2:

# Requirements and test methods for wet alarm valves, retard chambers and water motor alarms

## 1 Scope

This part of ISO 6182 specifies performance, requirements, methods of test and marking requirements, for wet alarm valves, retard chambers, water motor alarms and manufacturers' specified relevant trim used in wet pipe automatic fire protection systems.

Performance and test requirements for other auxiliary components or attachments to alarm valves are not covered by this part of ISO 6182.

## 2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 7-1, *Pipe threads where pressure-tight joints are made on the threads — Part 1: Dimensions, tolerances and designation*

ISO 37, *Rubber, vulcanized or thermoplastic — Determination of tensile stress-strain properties*

ISO 188, *Rubber, vulcanized or thermoplastic — Accelerated aging and heat resistance tests*

ISO 898-1, *Mechanical properties of fasteners made of carbon steel and alloy steel — Part 1: Bolts, screws and studs*

ISO 898-2, *Mechanical properties of fasteners — Part 2: Nuts with specified proof load values — Coarse thread*

## 3 Terms and definitions

For the purposes of this document, the following terms and definitions apply.

### 3.1

#### **alarm device**

mechanical or electrical device which sounds an alarm upon operation of the valve

### 3.2

#### **clapper**

type of sealing assembly

NOTE See also 3.12.

- 3.3 compensator**  
external or internal device such as an auxiliary valve that minimizes false alarms caused by a small increase of service pressure
- 3.4 corrosion-resistant material**  
bronze, brass, Monel <sup>1)</sup> metal, austenitic stainless steel, or equivalent metallic or plastic material conforming with the requirements of this document
- 3.5 flow velocity**  
speed of water flow through a valve expressed as the equivalent water velocity through a pipe of the same nominal size as the valve
- 3.6 rated working pressure**  
maximum service pressure at which a valve or retard chamber intended to operate
- 3.7 ready (set) condition**  
state of a valve with the sealing assembly in the closed or set position with service and system pressure applied
- 3.8 reinforced elastomeric element**  
element of clapper, clapper assembly or seat seals in a composite of an elastomeric compound with one or more other components
- 3.9 retard chamber**  
volumetric type of retard device designed to minimize false alarms caused by surges and fluctuations in sprinkler system water supplies
- 3.10 retard device**  
pneumatic, hydraulic or electric timer designed to minimize false alarms caused by surges and fluctuations in sprinkler system water supplies
- 3.11 retard time**  
difference in time for actuation of alarm devices, measured from the passage of water through the wet alarm valve port, with and without the retard chamber
- 3.12 sealing assembly**  
main movable sealing element (such as a clapper) of the valve which prevents the reverse flow of water
- 3.13 sealing assembly seat ring**  
main fixed sealing element of a valve which prevents the reverse flow of water
- 3.14 sensitivity**  
minimum rate of flow from a system outlet which will open the wet alarm valve, as indicated by satisfactory operation of alarms

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1) Monel is an example of a suitable product available commercially. This information is given for the convenience of users of this part of ISO 6182 and does not constitute an endorsement by ISO of this product.

**3.15****service pressure**

static water pressure at the inlet to a valve when the valve is in the ready condition

**3.16****system pressure**

static water pressure at the main outlet of a valve when the valve is in the ready condition

**3.17****trim**

external equipment and pipework, excluding the main installation pipework, fitted to the valve

**3.18****waste of water**

discharge of any water from the alarm port of a valve that is in the ready condition

**3.19****water motor alarm**

hydraulically actuated device which provides a local audible alarm as a result of a flow through an alarm valve

**3.20****wet alarm valve**

valve that permits flow of water into a wet sprinkler system, prevents the reverse flow of water and incorporates provision for actuation of an alarm under specified flow conditions

**3.21****wet pipe system**

an automatic fire protection system in which the piping contains water and is connected to a water supply so that water discharges upon operation of the sprinklers

**4 Requirements****4.1 Nominal sizes**

The nominal size of a valve shall be the nominal diameter of the inlet and outlet connections, i.e. the pipe size for which the connections are intended. Sizes shall be 40 mm, 50 mm, 65 mm, 80 mm, 100 mm, 125 mm, 150 mm, 200 mm or 250 mm. The diameter of the waterway through the sealing assembly seat ring shall be permitted to be less than the nominal size.

**4.2 Connections**

**4.2.1** All connections shall be designed for use at the rated working pressure of the valve.

**4.2.2** The dimensions of all connections shall conform with the applicable requirements of International Standards. If International Standards are not applicable, National standards shall be permitted to be used.

**4.2.3** An opening not smaller than 15 mm nominal diameter shall be provided for an alarm line connection.

In the case of using pneumatic type retard device without water motor alarm, the connection may be a minimum of 8 mm.

**4.3 Rated working pressure**

**4.3.1** The rated working pressure shall be not less than 1,2 MPa (12 bar).

**4.3.2** Inlet and outlet connections shall be permitted to be machined for lower working pressures to match installation equipment provided the valve is marked with the lower working pressure. See 7.3 f).

#### 4.4 Bodies and covers

4.4.1 The body and cover shall be made of a material having corrosion resistance at least equivalent to cast iron.

4.4.2 Cover fasteners shall be made of steel, stainless steel, titanium, or other materials with equivalent physical and mechanical properties.

4.4.3 If non-metallic materials other than gaskets and seals or metals with a melting point less than 800 °C form part of the body or cover, the valve assembly shall be subjected to a fire exposure test, as specified in 6.10. Following the fire exposure test, the sealing assembly shall open freely and fully and the valve shall withstand the body leakage test specified in 6.8.1 without permanent deformation or failure.

4.4.4 It shall not be possible to assemble the valve with the cover plate in a position which either improperly indicates flow direction or prevents proper operation of the valve.

#### 4.5 Strength

4.5.1 An assembled valve, with the sealing assembly blocked open, shall withstand, without rupture, an internal hydrostatic pressure of four times the rated working pressure for a period of 5 min, when tested as specified in 6.9.

4.5.2 If the test in accordance with 6.9 is not done with standard production fasteners, the supplier shall provide documentation showing that the calculated design load of any fastener, neglecting the force required to compress the gasket, shall not exceed the minimum tensile strength specified in ISO 898-1 and ISO 898-2 when the valve is pressurized to four times the rated working pressure. The area of the application of pressure shall be calculated as follows.

