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**Polygonal taper interface with flange  
contact surface —**

**Part 1:  
Dimensions and designation of shanks**

*Interfaces à cône polygonal avec face d'appui —*

*Partie 1: Dimensions et désignation des queues*

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Published in Switzerland

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular, the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see [www.iso.org/directives](http://www.iso.org/directives)).

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights. Details of any patent rights identified during the development of the document will be in the Introduction and/or on the ISO list of patent declarations received (see [www.iso.org/patents](http://www.iso.org/patents)).

Any trade name used in this document is information given for the convenience of users and does not constitute an endorsement.

For an explanation of the voluntary nature of standards, the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the World Trade Organization (WTO) principles in the Technical Barriers to Trade (TBT), see [www.iso.org/iso/foreword.html](http://www.iso.org/iso/foreword.html).

This document was prepared by Technical Committee ISO/TC 29, *Small tools*, Subcommittee SC 9, *Tools with defined cutting edges, holding tools, cutting items, adaptive items and interfaces*.

This third edition cancels and replaces the second edition ISO 26623-1:2014, which has been technically revised.

The main changes compared to the previous edition are as follows:

- size 80X is removed;
- Figure 1 is divided in 6 figures to achieve better readability;
- internal design is changed to fit medium-transfer unit referenced (ISO 22402-2);
- information about medium-transfer unit is added;
- dimensions for dynamical balancing by design when used are added in [Annex A](#);

A list of all parts in the ISO 26623 series can be found on the ISO website.

Any feedback or questions on this document should be directed to the user's national standards body. A complete listing of these bodies can be found at [www.iso.org/members.html](http://www.iso.org/members.html).

# Polygonal taper interface with flange contact surface —

## Part 1: Dimensions and designation of shanks

### 1 Scope

This document specifies dimensions for polygonal taper interfaces with flange contact surface — polygon shanks for automatic and manual tool exchange to be applied on machine tools (e. g. turning machines, drilling machines, milling machines and turn/milling centres as well as grinding machines). A range of shank sizes is specified.

These shanks incorporate a grooved flange to enable automatic tool exchange. The clamping can be realized by a circular groove using clamping segments or internal screw threads using centre-bolts.

The torque is transmitted by form lock (polygon).

### 2 Normative references

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 965-2, *ISO general purpose metric screw threads — Tolerances — Part 2: Limits of sizes for general purpose external and internal screw threads — Medium quality*

ISO 2768-1, *General tolerances — Part 1: Tolerances for linear and angular dimensions without individual tolerance indications*

ISO 2768-2, *General tolerances — Part 2: Geometrical tolerances for features without individual tolerance indications*

ISO 22402-2, *Medium-transfer units for tool interfaces — Part 2: Transfer units for polygonal taper interfaces in accordance with ISO 26623*

### 3 Terms and definitions

No terms and definitions are listed in this document.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- ISO Online browsing platform: available at <https://www.iso.org/obp>
- IEC Electropedia: available at <http://www.electropedia.org/>

## 4 Polygonal taper interface with flange contact surface

### 4.1 General

Tolerances for linear dimensions for features without individual tolerance indications shall be of tolerance class “m” in accordance with ISO 2768-1 and geometrical tolerances for features without individual tolerance indications shall be of tolerance class “K” in accordance with ISO 2768-2.

Tolerances for threads where tolerance is not stated shall be in accordance with ISO 965-2.

