

# INTERNATIONAL STANDARD

**ISO**  
**10297**

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## **Gas cylinders — Refillable gas cylinder valves — Specification and type testing**

*Bouteilles à gaz — Robinets de bouteilles à gaz rechargeables —  
Spécifications et essais de type*

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Reference number  
ISO 10297:1999(E)

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 3.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 10297 was prepared by Technical Committee ISO/TC 58, *Gas cylinders*, Subcommittee SC 2, *Cylinder fittings*.

Annexes B and C form an integral part of this International Standard, annex A and the Bibliography are for information only.

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# Gas cylinders — Refillable gas cylinder valves — Specification and type testing

## 1 Scope

This International Standard specifies requirements for refillable gas cylinder valves and the method of testing such valves for type approval.

This International Standard is applicable to valves to be fitted to gas cylinders of up to 150 l water capacity, intended to convey compressed, liquefied or dissolved gases.

This International Standard is only applicable to valves operated by a hand wheel or a key.

This International Standard is not applicable to valves for breathing equipment, fire extinguishers, cryogenic equipment or liquefied petroleum gas (LPG).

Additional specific requirements for valves fitted with pressure-reducing, pressure-retaining and non-return devices are not covered by this International Standard.

## 2 Normative references

The following normative documents contain provisions which, through reference in this text, constitute provisions of this International Standard. For dated references, subsequent amendments to, or revisions of, any of these publications do not apply. However, parties to agreements based on this International Standard are encouraged to investigate the possibility of applying the most recent edition of the normative documents indicated below. For undated references, the latest edition of the normative document referred to applies. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 188, *Rubber, vulcanized or thermoplastic — Accelerated ageing or heat-resistance tests.*

ISO 1817, *Rubber, vulcanized — Determination of the effect of liquids.*

ISO 5145, *Cylinder valve outlets for gases and gas mixtures — Selection and dimensioning.*

ISO 10156, *Gases and gas mixtures — Determination of fire potential and oxidizing ability for the selection of cylinder valve outlets.*

ISO 10920, *Transportable gas cylinders — 25E taper thread for connection of valves — Specification.*

ISO 11114-1, *Transportable gas cylinders — Compatibility of cylinder and valve materials with gas contents — Part 1: Metallic materials.*

ISO 11114-2, *Transportable gas cylinders — Compatibility of cylinder and valve materials with gas contents — Part 2: Non-metallic materials.*

ISO 11114-3, *Transportable gas cylinders — Compatibility of cylinder and valve materials with gas contents — Part 3: Autogenous ignition test in oxygen atmosphere.*

ISO 11116-1, *Gas cylinders — 17E taper thread for connection of valves to gas cylinders — Part 1: Specifications.*

ISO 11117, *Gas cylinders — Valve protection caps and valve guards for industrial and medical gas cylinders — Design, construction and tests.*

### 3 Definitions and symbols

For the purposes of this International Standard, the following definitions and symbols apply.

#### 3.1 working pressure

$p_w$   
settled pressure, at a uniform temperature of 15 °C, for a full gas cylinder

#### 3.2 operating pressure

$p_o$   
varying pressure which is developed in a cylinder during service

#### 3.3 valve test pressure

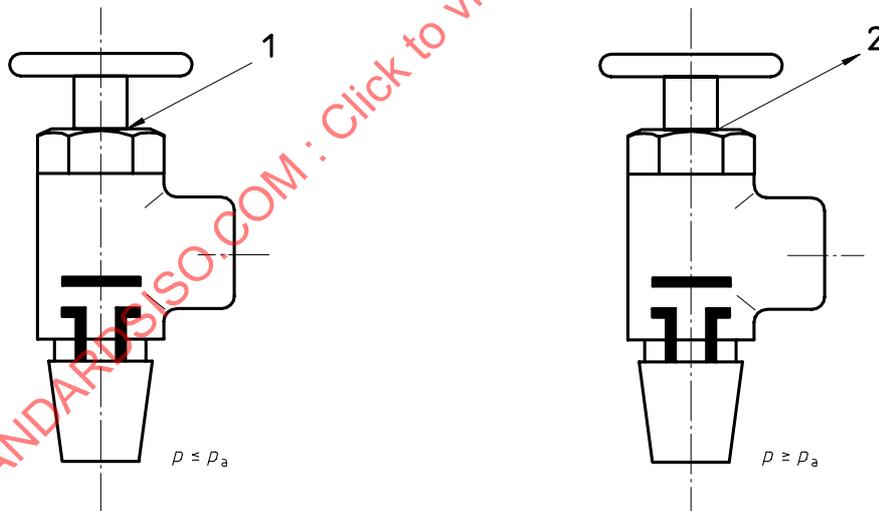
$p_{vt}$   
for permanent gases:

$$p_{vt} = 1,2 \times p_w$$

For liquefied gases and dissolved gases under pressure (for example, acetylene),  $p_{vt}$  shall be at least equal to the minimum test pressure of the cylinder quoted in the relevant transportation regulation for that gas or gas group.