- a) If a full-face gasket is used, the area of application of pressure is that extending out to a line defined by the inner edge of the bolts.
- b) If an "O"-ring seal or ring gasket is used, the area of application of force is that extending out to the centre line of the "O"-ring or gasket.

#### 4.6 Access for maintenance

Means shall be provided to permit access to working parts and removal of the sealing assembly. Any method adopted shall permit ready maintenance by one person with a minimum of down time.

#### 4.7 Components

4.7.1 Any component which is normally disassembled during servicing shall be designed so that it can not be reassembled improperly without providing an external visual indication, when the valve is returned to service.

4.7.2 With the exception of valve seats, all parts intended for field replacement shall be capable of being disassembled and reassembled using tools normally employed by the trade.

4.7.3 All components shall be non-detachable during normal operation of the valve.

4.7.4 Failure of the sealing assembly diaphragms or seals shall not prevent the valve from opening.

4.7.5 Sealing surfaces of sealing assemblies shall have corrosion resistance equivalent to brass or bronze and have sufficient width of surface contact to withstand ordinary wear and tear, rough usage, compression stresses and damage due to pipe scale or foreign matter carried by the water.

4.7.6 Springs and diaphragms shall not fracture or rupture during 50 000 cycles of normal operation, when tested in accordance with 6.2.

**4.7.7** There shall be no sign, on visual examination, of damage to the sealing assembly after testing for the operational requirements of 4.14 in accordance with 6.11.

**4.7.8** When wide open, the sealing assembly shall bear against a definite stop. The point of contact shall be located so that impact or reaction of the water flow will not permanently twist, bend or fracture valve parts.

**4.7.9** Where rotation or sliding motion is required, the part or its bearing shall be made of a corrosion resistant material. Materials lacking corrosion resistance shall be fitted with bushings, inserts or other parts made of corrosion resistant materials at those points where freedom of movement is required.

**4.7.10** The sealing assembly shall close towards the seat when water flow ceases. Springs shall be permitted to ensure full and proper seating.

## **4.8 Leakage**

**4.8.1** There shall be no leakage, permanent distortion or rupture of a valve when an internal pressure of twice the rated working pressure is applied for 5 min with the sealing assembly open in accordance with 6.8.1.

**4.8.2** There shall be no leakage, permanent distortion or rupture of a valve at an internal pressure of twice the rated working pressure applied to the downstream side of the sealing assembly for 5 min with the upstream end vented in accordance with 6.8.2.1.

**4.8.3** A valve shall not leak while being subjected to an internal hydrostatic pressure equivalent to a column of water 1,5 m high for 16 h in accordance with 6.8.2.2.

**4.8.4** Sealing surfaces shall prevent leakage of water into the alarm port when the valve is tested in the ready position in accordance with 6.11.

## **4.9 Non-metallic components (excluding gaskets, seals and other elastomeric parts)**

**4.9.1** Non-metallic valve parts that affect proper valve function shall be subjected to the applicable ageing of its non-metallic parts, as described in 6.4 and 6.5, using separate sets of samples, as applicable. After aging, a valve shall meet the requirements of 4.8, 4.13 and 4.14.4 when tested in accordance with the applicable tests described in 6.6, 6.8 and 6.11.

**4.9.2** There shall be no cracking, warping, creep or other signs of deterioration, which could preclude proper operation of the valve.

## **4.10 Sealing assembly elements**

**4.10.1** A seal made of elastomeric or other resilient materials shall not adhere to the mating surface when tested in accordance with 6.3.1. Where the same design of seat is used for more than one size of valve, it shall be permitted to only test the size with the highest stress on the seating surface.

**4.10.2** Any non-reinforced elastomer forming the seal shall have the following properties when tested in accordance with 6.3.2 and the appropriate sections of ISO 37:

- a) maximum set of 5 mm when 25 mm long marks are stretched to 75 mm, held for 2 min and measured 2 min after release;
- b) either:
  - 1) minimum tensile strength 10 MPa (100 bar) and minimum ultimate elongation 300 % (25 mm to 100 mm); or
  - 2) minimum tensile strength 15 MPa (150 bar) and minimum ultimate elongation 200 % (25 mm to 75 mm);

- c) after exposure to oxygen for 96 h at  $(70 \pm 1,5) ^\circ\text{C}$  and 2,0 MPa (20 bar), the tensile strength and ultimate elongation shall be not less than 70 % of the corresponding properties of specimens which have not been heated in oxygen, and any change in hardness shall not be greater than 5 type-A durometer units;
- d) after immersion in distilled water for 70 h at  $(97,5 \pm 2,5) ^\circ\text{C}$ , the tensile strength and ultimate elongation shall not be less than 70 % of the corresponding properties of specimens which have not been heated in water and the change in volume of the specimens shall be not greater than 20 %.

**4.10.3** A reinforced elastomeric sealing element shall be capable of being flexed without cracking or breaking and shall have a change in volumetric expansion not greater than 20 % when tested in accordance with 6.3.2.

**4.10.4** For a composite of an elastomeric compound with one or more other components, the tensile strength of the combination shall be at least twice that of the elastomeric material alone.

#### 4.11 Clearances

**4.11.1** The radial clearance between a hinged sealing assembly and the inside walls in every position, except wide open, shall not be less than 12 mm for cast iron bodies and shall not be less than 6 mm if the body and sealing assembly are of cast iron or steel with corrosion protective coatings tested in accordance with 6.14, non-ferrous material, stainless steel or materials having equivalent physical, mechanical and corrosion resistant properties. See Figure 1 a).

**4.11.2** There shall be a diametrical clearance of not less than 6 mm between the inner edges of a seat ring and the metal parts of a hinged sealing assembly when the valve is in the closed position. See Figure 1 b).

**4.11.3** Any space in which the sealing assembly can trap debris beyond the seat shall be not less than 3 mm deep.

**4.11.4** The diametrical clearance ( $D_2 - D_1$ ) between hinge pins and their bearings shall be not less than 0,125 mm. See Figure 1 b).

**4.11.5** The total axial clearance between the clapper hinge and adjacent valve body bearing surfaces shall be not less than 0,25 mm. See Figure 1 c).