## 4.2 Polygon shanks

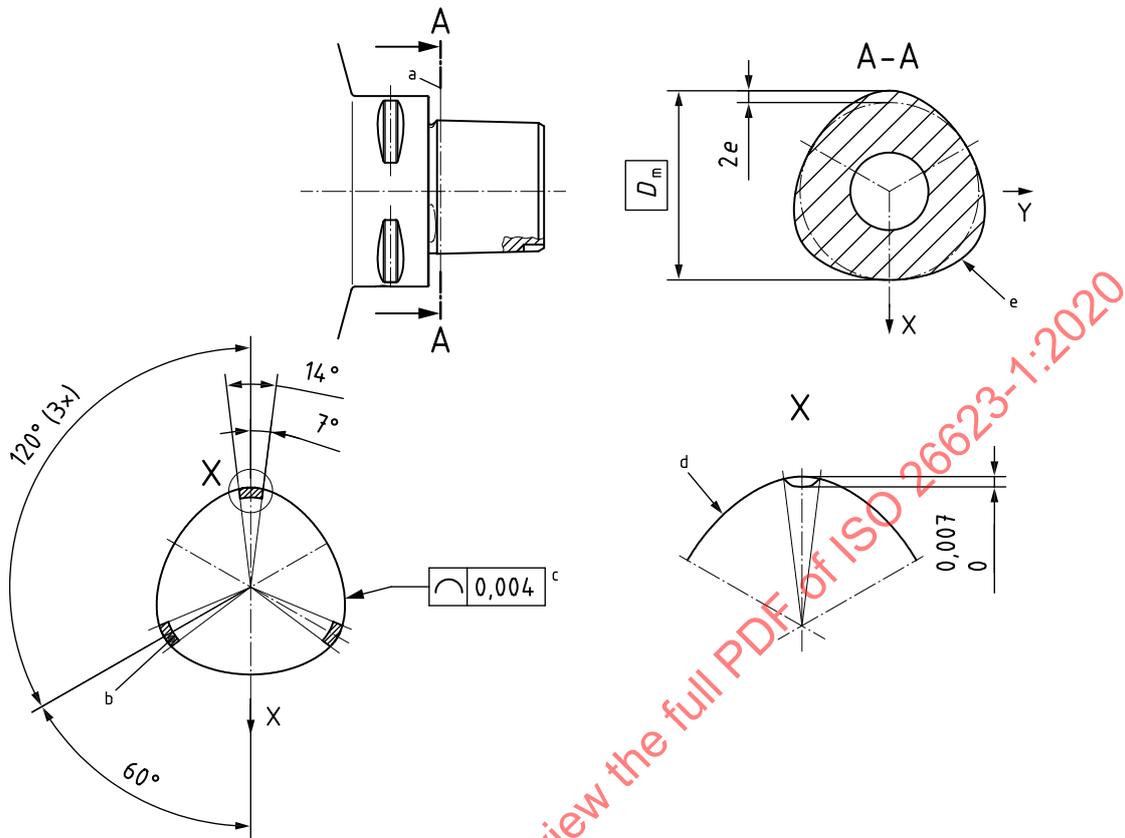
The dimensions of polygon shanks shall be in conformance with [Figure 1](#) to [Figure 7](#) and [Table 1](#). [Figure 6](#) only applies if chip hole bore is used.

NOTE Additional recommendations for use and application (i.e. dynamical balancing by design) are given in [Annex A](#). Only in cases where stationary tools need extra room for clamping devices or coolant supply, dimensions specified in [Annex B](#) are valid.