#### 3.4 external tightness

tightness to atmosphere (leakage in and/or leakage out) when the valve is open (see Figure 1)



#### Key

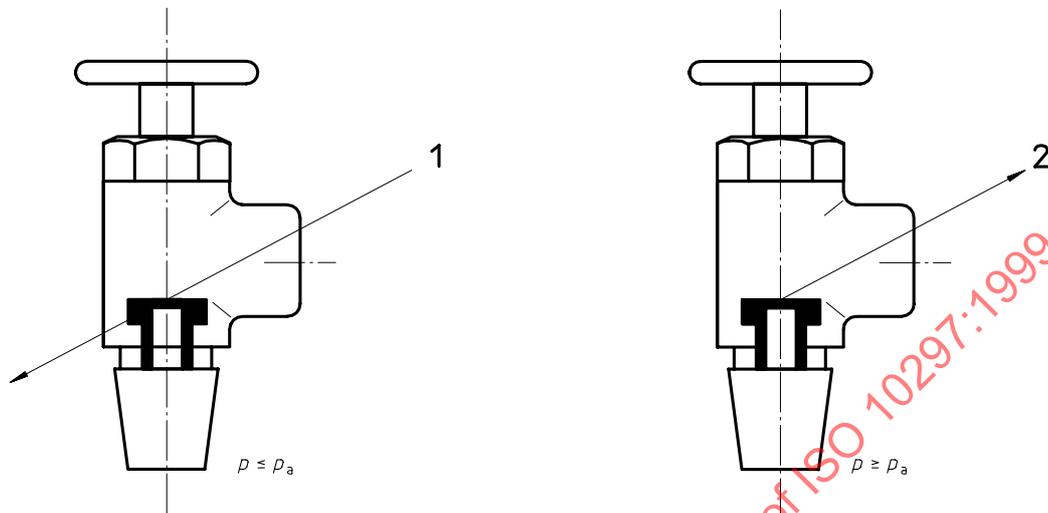
- $p$  = internal pressure
- $p_a$  = atmospheric pressure
- 1 Leakage in
- 2 Leakage out

Figure 1 — External tightness

### 3.5

#### Internal tightness

tightness across the valve seat (leakage in and/or leakage out) when the valve is closed (see Figure 2)



#### Key

$p$  = internal pressure

$p_a$  = atmospheric pressure

1 Leakage in

2 Leakage out

Figure 2 — Internal tightness

### 3.6

#### minimum closing torque

$T_c$

minimum closing torque applied to valve operating mechanism necessary to obtain the internal tightness

### 3.7

#### resistance torque

maximum closing torque applied to valve operating mechanism which the valve can withstand without suffering damage

### 3.8

#### valve operating mechanism

manually rotated device, which closes and opens the valve orifice

## 4 Valve requirements

### 4.1 General

Valves shall operate satisfactorily over the full range of service temperatures, from  $-20\text{ }^{\circ}\text{C}$  to  $+65\text{ }^{\circ}\text{C}$ . The range may be extended for short periods (e.g. during filling). Where higher or lower service temperatures are required for longer periods, the purchaser shall specify this.

Valves shall be capable of withstanding any mechanical stresses or chemical attack they may experience during normal service.

Valves shall be cleaned to meet the requirements of the intended service.

## 4.2 Description and dimensions

A cylinder valve comprises:

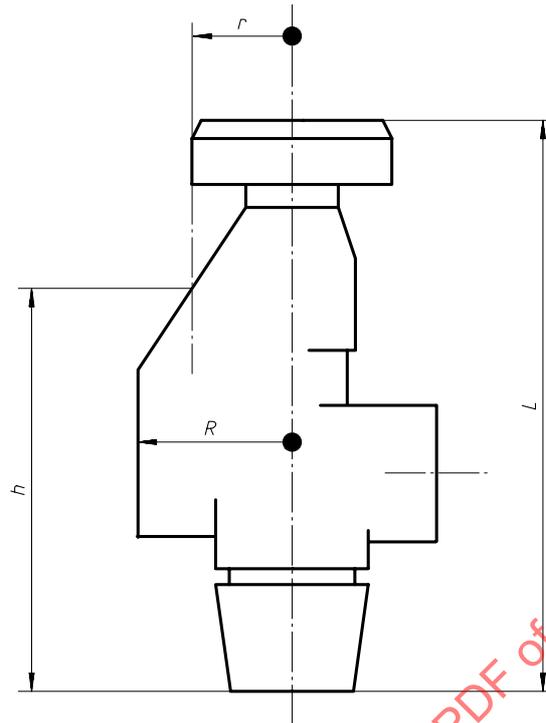
- a body;
- a valve operating mechanism and internal sealing device;
- an external sealing mechanism;
- a connection(s) for usage (fill and discharge);
- a connection system, between the valve and gas cylinder.

In addition, a valve may also comprise:

- a safety device, against overpressurization;
- a siphon tube;
- a screwed plug or cap, on the outlet connection, to ensure leak tightness or protection;
- an excess flow limiting device;
- a filter.