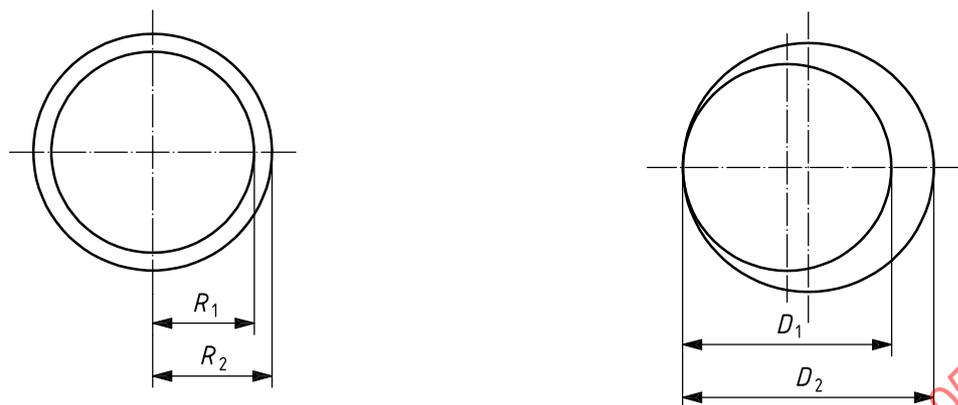
**4.11.6** Any reciprocating guide components, which are essential to allow a valve to open, shall have a minimum diametrical clearance of not less than 0,7 mm in that portion over which the moving component enters the fixed component and of not less than 0,05 mm in that portion of the moving component continuously in contact with the fixed component in the ready (set) position.

**4.11.7** Sealing assembly guide bushings or hinge-pin bearings shall project a sufficient axial distance to maintain not less than 1,5 mm (Clearance A) clearance between ferrous metal parts. See Figure 1. Clearance less than 1,5 mm shall be permitted where adjacent parts are of bronze, brass, Monel <sup>1)</sup> metal, austenitic stainless steel, titanium, or similar corrosion resistant materials. When corrosion resistance of steel parts is provided by a protective coating, the parts shall show no visible signs of deterioration of the coating such as blistering, delamination, flaking or increased resistance to movement when tested in accordance with 6.14.

**4.11.8** If provided, a compensator shall be designed such that deposits or sediment will not readily accumulate to an extent sufficient to interfere with its proper operation. There shall be sufficient clearances between the working parts to allow proper sealing of the main and any auxiliary valves.

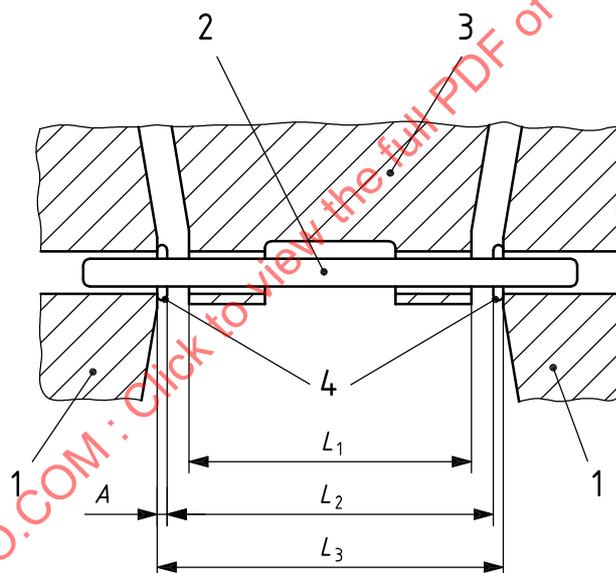
#### 4.12 Hydraulic friction loss

The maximum pressure loss across the valve at the appropriate flow given in Table 1, as tested by the method of 6.7, shall not exceed 0,04 MPa (0,4 bar). If the pressure loss exceeds 0,02 MPa (0,2 bar), the pressure loss shall be marked on the valve. See 7.3 j).



a) Radial clearance,  $C_R = R_2 - R_1$

b) Diametrical clearance,  $C_D = D_2 - D_1$

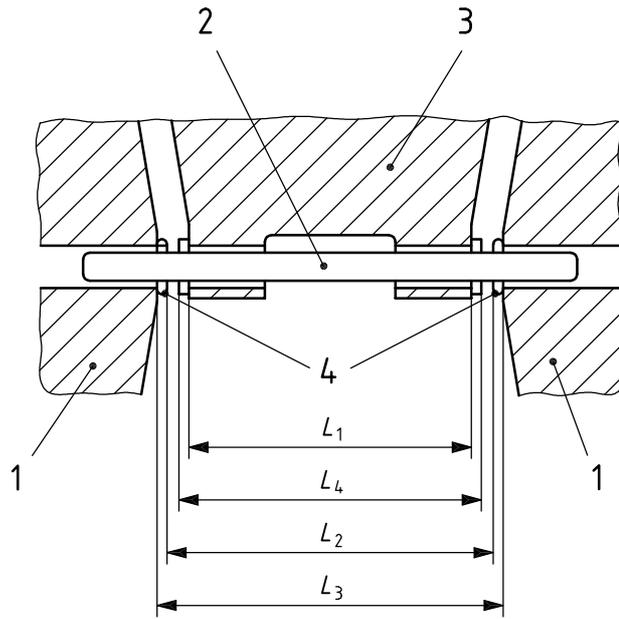


**Key**

- 1 valve body
- 2 pin
- 3 sealing assembly
- 4 bushings

c) Total axial clearance,  $C_{TA}$   
 $C_{TA} = L_2 - L_3$ ; Clearance  $A = (L_3 - L_2)/2$

**Figure 1 — Types of clearances (continued)**



- Key**
- 1 valve body
  - 2 pin
  - 3 sealing assembly
  - 4 bushings

**d) Inside bushing dimensions**

$$C_{TA} = (L_3 - L_2)/2 + (L_4 - L_1)/2$$

**Figure 1**

**Table 1 — Required flow rates for pressure drop determination**

Nominal valve size mm	Flow rate l/min
40	380
50	590
65	1 000
80	1 510
100	2 360
125	3 680
150	5 300
200	9 920
250	14 720

**4.13 Endurance**

The valve and its moving parts shall show no sign of distortion, cracks, loosening, separation or other sign of failure, following 30 min of water flow in accordance with 6.11.2.2.