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Dimensions in millimetres

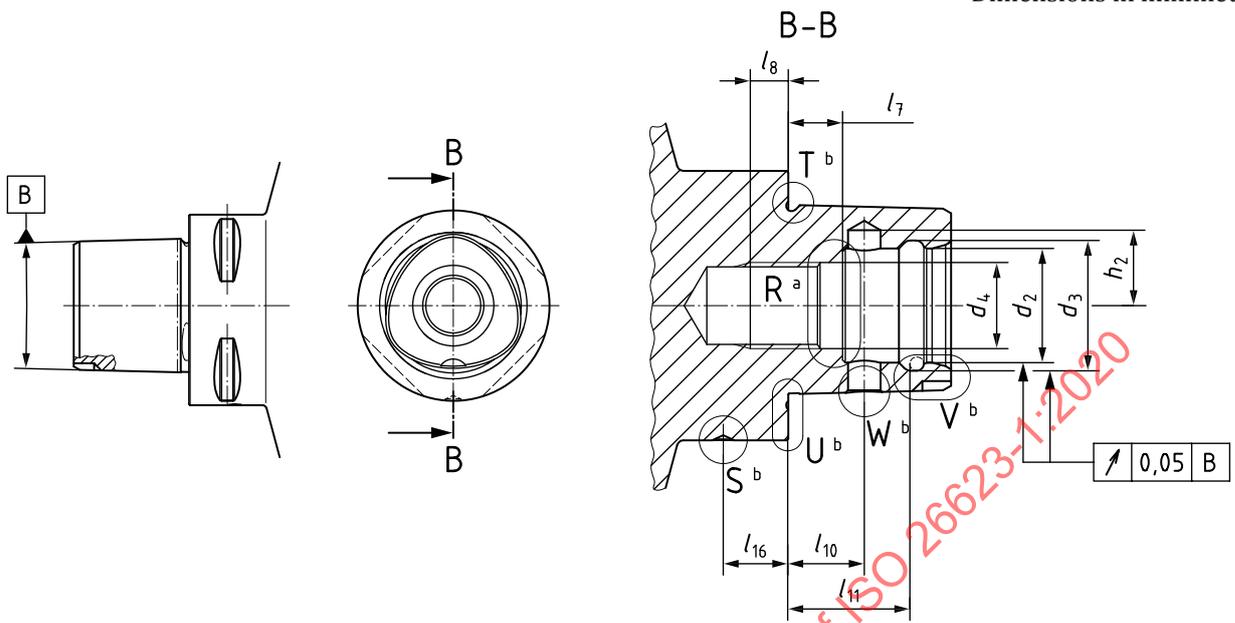


**Key**

- |   |  |   |  |
|---|--|---|--|
| a | Gauge line.  | e | Polygon curve according to <a href="#">Figure 7</a> .                      |
| b | Form of profile from actual ground curve = $\begin{matrix} +0 \\ -0,007 \end{matrix}$ (sectioned areas). | X | x-axis for theoretical polygon curve according to <a href="#">Figure 7</a> |
| c | Theoretical polygon curve.   | Y | y-axis for theoretical polygon curve according to <a href="#">Figure 7</a> |
| d | Actual ground curve.   |   |  |

**Figure 2 — Shape of polygon shanks**

Dimensions in millimetres



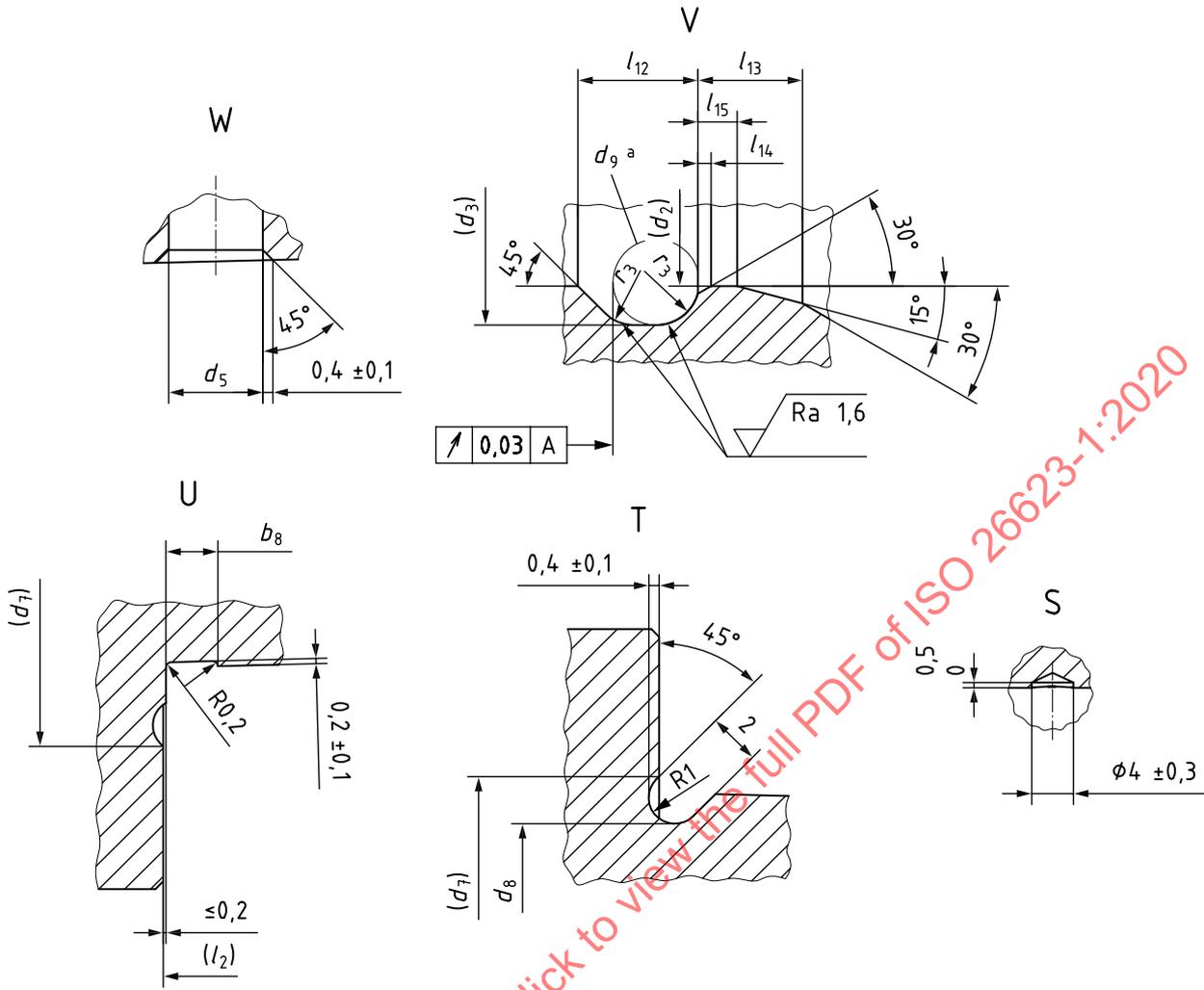
a See [Figure 5](#).