The bore of the valve shall be adequate to meet the requirements of flow rate, without unacceptably reducing the strength of the stem connection. The bore diameter, typically 3,5 mm for valves with a 25E stem thread (see ISO 10920) and 2 mm for valves with a 17E stem thread (see ISO 11116-1), shall be agreed between customer and supplier.

Where a valve is to be protected by a cap in accordance with ISO 11117, the valve should comply with the dimensions given in Figure 3.



$r \leq 32,5$  mm

$h \leq 90$  mm

$R \leq 38$  mm

$L \leq 125$  mm

NOTE 1  $h$  represents the height between the lower part of the valve and the point at which the overall radius of the valve is equal to the radius of the handwheel when the radius of the valve is greater than the radius of the handwheel.

NOTE 2 When the axes of the valve, the valve stem thread and handwheel do not coincide the distance between the two axes should be added to  $r$ .

NOTE 3  $L$  is the maximum length of a valve in the closed position when not fitted to a cylinder.

NOTE 4  $R$  should be measured to the part of the valve furthest from the stem axis and includes any outlet plugs or caps if fitted.

**Figure 3 — Maximum dimensions for gas cylinder valves, protected by a cap**

### 4.3 Materials

Metallic and non-metallic materials in contact with the gas shall be chemically and/or physically compatible with the gas, under all intended operating conditions (see ISO 11114-1 and ISO 11114-2).

Compatibility of materials with oxygen and other oxidizing gases, ignition resistance of materials and lubricants, shall be established by an appropriate test procedure (see ISO 11114-3).

Valves for acetylene may be manufactured from copper-based alloys if the copper content does not exceed 70 % ( $m/m$ ). The manufacturer shall not use any procedure resulting in copper enrichment of the surface. Silver content of alloys shall be limited for acetylene valves. The acceptable limit varies between 43 % ( $m/m$ ) and 50 % ( $m/m$ ), depending on the composition of the alloy.

Non-metallic sealing material for use with air, oxygen and oxygen enriched gases, shall be capable of withstanding an ageing sensitivity test in accordance with ISO 188.

Non-metallic sealing material in valves shall be capable of withstanding corrosive media tests in accordance with ISO 1817.

## 4.4 Design and construction

### 4.4.1 Valve body

The valve body shall be manufactured by a process that will ensure the reproducibility of the mechanical characteristics necessary to meet the requirements specified in this International Standard, particularly that specified in 5.4.2. The anisotropy of the material shall be considered.

### 4.4.2 Valve connections

Valves are normally connected to the cylinder by means of a taper or parallel male thread and to the filling and utilisation appliances by means of a separate outlet connection, complying with an accepted standard.

### 4.4.3 Valve operating mechanism

The valve operating mechanism shall be manufactured from materials capable of withstanding mechanical stress including possible dynamic loads (eg., pressure shocks or cyclic changes) and the extremes of service temperature, to which it may be subjected.

The materials of the valve operating mechanism shall withstand fire engulfment in accordance with 5.4.10.

The valve operating mechanism shall satisfy the following conditions:

- it shall not be dependent on the pressure in the cylinder;
- it shall, under normal conditions, operate without difficulty throughout its service life;
- it shall be designed in such a way that it cannot be unscrewed from the valve body with a torque of less than 40 N·m;
- it shall be designed in such a way that the setting of the operating position of the valve, cannot be inadvertently altered;
- it shall close the valve by clockwise rotation;
- it shall be designed to ensure that lubricants that are not oxygen compatible, do not come into contact with highly oxidizing gases as defined in ISO 10156;
- for acetylene valves, it shall be designed to meet the requirements of 5.4.9.

For valves for oxygen or highly oxidizing gases, the opening of the valve orifice shall be progressive. Complete opening shall require more than one rotation of the operating mechanism. For valves, in which it is technically difficult to limit opening in this way (for example diaphragm valves) other means shall be provided to delay full gas flow.

### 4.4.4 Tightness

Methods of achieving external tightness include:

- gland packing;
- one or several O-rings;
- diaphragm;
- bellows;
- any other appropriate device.

External and internal tightness, shall be achieved over the full range of service pressures and temperatures.

The external tightness shall be maintained for all positions of the valve spindle, from the opening, to complete closing and during operation.

All sealing devices shall withstand 2 000 opening and closing cycles, at  $p_{vt}$ , without replacement of the sealing device. Adjustment is permitted.

The minimum gauge pressure, during the tightness test, shall be 0,1 bar. Where the valve is not intended for use with inflammable or toxic gases this pressure may be increased to 0,5 bar.

At the customer's request the tightness test may be carried out under vacuum.

When incorporating diaphragms or bellows, additional gland packings or O-rings, may be used to ensure safety, in case of deterioration of the diaphragms or the bellows. This particularly applies for toxic gases.

The tightness test is normally carried out with air or nitrogen. Valves designated for use with gases lighter than air, or very searching gases (for example carbon dioxide) may be subjected to a test using helium.

For the definition of an inflammable gas see ISO 10156, and for the definition of a toxic gas see annex A of ISO 5145.