#### 4.14 Operational performance

**4.14.1** The valve shall operate correctly, without adjustment, at service pressures within the range of 0,14 MPa (1,4 bar) to the rated working pressure and flow velocities up to 5 m/s, when tested to meet the requirements of this section.

**4.14.2** The valve shall meet the requirements of 4.14.3 and 4.14.4 when tested before and after the hydrostatic pressure test.

**4.14.3** The valve with associated trim shall not signal an alarm when discharge takes place downstream from the wet alarm valve at a flow rate of 15 l/min with a service pressure of 0,14 MPa (1,4 bar) when tested in accordance with 6.11.2.2.

**4.14.4** The valve with associated fittings shall signal an alarm when continuous discharge takes place downstream from the wet alarm valve at flow rates of

- a) 60 l/min at a service pressure of 0,14 MPa (1,4 bar);
- b) 80 l/min at a service pressure of 0,7 MPa (7 bar);
- c) 170 l/min at a service pressure of 1,2 MPa (12 bar);

when tested in accordance with 6.11.2.2.

**4.14.5** Valves without a retard device shall initiate continuous operation of mechanical and electrical alarm devices within 15 sec from the time that the downstream valve is opened. Wet alarm valves with retard devices shall initiate continuous operation of electrical alarm devices between 5 sec and 90 sec, and for mechanical devices a maximum of 90 sec, after the wet alarm valve opens when tested in accordance with 6.11.2.2.

**4.14.6** The ratio of service pressure to system pressure shall not exceed 1,16:1 at service pressures of 0,14 MPa (1,4 bar), 0,7 MPa (7 bar) and 1,2 MPa (12 bar), as measured by the opening of the sealing assembly and pressure equalization upstream and downstream of the sealing assembly when tested in accordance with 6.11.2.1.

**4.14.7** The valve shall stop water flow to audible alarm devices on cessation of water flow downstream of the valve when tested in accordance with 6.11.2.2.

**4.14.8** The valve shall transmit successive alarms without requiring resetting when tested in accordance with 6.11.2.2.

#### 4.15 Drains

**4.15.1** The valve shall be provided with a tapped opening to drain water from the valve body when the valve is installed in any position specified or recommended by the manufacturer. The minimum opening size shall be 20 mm nominal.

**4.15.2** Drain openings on valves shall be permitted to be used for draining the system pipework when sized in conformance with applicable system installation standards.

**4.15.3** Means shall be provided to automatically drain the piping between the valve, or any alarm shut-off valve, and the water motor alarm or water motor transmitter.

#### 4.16 Alarms

**4.16.1** A valve shall actuate its associated mechanical and electrical alarm devices at flow velocities through the valve up to 5 m/s, based on nominal pipe size, at inlet supply pressures of 0,14 MPa (1,4 bar) to the rated working pressure, when tested for operation in accordance with 6.11.2.2.

**4.16.2** The valve shall provide at least a pressure of 0,05 MPa (0,5 bar) at its alarm port at a service pressure of 0,14 MPa (1,4 bar) while actuating relevant alarm devices, when tested in accordance with 6.11.2.2.

#### **4.17 Retard device**

##### **4.17.1 Rated working pressure**

The rated working pressure shall be not less than 1,2 MPa (12 bar).

##### **4.17.2 Strength**

A retard device shall withstand an internal hydrostatic pressure of twice the rated working pressure for 5 min without failure or leakage, when tested in accordance with 6.13.1.

##### **4.17.3 Strainer**

A strainer made of corrosion-resistant material shall be provided where water passages in retard devices are 6 mm or less in diameter. The maximum dimension of a hole in the strainer shall not exceed two-thirds of the diameter of the smallest orifice protected by the strainer. The total area of the openings in the strainer shall be at least 20 times the area of the openings which the strainer is designed to protect.

##### **4.17.4 Support**

A retard device shall include means for its support. If piping is used for this support, the pipe size and length shall be stated on the instruction charts provided with the wet alarm valve.

##### **4.17.5 Connections**

**4.17.5.1** A tapped opening suitable for a pipe size not less than 20 mm shall be provided for connection of alarm devices.

**4.17.5.2** Any control valve fitted between a wet alarm valve and a retard device shall be of a type that can be locked or sealed in an open position. It shall also have external means for visual indication of the open and closed positions.

##### **4.17.6 Retard chamber drain**

A retard chamber shall be provided with means for automatic draining. The time for a retard chamber filled with water to its alarm level, including associated trim specified by the manufacturer, to drain to atmosphere shall not exceed 5 min when tested in accordance with 6.13.2.

##### **4.17.7 Components**

**4.17.7.1** Springs and diaphragms shall not fracture or rupture during 50 000 cycles of normal operation, when tested in accordance with 6.2.

**4.17.7.2** Where practical, any component which is normally disassembled during servicing shall be designed such that it cannot be reassembled improperly.

**4.17.7.3** All parts intended for field replacement shall be capable of being disassembled and reassembled using tools normally employed by the trade.

**4.17.7.4** After aging of the non-metallic parts, as described in 6.4, a retard device shall meet the requirements of 4.14, when tested in accordance with 6.11.

**4.17.7.5** After aging of the non-metallic parts, as described in 6.5, a retard device shall meet the requirements of 4.14, when tested in accordance with 6.11.

## 4.18 Water motor alarms

### 4.18.1 General

**4.18.1.1** A water motor alarm shall be designed so that it can be readily installed and serviced without damage, using non-specified tools.

Subassemblies intended to be assembled in the field as a unit shall be capable of being joined together without misalignment and without requiring any of the parts to be drilled, welded, or otherwise altered except for a part required to be cut in length and/or threaded.

**4.18.1.2** After aging of its non-metallic parts (excluding gaskets and seals), as described in 6.4 and 6.5, a water motor alarm shall show no cracking, warping, creep or other signs of deterioration that may preclude the proper operation of the water motor alarm.

Materials shall be resistant to the effects of temperature within the range of  $-35\text{ }^{\circ}\text{C}$  to  $+60\text{ }^{\circ}\text{C}$  and to the effects of sunlight excluding effects on colour.

**4.18.1.3** Water motor alarm bearings shall be self-lubricating. All moving parts shall require minimal maintenance.

**4.18.1.4** Any water motor having a non-metallic bearing or Pelten wheel shall be tested in an assembled condition in accordance with 6.12.4, following which it shall operate first at 0,05 MPa (0,5 bar) and then at 1,2 MPa (12 bar) for periods of 5 min each.