b See [Figure 4](#).

NOTE Details S, T, U, V and W are shown in [Figure 4](#).

**Figure 3 — Dimensions of polygon shanks (overview)**

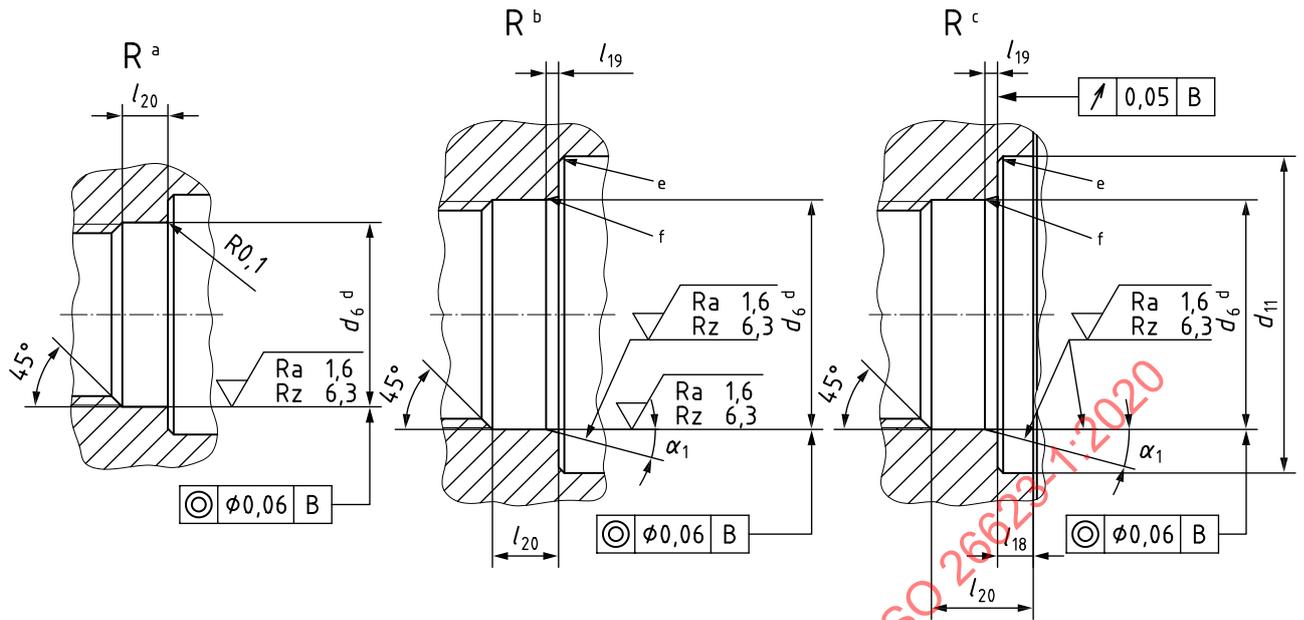
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<sup>a</sup> Gauge ball.