#### 4.4.5 Leakage rate

The internal or external leakage rate shall not exceed 6 cm<sup>3</sup>/h at 20 °C and 1 013 mbar.

The specified rate may be amended by agreement and subject to special applications, eg., for valves for highly toxic or high purity gas service, a lower leakage rate may be specified.

#### 4.4.6 Operating torque

For handwheel operated valves, with a handwheel diameter of 65 mm, the closing torque to obtain internal tightness, shall be 7 N·m or less. For certain valves (for example key operated or diaphragm valves) this torque may be higher. The size of the handwheel, or equivalent operating device, shall be appropriate for the required closing torque (see 5.4.3.2 and 5.4.6).

The torque necessary for the complete closing and opening of the valve, shall not increase significantly during the valve's service life (see 5.4.4).

## 5 Prototype valve test

### 5.1 General

Before valves are introduced into service, they shall be submitted for prototype approval (see 5.2 and 5.3). A prototype approval is valid for a given family of valves, having the same basic design.

Variations to connections do not require further prototype testing.

Changes to the internal components for reasons of gas/material compatibility (for example 'O' ring, packing, diaphragm, spindle, lubricant) constitute a type variant within the given family.

Type variants require repetition of the relevant parts of the type test.

Changes of the basic design dimensions of components or changes of valve body material, constitute a new family and require the full type test.

## 5.2 Documents

The manufacturer shall make available to the test authority, the following documents:

- a set of drawings consisting of the general arrangement, parts list, material specifications and detail drawings. Any type variant, within the given family, shall be clearly identified;
- description of valve and method of operation;
- information on the field of application of the valve (gases and gas mixtures, pressures, use with or without valve protection device, etc). It shall be clearly indicated which gases and gas mixtures can be used with each type variant;
- certificates of material compatibility as required.

## 5.3 Test valves

A minimum of nine sample valves are required (more samples may be necessary, depending on the number of type variants to be tested):

- a) one sample (No. 1) for the hydraulic pressure test;
- b) samples for tightness tests and endurance test as follows:
  - 1) when no type variants are specified, five samples of the basic specification shall be tested. (Nos. 2 to 6);
  - 2) when one type variant (a) is specified, three samples (Nos 2, 3, 4) of the basic specification and two samples (Nos. 5a and 6a) of the type variant shall be tested;
  - 3) when two or more type variants (a, b, etc.) are specified, two samples (Nos. 2 and 3) of the basic specification, and two samples of each type variant (Nos. 4a and 5a, 4b and 5b, etc.) shall be tested;
- c) one sample (No. 2) is also used for the fire exposure test;
- d) one sample (No. 7) for any additional test which may be required;
- e) two samples (Nos. 8 and 9) for determination of operating torques.

In addition, for oxygen or highly oxidizing gas service, three sample valves (10n, 11n and 12n) are required for the oxygen pressure surge test and for acetylene service, three sample valves (10m, 11m, and 12m) are required for the internal tightness test after flashback.

## 5.4 Test procedure

### 5.4.1 Table of tests

Tests shall be carried out, in accordance with the schedule given in Table 1.

See annex A, for an example of test sequence for a basic design with type variants.

Table 1 — Sequence of tests (tightness, endurance and fire resistance) for type approval (no variants)

Test sequence	Test and sub clause No.	Condition of test valve	Test temperature °C	Valve sample number	No. of tests/valve	Total No. of tests
1	Hydraulic 5.4.2	As received	20 ± 5	1	1	1
2	Internal/external tightness 5.4.3	As received	20 ± 5	2 <sup>a</sup> to 6	6 <sup>b</sup> or 8	30 <sup>b</sup> or 40
3	Internal/external tightness 5.4.3	From test sequence 2, aged at 65 °C for 5 days	20 ± 5	2 <sup>a</sup> to 6	6 <sup>b</sup> or 8	30 <sup>b</sup> or 40
4	Endurance 5.4.4	From test sequence 3	20 ± 5	2 <sup>a</sup> to 6	1	5
5	Internal/external tightness 5.4.3	From test sequence 4	20 ± 5	2 <sup>a</sup> to 6	6 <sup>b</sup> or 8	30 <sup>b</sup> or 40
6	Internal/external tightness 5.4.3	From test sequence 5	65 ± 2,5	2 <sup>a</sup> to 6	6 <sup>b</sup> or 8	30 <sup>b</sup> or 40
7	Internal/external tightness 5.4.3	From test sequence 6	−20 ± 2,5 <sup>c</sup>	2 <sup>a</sup> to 6	6 <sup>b</sup> or 8	30 <sup>b</sup> or 40
8	Visual examination 5.4.5	From test sequence 7	20 ± 5	2 <sup>a</sup> to 6	1	5
9	Excessive torque 5.4.6	As received	20 ± 5	8 and 9	1	2
10	Fire exposure 5.4.10	From test sequence 8	800 to 1 000	2	1	1

<sup>a</sup> For additional type variants, valve sample numbers and tests will change in accordance with 5.3.  
<sup>b</sup> The total number of tests will be 30 without vacuum test and 40 if a vacuum test is required.  
<sup>c</sup> For some parts of the world and for certain applications, tests at a lower temperature (e.g. −30 °C) shall be considered.