### 4.18.2 Connections

**4.18.2.1** A water motor gong body shall have a threaded opening for the water supply connection of at least 20 mm nominal bore diameter. The water supply connections shall not leak or rupture when tested at 2,4 MPa (24 bar) in accordance with 6.12.3.

**4.18.2.2** A water motor gong body shall have a threaded opening for the water drain connection of at least 50 times the area of the water nozzle or jet.

### 4.18.3 Nozzles and strainers

Nozzles shall have a diameter of not less than 3 mm and shall be made of corrosion-resistant material. Sumps, strainers or other means of preventing foreign matter from entering the nozzle or jet shall be accessible for cleaning. Strainers shall be of corrosion-resistant material. The strainer shall have openings with a maximum dimension not exceeding two-thirds of the nozzle or port diameter. The total area of the openings in the strainer shall be at least 10 times the nozzle or port area.

### 4.18.4 Operation

A water motor and gong shall operate satisfactorily for the periods specified in Table 2, when tested in accordance with 6.12.1.

Rotation of the striker shall commence at a pressure not greater than 0,035 MPa (0,35 bar) measured at the entrance to the nozzle.

### 4.18.5 Bodies and covers

A cover, housing or other means shall be provided to protect the operating mechanism of a motor water alarm against weather, birds and vermin.

#### 4.18.6 Audibility

The average of three audibility test readings at positions A, B and C shall not be less than 85 dB (A), at a pressure of 0,2 MPa (2 bar) and above at a 3 m distance, with no individual reading less than 80 dB (A). See Figure 2. The average of the three audibility test readings shall be not less than 70 dB (A) at a pressure of 0,05 MPa (0,5 bar), when tested in accordance with 6.12.2.

### 5 Production testing and quality control

**5.1** It shall be the responsibility of the manufacturer to implement and maintain a quality control program to ensure that production continuously meets the requirements of this part of ISO 6182 in the same manner as the originally tested samples.

**5.2** Every manufactured valve shall pass a hydrostatic body test for a period not less than 1 min at twice the rated working pressure without leakage.

**5.3** Following the hydrostatic body test of 5.2, every manufactured valve shall pass an operation test to verify correct functioning, including flow from the alarm port.

**5.4** Every manufactured valve shall withstand, without leakage at the valve seat, an internal hydrostatic pressure of twice the rated working pressure applied downstream of the clapper for a period of not less than 1 min.

### 6 Tests

#### 6.1 Samples

A representative sample of each size of valve shall be subjected to the following tests.

#### 6.2 Spring and diaphragm test

Subject the spring or diaphragm in the normal mounting to 50 000 cycles of normal operation in air or water. The components shall not be operated at a rate exceeding 6 cycles per minute. For sealing assembly springs, the sealing assembly shall be rotated off the seat to a 45° angle and slowly return to the closed position. For internal bypass springs, the bypass shall be operated from the full open position to the closed position. Diaphragms shall be flexed from the normally open to the normally closed position.

#### 6.3 Sealing element tests

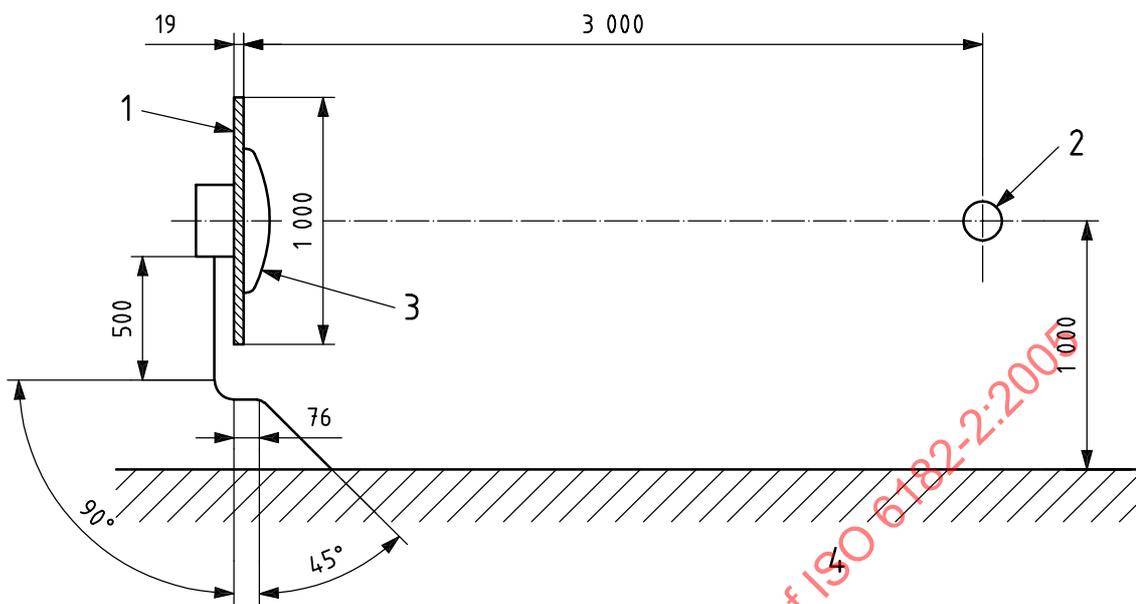
##### 6.3.1 Release test

With the valve in a normal working position and the sealing assembly in the closed position, a hydrostatic pressure of 0,35 MPa (3,5 bar) shall be applied to the outlet end of the valve for a period of 90 days. During this period, the water temperature shall be maintained at  $87 \pm 2$  °C by an immersion heater or other suitable heating device. Provisions shall be made to maintain the water in the inlet end of the valve at atmospheric pressure.