NOTE An overview and the location of details S, T, U, V and W are shown in [Figure 3](#).

**Figure 4 — Dimensions of polygon shanks (details S, T, U, V and W)**

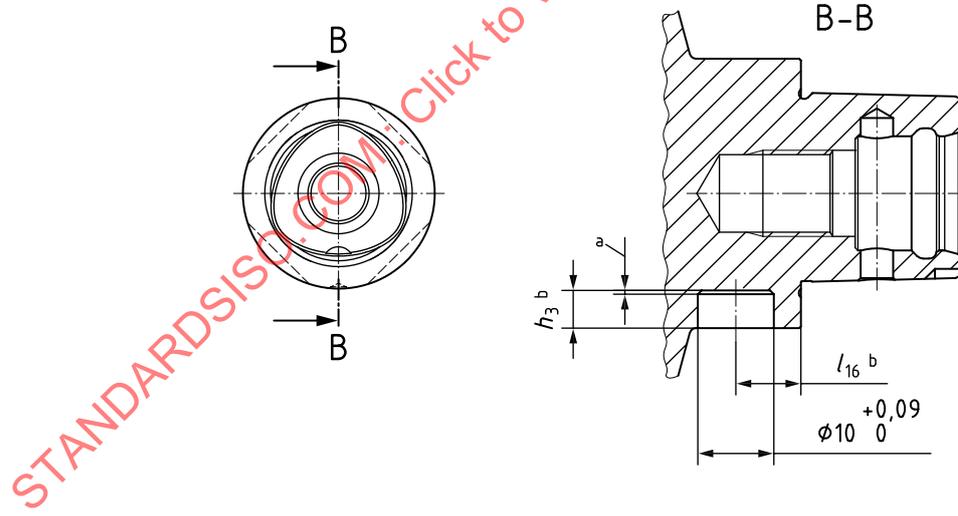


- a Sizes 32 to 50.
- b Sizes 63 and 80.
- c Size 100.

- d Machined at the same set-up as d4.
- e  $0,5 \times 45^\circ$  or R0,5.
- f Rounded.

Figure 5 — Dimensions of polygon shanks (detail R)

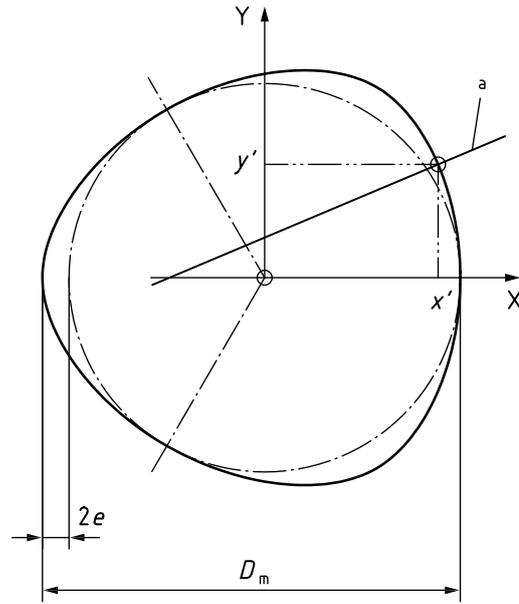
Dimensions in millimetres



- a  $0,3 \times 45^\circ$  or R0,3.

- b Data chip hole only optional/if used.

Figure 6 — Polygon shanks (data chip hole design — if used)



$$x' = D_m/2 \times \cos \gamma - 2 \times e \times \cos(2\gamma) + e \times \cos(4\gamma)$$

$$y' = D_m/2 \times \sin \gamma + 2 \times e \times \sin(2\gamma) + e \times \sin(4\gamma)$$

a Normal to polygon curve.