### 5.4.2 Hydraulic pressure test

For safety reasons this test shall be carried out prior to all other tests.

The hydraulic pressure test shall be carried out under the following conditions:

- valve seat in open position;
- valve outlet connection sealed;
- safety relief devices (where fitted) removed and the opening sealed;
- test medium, water or other suitable fluid;
- for permanent gases  $1,5 \times 1,5 p_W$ ;
- for liquefied gases  $1,5 p_{vt}$ ;
- for dissolved gases, such as acetylene, the test pressure of the valve shall be 450 bar;
- test temperature, ambient temperature (20 ± 5) °C;
- pressure holding time, 2 min minimum.

The pressure shall be raised continuously and gradually. The prototype valve shall withstand the test, without permanent deformation or rupture.

### 5.4.3 Tightness tests

#### 5.4.3.1 General

Each internal and external tightness test sequence shall comprise a test at four pressure settings as shown in Table 2.

The test pressure shall be held for not less than 1 min.

**Table 2 — Test pressures for tightness tests**

Test pressure sequence	Test pressure for tightness tests
1	Vacuum (e.g. $5 \times 10^{-3}$ bar) if required
2	0,1 bar for toxic and flammable gases, 0,5 bar for all other gases
3	10 bar for all gases
4	Whichever is higher of either $p_{vt}$ for permanent gases or the test pressure of the gas cylinder for liquefied and dissolved gases, but never less than 20 bar.

#### 5.4.3.2 Internal tightness

The internal tightness shall be determined in relation to the closing torque in accordance with the following procedure, on each of the five or more sample valves (see 5.3):

- a) blank off the valve outlet connection;
- b) remove the safety relief device (where fitted) and seal the opening;
- c) open the valve;
- d) apply the specified pressure to the valve inlet;
- e) close the valve to the desired torque;
- f) open the valve outlet connection;
- g) wait at least 1 min before measuring the seat leakage rate;
- h) if the leakage rate is unacceptable (see 4.4.5) repeat the test sequence at a higher valve closing torque.

This test sequence shall be repeated for each test pressure given in Table 2.

NOTE It is especially important that diaphragm valves be pressurized when the valve is being closed.

#### 5.4.3.3 External tightness

The external tightness shall be determined, using the following procedure, on each of the five or more sample valves (see 5.3):

- a) blank off one opening (inlet or outlet) of the valve;
- b) remove the safety relief device (where fitted) and seal the opening;
- c) fully open the valve;
- d) apply the specified pressure through the other opening;
- e) measure the leakage rate;

- f) partially close the valve;
- g) measure the leakage rate.

If required, action f) and action g) may be repeated for different partial closings.

For test sequence 7 of Table 1, carried out at  $(-20 \pm 2,5) ^\circ\text{C}$ , the external leakage rate shall also be measured while the handwheel is being rotated.

#### 5.4.4 Endurance test

An endurance test of 2 000 cycles, consisting of fully opening and closing of the valve, shall be carried out at  $p_{vt}$ .

After each closure the pressure downstream of the seat shall be released to atmosphere. There shall be a pause of at least 6 s at each fully open and fully closed position.

Care should be taken to ensure that, during the test, friction does not cause the valve temperature to significantly exceed that specified in Table 1.

For handwheel valves, the closing torque used during the test shall be 7 N·m. No significant torque shall be applied in the fully open position.

For key-operated or diaphragm valves which require a minimum closing torque ( $T_c$ ) greater than 7 N·m, the torque used during the test shall be equal to  $1,5 \times T_c$ .

For small valves, with a minimum closing torque of less than 7 N·m, the torque used during the test, shall be twice the minimum closing torque, subject to a maximum of 7 N·m.

For all subsequent tests, the torque used during the endurance test shall not be exceeded.

#### 5.4.5 Visual examination

When the endurance test and the subsequent tightness tests have been completed, sealing elements such as diaphragms, bellows, O-rings, shall be subject to a visual check for unacceptable wear and/or damage.

#### 5.4.6 Determination of resistance to excessive torque

The intent of these tests is to check that the valve operating mechanism has adequate strength, and fails safely if subjected to excessive torque.

These tests shall be carried out on the sample valves 8 and 9, at atmospheric pressure.

The closing torque on sample valve 8 shall be gradually increased to a given torque  $T$  (see below), at this torque the valve shall be able to work without noticeable difficulties and shall not show any significant damage. The torque shall then be increased slowly until failure of any part of the operating device occurs. The value of the torque, at failure, shall be not less than  $1,25 \times T$ .

This test shall then be repeated, on sample valve 9, but with an opening torque instead of a closing torque.

After this test, the valve operating mechanism may be severely damaged and not working. No pressure-retaining components shall have failed. Disassembly of the mechanism is not permitted during this test.

The value of  $T$  for standard industrial gas cylinder valves, fitted with a 65 mm diameter handwheel, is 20 N·m.