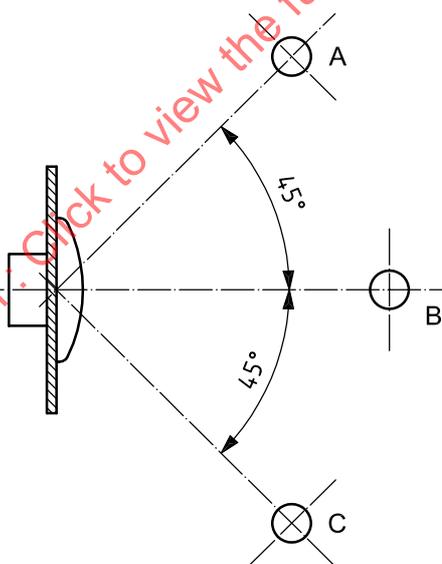
Upon completion of this period of exposure, the water shall be drained from the valve and the valve shall be allowed to cool to ambient temperature for at least 24 h. With the outlet end of the valve at atmospheric pressure, a hydrostatic pressure of 0,035 MPa (0,35 bar) shall be gradually applied to the inlet end of the valve. The sealing assembly shall move off the seat and no piece of the seal, other than colour shall adhere to the mating surface.

Where the same design of a seal is used for more than one size of valve, only a sample of the size with the highest stress on the seating surface shall be tested.

Dimensions in millimetres



a) Side view



b) Top view

**Key**

- 1 plywood, 19 × 1 000 × 1 000
- 2 sound meter
- 3 water motor gong
- 4 paved surface

A, B, C are test positions.

Diameter of piping to be required by manufacturer.

**Figure 2 — Installation for audibility test**

### 6.3.2 Non-reinforced elastomeric sealing element test

Prepare sixteen test specimens in accordance with ISO 37. Four samples shall be used to satisfy each of the following requirements:

- a) 4.10.2 a);
- b) 4.10.2 b) 1) or 4.10.2 b) 2);
- c) 4.10.2 c);
- d) 4.10.2 d).

### 6.3.3 Reinforced elastomeric sealing element test

The volume of eight reinforced elastomer sealing elements shall be measured. Each sample shall be uniquely identified. Four samples shall be exposed to an atmosphere of oxygen in accordance with ISO 188 at a pressure of 2 MPa (20 bar) for 96 h at 70 °C. The remaining four samples shall be submerged in boiling distilled water for 70 h. Following the exposure, the samples shall be allowed to cool to room temperature for at least 24 h. The volume of each shall be measured. Each sample shall then be bent back upon itself by hand three times in the same direction around a rod having a diameter four to five times the material thickness.

### 6.4 Warm water aging test for non-metallic components (excluding gaskets and seals and other elastomeric parts)

Four untested samples of each component shall be immersed in tap water at  $87 \pm 2$  °C for 180 days.

If a material cannot withstand the temperature indicated without excessive softening, distortion, or deterioration, a water aging test shall be conducted at a lower temperature, but not less than 70 °C, for a longer period of time. The duration of exposure shall be calculated from Equation (1):

$$t = 74\,857 e^{-0,069\,3T} \quad (1)$$

where

- $t$  is the exposure duration, expressed in days;
- $e$  is the base of natural logarithms (= 2,718 3);
- $T$  is the test temperature, expressed in degrees centigrade.

NOTE This equation is based on the 10 °C rule, i.e. for every 10 °C rise, the rate of a chemical reaction is approximately doubled. When applied to plastic aging, it is assumed that the life at a temperature,  $t$ , in °C is half the life at  $(t - 10)$  °C.

The samples shall be removed from the water and allowed to cool to room temperature for examination for a minimum of 24 h. The components shall be examined for cracking, warping, creep, or other signs of deterioration which would preclude the proper operation of the device. The parts are then to be assembled into valves and shall comply with the requirements of 4.8.1 and 4.14, when tested in accordance with 6.7 and 6.10.

### 6.5 Air aging test for non-metallic components (excluding gaskets and seals and other elastomeric parts)

Four untested samples of each component shall be aged in an air oven at  $120 \pm 2$  °C for 180 days. The samples shall be tested in contact with the mating materials under stresses comparable to the intended use at rated working pressure. The components shall be supported so that they do not touch each other or the sides of the oven.

If a material cannot withstand the temperature indicated without excessive softening, distortion, or deterioration, a water aging test shall be conducted at a lower temperature, but not less than 70 °C, for a longer period of time. The duration of exposure shall be calculated from Equation (2):

$$t = 737\,000 e^{-0,069\,3T} \quad (2)$$

where

$t$  is the exposure duration, expressed in days;

$e$  is the base of natural logarithms (= 2,718 3);

$T$  is the test temperature, expressed in degrees centigrade.

**NOTE** This equation is based on the 10 °C rule, i.e. for every 10 °C rise, the rate of a chemical reaction is approximately doubled. When applied to plastic aging, it is assumed that the life at a temperature,  $t$ , in °C is half the life at  $(t - 10)$  °C.

The samples shall be removed from the oven and shall be allowed to cool to room temperature for at least 24 h. All post-exposure tests shall be conducted within 72 h. The components shall be examined for cracking, warping, creep, or other signs of deterioration which would preclude the proper operation of the device. The parts are then to be assembled into valves and comply with the requirements of 4.8.1 and 4.14, when tested in accordance with 6.7 and 6.10.

## 6.6 Endurance test

Using the test apparatus described in 6.5, adjust the flow rate to the appropriate value given in Table 1, with a tolerance of  $^{+5}_0$  %. Sustain a water flow through the valve at this rate for  $(30 \text{ }^{+5}_0)$  min. Examine the valve for compliance with the requirements of 4.13.

## 6.7 Hydraulic friction loss test

Install the valve in a test apparatus using piping of the same nominal diameter. Use a differential pressure-measuring device accurate to  $\pm 2$  %.

Measure and record the differential pressure across the valve at a range of flows above and below the flows shown in Table 1. Replace the valve in the test apparatus by a section of pipe of the same nominal size and measure the differential pressure over the same range of flows. Using graphical methods, determine the pressure drops at the flows shown in Table 1. Record the hydraulic friction loss as the difference between the pressure drop across the valve and the pressure drop across the replacement pipe.

## 6.8 Valve leakage and deformation tests

### 6.8.1 Body leakage test

Install the valve in a pressure test apparatus with the sealing assembly in the open position. Blank off all openings in the valve body. Apply hydrostatic pressure of twice the rated working pressure for a period of 5 min and inspect the valve during this time for signs of leakage. The valve shall conform to the requirements of 4.8.1.

### 6.8.2 Valve leakage and deformation test

Fit the valve under test with appropriate trim, including any external compensator and blank off the downstream outlet connection.