Figure 7 — Theoretical polygon curve

### 4.3 Dimensions

The dimensions of polygon shanks shall be in conformance with [Table 1](#).

Tolerances for linear dimensions and geometrical tolerance for features without individual tolerance indication shall be ±0,05 mm for dimensions from 0 mm up to 0,2 mm, and ± 0,1 mm for dimensions from 0,2 mm up to 0,499 mm, the others in accordance with ISO 2768-mk.

Tolerances for threads, where tolerance is not stated, shall be in accordance with ISO 965-2.

Table 1 — Polygon shanks — Dimensions

Dimensions in millimetres

Size	32	40	50	63	80	100
$\alpha_1$	—			15	20	20
$b_1$ ±0,1	39	46	59,3	70,7	86	110
$b_2$	28,3	35,3	44,4	55,8	71,1	88,7
$b_3$ ±0,1	27,9	34,9	44	55,4	70,7	88,3
$b_4$	4,2	5,2	6,5	8,5	9,6	12,4
$b_5$	4,5	5,5	7	9	10,1	13,2
$b_6$	2,5	2,5	3,5	3,5	3,5	5
$b_7$	2,6	2,6	4,1	4,1	4,1	6,1
$b_8$	1,5	1,5	2	2	2	2

<sup>a</sup> Dimensions for dynamical balancing by design are specified in [Figure A.1](#) and [Table A.2](#).

<sup>b</sup> Alternative dimensions for stationary tools are specified in [Annex B](#).

Table 1 (continued)

Size		32	40	50	63	80	100	
$d_1$	$\pm 0,1$	32	40	50	63	80	100	
$d_2$	$+0,1$ $-0,05$	15	18	21	28	32	43	
$d_3$	$\pm 0,05$	16,5	20	24	32	38	50,5	
$d_4$		M12×1,5	M14×1,5	M16×1,5	M20×2	M20×2	M24×2	
$d_5$	$\pm 0,1$	3,6	4,6	6,1	8,1	9,1	12	
$d_6$	H10	12,3	14,3	16,3	20,3	20,3	26	
$d_7$		25,2	31,6	39,1	48,5	60,8	79,4	
$d_8$	$\pm 0,1$	21,6	28	35,5	44,9	57,2	75,8	
$d_9$		1,5	2	3	4	6	7,5	
$d_{10}$		5	5	7	7	7	10	
$d_{11}$	$\pm 0,05$	—						31
$D_m$		22	28	35	44	55	72	
$e$		0,7	0,9	1,12	1,4	2	2,8	
$f_1$		$0,3 \times 45^\circ$	$0,3 \times 45^\circ$	$0,5 \times 45^\circ$	$0,5 \times 45^\circ$	$0,5 \times 45^\circ$	$0,5 \times 45^\circ$	
$h_1$	tol.	9	11	14	18	22,2	29	
$h_2$		—	11	14	17,5	22	—	
$h_3$	$+0,2$ $0$	5,4	5,2	5,1	5	4,9	4,9	
$h_4$		a						
$h_5$		a						
$h_6$		a						
$h_7$		a						
$h_8$		a						
$l_1$		2,5	2,5	3	3	3	3	
$l_2$	$\pm 0,1$	19	24	30	38	48	60	
$l_3$	min	15	20	20	22	30	36	
$l_4$		1	1,5	1,5	1,5	1,5	2	
$l_5$	tol.	3,35	4,2	5,55	6,45	8,25	11,25	
$l_6$	$\pm 0,15$	$\pm 0,15$	$\pm 0,2$	$\pm 0,25$				
$l_6$	$\pm 0,15$	6	8	10	12	12	16	
$l_7$	$\pm 0,15$	6	9	10	11	20	20	
$l_8$	min	6	6	7	9	0	10	
$l_9$		a						
$l_{10}$	$\pm 0,2$	8	11,5	14	15,5	25	26,5	
$l_{11}$	$\pm 0,1$	13,5	17,5	22	26	34	42,5	
$l_{12}$	tol.	2,8	3,4	4,6	5,8	8,5	11	
$l_{12}$		$\pm 0,15$					—	
$l_{13}$		3,6	3,5	4	6,5	6,5	8	
$l_{14}$		0,3	0,4	0,5	0,6	0,6	0,6	
$l_{15}$	$\pm 0,2$	2	1,4	1,5	1,6	1,6	1,9	
$l_{16}$		9	12	12	12	12	16	

a Dimensions for dynamical balancing by design are specified in [Figure A.1](#) and [Table A.2](#).

b Alternative dimensions for stationary tools are specified in [Annex B](#).