This value of  $T$  will vary however with the design of valve and operating mechanism, it may, for small valves, be lower and for key-operated valves be greater.

The operating mechanism shall not be capable of withstanding a torque in excess of that required to disassemble the operating mechanism from the valve body (see 4.4.3).

### 5.4.7 Mechanical impact test

If it is not intended for the valve to be protected by a cap or another form of guarding, an impact test shall be carried out on valve sample 7, as described in annex B.

### 5.4.8 Oxygen pressure surge test

This test shall be carried out for valves used in all applications where the gas or gas mixture has an oxidizing potential greater than that of air (for the definition of oxidizing potential see ISO 10156). For all types of valve the pressure surge test shall be carried out with pure oxygen.

The purpose of the test is to check whether the valve safely withstands an oxygen pressure surge.

Three sample valves, samples 10 to 12, in the "as-received" condition (or lubricated if a lubricant is used for such a valve) shall be tested.

Before the test, the ignition test installation shall be checked for the required pressure rise (for examples of test installation and pressure cycle specification, see Figures 4 and 5). For this purpose the test valve, at the end of the 1 m length of copper tube, is replaced by a reliable pressure gauge.

The maximum pressure at the dead end of the copper tube or tube in monel or any other material with equivalent resistance to oxygen (measured by pressure gauge and recorded on an oscilloscope) shall be achieved within  $(20_{-5}^0)$  ms (time necessary to reach  $p_{vt}$  starting from atmospheric pressure).

Stabilization time at  $p_{vt}$  is not fixed but shall be greater than or equal to 3 s. Before the next pressure surge the system (sample valve and copper tube) shall be depressurized down to atmospheric pressure. Stabilization time at atmospheric pressure is not fixed but shall be greater than, or equal to 3 s.

The total time of the pressure cycle shall be 30 s, as illustrated in Figure 5. Total time is the time between the beginning of two consecutive pressure surges.

For calibration purposes, heated oxygen at  $(60 \pm 3)$  °C shall be used.

The quality of oxygen shall be:

- minimum purity 99,5 % (V/V);
- hydrocarbon content  $\leq 0,001$  % (V/V).

Each test shall be carried out as follows:

- supply oxygen at a temperature of  $(60 \pm 3)$  °C, directly into the connection of the valve to be tested, by means of a copper tube having an internal diameter of 5 mm and a length of 1 m. The specified material and dimensions of the tube, are essential in order to ensure that a well-defined energy input into the valve to be tested is achieved;
- two test sequences shall be carried out in accordance with Table 3;

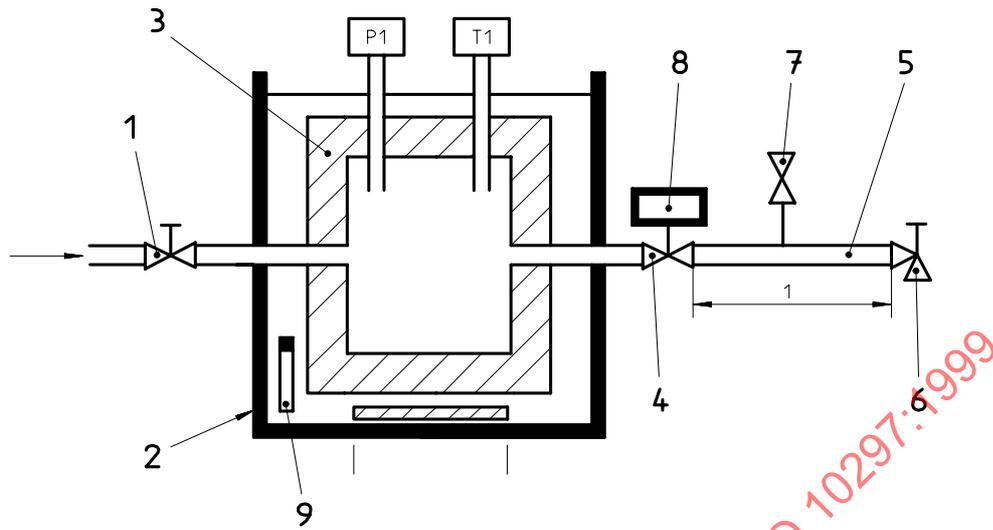
**Table 3 — Test sequence**

Test sequence	Valve operating system	Valve stem
1	Closed	Open
2	Open	Sealed with a screwed metallic plug

- oxygen is heated up to  $(60 \pm 3)$  °C, in the oxygen pre-heater. Flow of oxygen, to the test sample valve, is controlled by a quick opening valve (see Figure 4). The test consists of subjecting the sample valve to 20 pressure cycles from atmospheric pressure to the valve test pressure ( $p_{vt}$ ) (see Figure 5).

After the tests, the sample valve shall be dismantled and carefully checked, including close examination of non-metallic components. It shall not show any traces of ignition.