**6.8.2.1** Fit a connector and bleed valve to enable the downstream portion of the valve to be hydrostatically pressurized. Plug all other connections on the portion of the valve downstream of the sealing assembly. Apply an internal hydrostatic pressure of twice the rated working pressure downstream of the closed sealing assembly for a period of 5 min. Place a sheet of paper under the valve. Leakage past the sealing assembly will be indicated by wetting of the paper.

Examine the alarm outlet for leakage.

**6.8.2.2** Fit the valve outlet connection with an open-ended rise pipe and any trim which includes an external compensator.

With the clapper assembly in the closed position, fill the rise pipe with water to a level 1,5 m above the sealing assembly centre. Place a sheet of paper under the valve assembly. Leakage past the sealing assembly will be indicated by wetting of the paper. Test the valve in its intended position or positions of use and check for leakage after testing for 16 h.

**6.8.2.3** Fit the valve with a connector on the inlet side of the sealing assembly and a bleed valve on the outlet side. Blank off or plug all other openings. Apply a hydrostatic pressure of twice the rated working pressure. Examine the valve for leakage for a period of 5 min. Release the pressure and examine the internal component for leakage, permanent distortion or rupture.

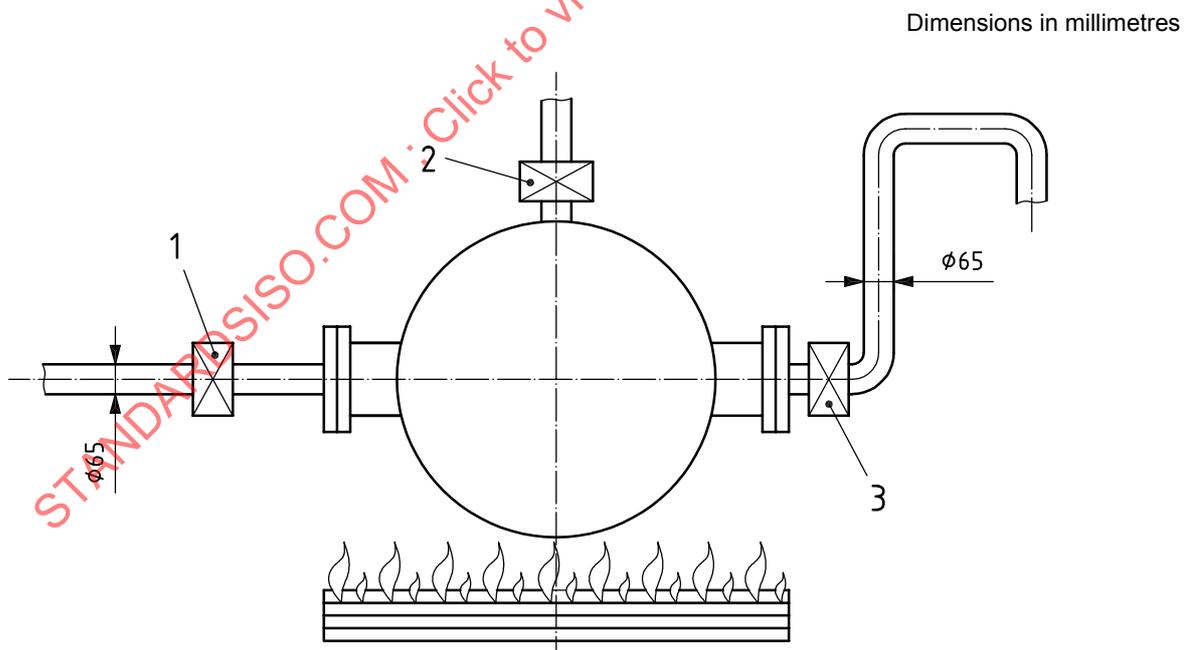
**6.9 Body strength test**

For the purpose of this test, standard production bolts, gaskets and seals may be replaced by components capable of withstanding the test pressure. The valve inlet and outlet connections and all other openings shall be suitably blanked off or plugged.

There shall be a connection for hydrostatically pressurizing the assembled sample valve at the inlet connection and a means of venting air and pressurizing fluid at the outlet connection. With the sealing assembly blocked open, the sample valve assembly shall be internally hydrostatically pressurized at 4 times the rated working pressure, but not less than 4,8 MPa (48 bar), for a period of 5 min. The valve shall conform to the requirements of 4.5.1.

**6.10 Fire exposure test**

Mount the valve horizontally with body openings sealed as shown in Figure 3. Open shut-off valves A and B. Fill the pipework and valve with water. Open the test valve to vent all air.



**Key**

- 1 shut-off valve A
- 2 test valve
- 3 shut-off valve B

**Figure 3 — Fire test installation**

Close valve A and valve B.

Position a fire tray, having a surface area not less than 1 m<sup>2</sup>, centrally beneath the sample valve. Place a sufficient volume of a suitable fuel in the tray to give an average air temperature between 800 °C and 900 °C around the valve for a period of 15 min after a temperature of 800 °C is reached.

Measure the temperature with a thermocouple positioned 10 mm from the surface of the sample valve on a horizontal plane parallel to the axis at the mid-point between the mounting connections.

**CAUTION — Ensure the test valve remains open to atmosphere while testing to permit venting of any pressure build-up.**

Ignite the fuel and, 15 min after 800 °C is attained, remove the fire tray or extinguish the fire. Starting within 1 min of extinguishment, or removal of the tray, cool the sample valve by flowing 100 l/min of water through the pipework for 1 min. Test the sample valve with an internal hydrostatic pressure by the method of 6.8.1. Gaskets and seals may be replaced for this hydrostatic test. The valve shall conform to the requirements of 4.4.2.

## 6.11 Operational test

### 6.11.1 General

Subject the valve to a series of operational tests at water service pressures of 0,04 MPa (0,4 bar), 0,14 MPa (1,4 bar) and from 0,2 MPa (2 bar) to the rated working pressure in increments of 0,1 MPa (1 bar), using the test installation shown in Figure 3.

### 6.11.2 Wet alarm valve installation

Install the wet alarm valve in the apparatus generally depicted in Figure 3. Additionally, install an alarm line shut-off valve, alarm line drain and mechanical and electrical alarm devices recommended as suitable by the manufacturer.