**Table 1** (continued)

Size	32	40	50	63	80	100
$l_{17}$	b					
$l_{18}$	—					3,8
$l_{19}$	—			1,1	1,5	1,5
$l_{20}$	3,9	4	4	5,8	5,8	9,5
$r_1$ ±0,3	4	4	5	6	7	8
$r_2$	0,3	0,3	0,5	0,5	0,5	0,5
$r_3$ $\begin{matrix} 0 \\ -0,1 \end{matrix}$	0,75	1	1,5	2	3	3,75
$r_4$	0,3	0,3	0,3	0,3	0,3	0,4
$r_5$	a					
<sup>a</sup> Dimensions for dynamical balancing by design are specified in <a href="#">Figure A.1</a> and <a href="#">Table A.2</a> .						
<sup>b</sup> Alternative dimensions for stationary tools are specified in <a href="#">Annex B</a> .						

## 5 Design

### 5.1 Medium-transfer unit

Medium-transfer units in accordance with ISO 22402-2 shall be used.

### 5.2 Clamping forces

The clamping system shall provide sufficient clamping force to ensure contact of the shank flange with the face of the receiver. A guide to clamping forces for polygon shank is given in [Annex A](#).

## 6 Designation

A polygon shank in accordance with this document shall be designated by:

- a) "POLYGON SHANK";
- b) reference to this document (i.e. ISO 26623-1);
- c) designation symbols PSC;
- d) nominal size, in millimetres.

EXAMPLE Designation of a polygon shank for automatic and manual tool exchange with nominal size 32:

**POLYGON SHANK ISO 26623-1 — PSC 32**

## Annex A (informative)

### Recommendations for use and application

#### A.1 Clamping forces

Variations of taper shank and receiver size within the specified limits of tolerance will cause the portion of the clamping force acting on the flange surface to vary. However, the clamping forces given in [Table A.1](#) will ensure that the portion of acting on the flange surface is never less than 80 % of the total. The flange contact is decisive for the stiffness of the polygonal taper interface.

**Table A.1 — Clamping force**

<b>Nominal size, mm</b>	32	40	50	63	80	100
<b>Clamping force, kN</b>	15	20	25	30	40	60

Lower clamping forces can be sufficient when operational loads are low (e.g. cutting forces in finish machining). Conversely, higher clamping forces can be required when high operational loads are encountered (e.g. cutting and feed forces in heavy machining).

#### A.2 Information about speeds, torques, bending loads and stiffness

The manufacturer should provide information regarding permissible speeds, torque transmitting capacities, bending loads and stiffness.

#### A.3 Information about material and heat treatment

The polygonal taper shanks should be heat treated with considerations for strength, hardness and case depth (if not through hardened). Toughness and wear requirements are to be taken into consideration.