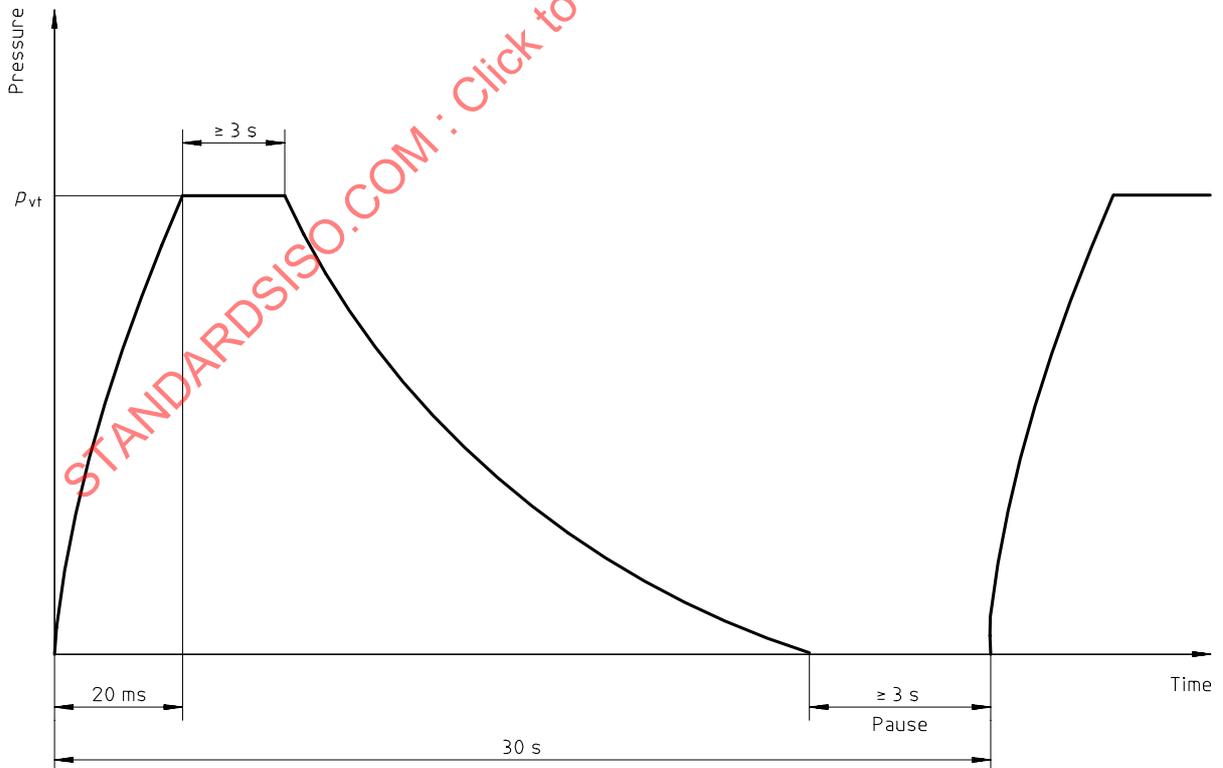
Dimensions in metres



**Key**

- |  |                          |                      |
|--|--------------------------|----------------------|
| 1 Inlet valve  | 4 Quick opening valve    | 8 Actuator           |
| 2 Pre-heating device (e.g. water bath with electric heating) | 5 Copper tube            | 9 Thermostat         |
| 3 Oxygen vessel  | 6 Sample valve           | P1 Pressure gauge    |
|  | 7 Depressurization valve | T1 Temperature gauge |

**Figure 4 — Example of an ignition test installation**



**Figure 5 — Pressure cycle specification**

#### 5.4.9 Acetylene flashback test

This test is for valves intended for acetylene service.

The purpose of the test is to establish whether the sample valve is able to withstand an acetylene flashback. After the flashback test, it shall be possible to close the valve.

Where the sample valve is fitted with an integral pressure relief device, this device shall be closed for the test.

The acetylene cylinder used for the test, shall have a free volume at the top of the cylinder of approximately 150 cm<sup>3</sup>.