Before beginning the test program, clean sealing assembly seats and seat rings and all other operating parts. Seat the main sealing assembly member. Bolt the cover plate in place. Fill the valve with water and equalize upstream and downstream pressures. Fully open the main water supply valve and check for leakage into the alarm port.

Use the alarm devices to determine that the operational requirements of 4.16 are satisfied. Examine the wet alarm valve after completion of the tests for damage to the sealing element.

#### 6.11.2.1 Ratio test

Apply a service pressure  $(0,14 \pm 0,0028)$  MPa  $[(1,4 \pm 0,028)$  bar]. Release a small flow of water from the downstream side of the wet alarm valve and record the maximum achieved differential pressure to an accuracy of  $\pm 2$  %. This is indicated by the maximum value of differential pressure obtained just before the valve opens.

Calculate the ratio,  $R_p$ , as given in Equation (3):

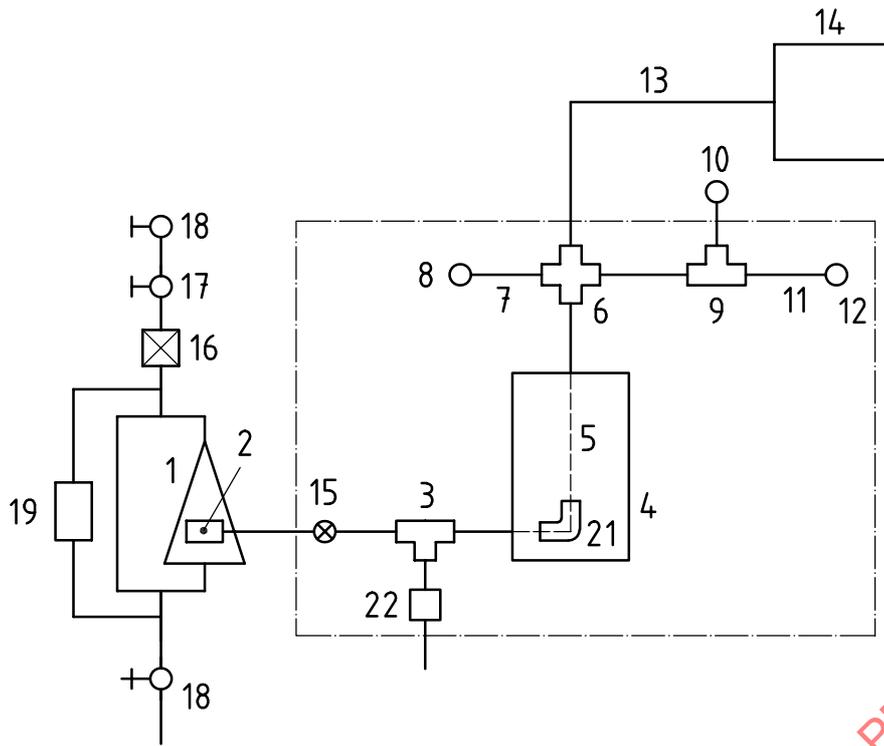
$$R_p = p_s / (p_s - \Delta p_{\max}) \quad (3)$$

where

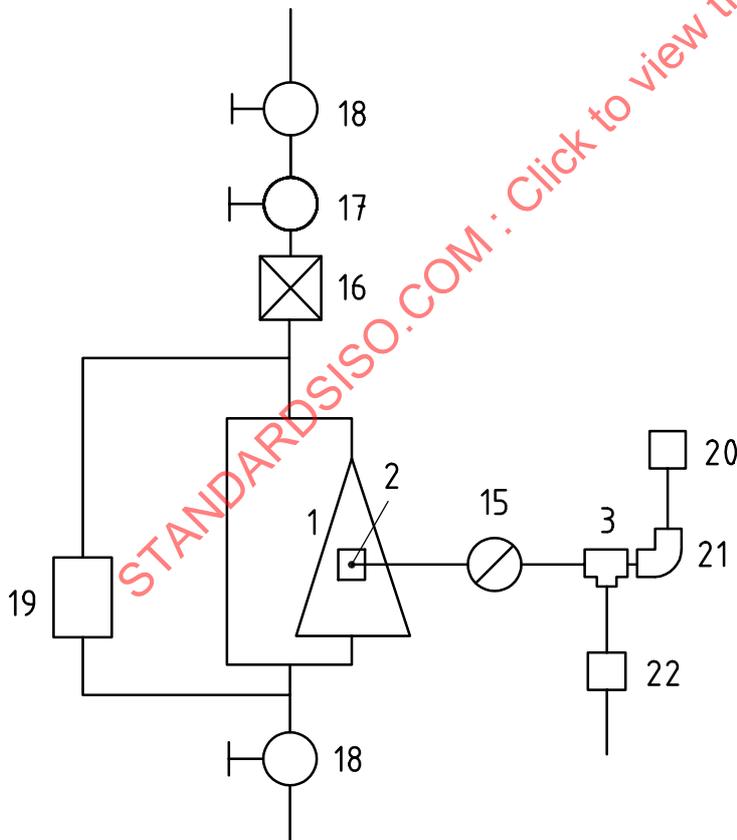
$p_s$  is the service pressure;

$\Delta p_{\max}$  is the maximum differential pressure.

Repeat the test at service pressures of 0,7 MPa (7 bar) and 1,2 MPa (12 bar). The valve shall conform to the requirements of 4.14.7.



a)



b)

**Key**

- 1 wet alarm valve
- 2 alarm port
- 3 tee connection to drain and drain valve (if not fitter with retard chamber or trim)
- 4 retard chamber (if fitted)
- 5 alternate piping for system without retard chamber
- 6 cross fitting
- 7 pipe nipple (252 maximum length)
- 8 pressure switch [0,05 MPa (0,5 bar)]
- 9 tee control
- 10 pressure gauge for monitoring line pressure
- 11 pipe nipple (77 maximum length)
- 12 ball valve
- 13 pipe to water motor alarm ( $\leq 0,5$  length)
- 14 water motor alarm
- 15 control valve (normally part of trim)
- 16 flow measure device
- 17 control valve (quick-opening type)
- 18 control valve
- 19 differential pressure measuring device
- 20 retard device
- 21 elbow
- 22 auto drip

**Figure 4 — Installation for operational tests**