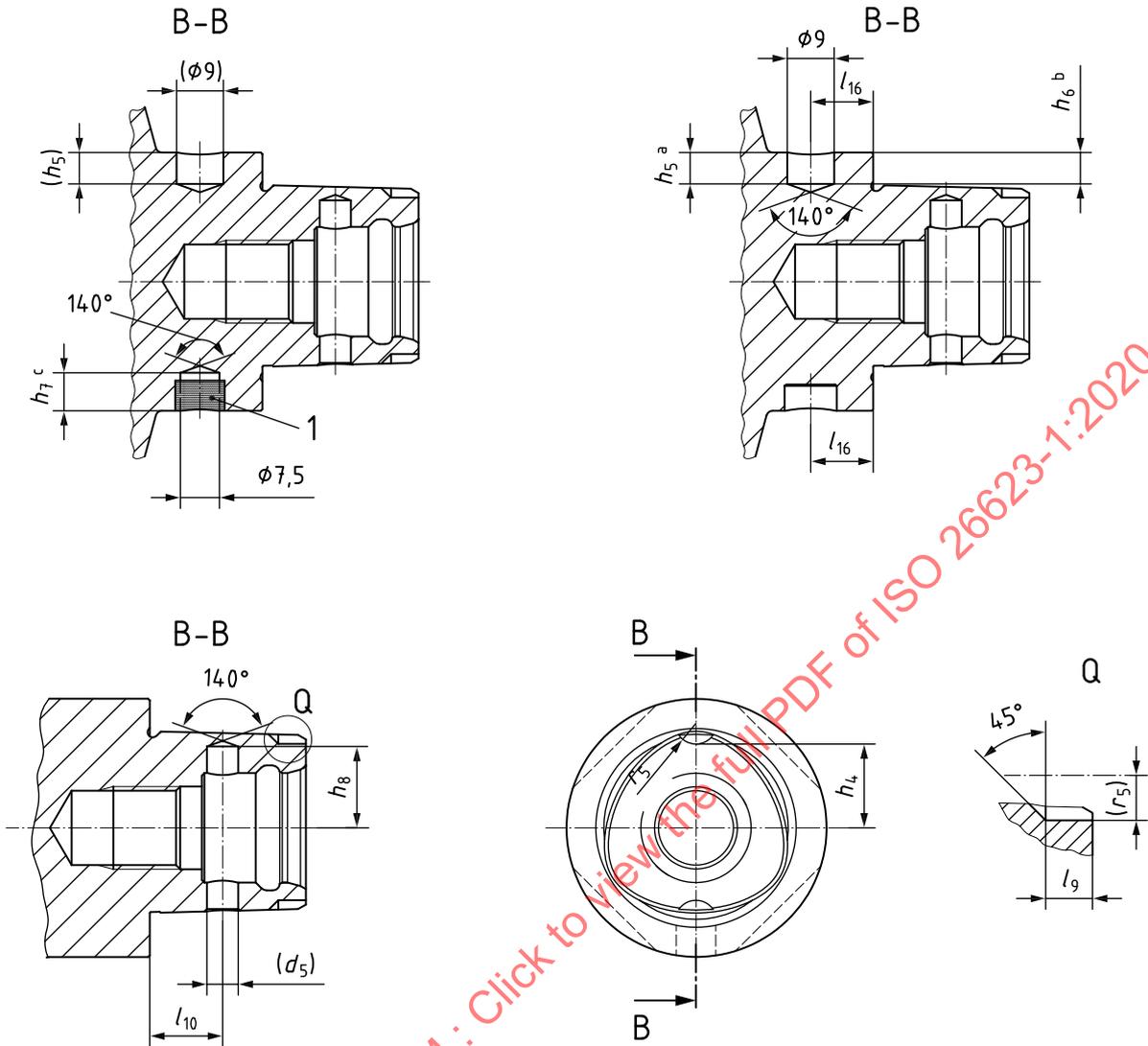
The coupling could be either through hardened or surface hardened depending on what is suitable for the range of application.

#### A.4 General surface hardness recommendations

Tapered polygon, internal clamping groove and axial contact surface: HRC 42 minimum. External gripper groove area: HRC 51 minimum.

#### A.5 Dynamical balancing by design

Especially for Polygon shank adapters and cutting tools with symmetrical front parts of small diameters, only the balancing measures at the polygon shank as described in [Figure A.1](#) and [Table A.2](#) can provide a dynamically balanced product as there are no other balancing areas available. Chip holes on tools that are dynamically balanced by design shall be made with a flat bottom when originally manufactured.



**Key**

- 1 data chip (dimensions for chip bore hole see [Figure 6](#))
- a Dimension is used for balancing when no data chip is initially mounted.
- b Dimension is used for balancing when a data chip is initially mounted.
- c Dimension is used to make rework for balancing when no data chip is initially mounted but is mounted afterwards.

**Figure A.1 — Dimensions for dynamical balancing by design**