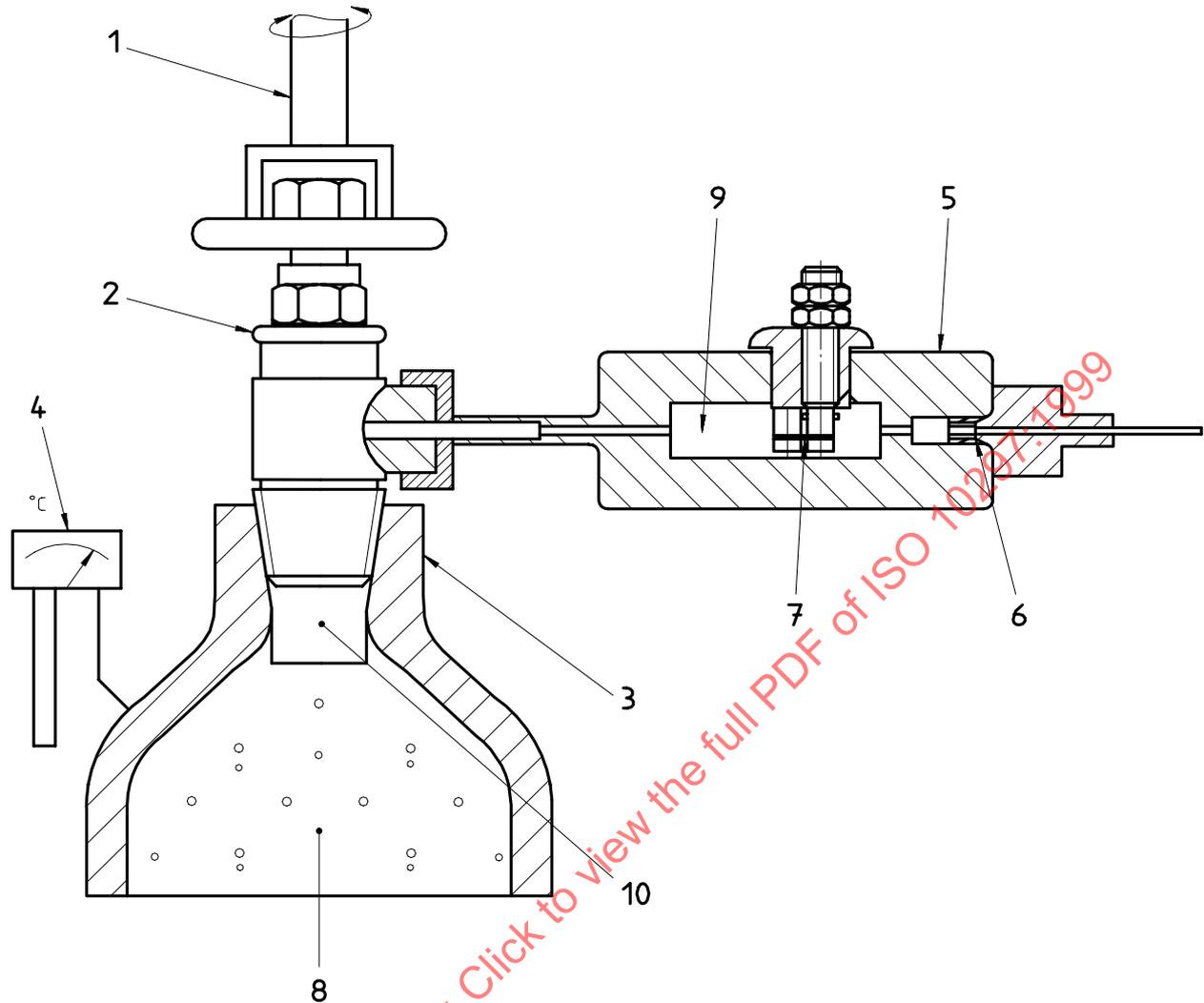
The sample valve shall be screwed into an acetylene cylinder of 5 l water capacity (prepared with porous mass and solvent). No neck filter shall be fitted to the cylinder or to the valve.

The cylinder shall be filled with at least half of its maximum permissible quantity of acetylene. An igniter tube of volume 30 cm<sup>3</sup> is connected to the outlet boss (see Figure 6). This igniter tube is closed at one end by a bursting disc having a maximum burst pressure of 40 bar. The decomposition of the acetylene propagates into the cylinder and, due to a pressure rise, causes the rupture of the bursting disc. Hot decomposition gases flow out of the valve.

After 30 s the sample valve shall be closed from a safe distance (i.e. by remote control).

The cylinder shall be left until stable (approximately 24 h). The internal tightness of the valve shall then be checked, and the leakage rate shall not exceed 50 cm<sup>3</sup>/h.

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**Key**

- |                                    |   |
|------------------------------------|---|
| 1 Remotely operated closing device | 6 Bursting disc                           |
| 2 Sample valve                     | 7 Constantan wire                         |
| 3 Acetylene cylinder               | 8 Porous mass                             |
| 4 Temperature indicator            | 9 Approximate volume 30 cm <sup>3</sup>   |
| 5 Igniter tube                     | 10 Approximate volume 150 cm <sup>3</sup> |

**Figure 6 — Example of acetylene flashback test apparatus**

#### 5.4.10 Fire exposure test for valve operating mechanism

The sample valve operating mechanism (handwheel) shall be exposed for 1 min to a gas torch flame of length 150 mm, without additional air supply, such that the flame reaches a temperature of 800 °C to 1 000 °C. The operating mechanism shall be completely enveloped by the flame.

Although the operating mechanism may be damaged during the test, the valve shall still be capable of being closed manually after sufficient cooling.

## 6 Marking

Gas cylinder valves, conforming to this International Standard, shall have the following marking:

- a) the number of this International Standard, i.e. ISO 10297;
- b) the manufacturer's designation;
- c) the year and month of manufacture;
- d) the identification of the cylinder/valve connection.

In addition for permanent gases, the working pressure shall be marked. Further additional markings may be specified as required (e.g. cap or guard not required).

## 7 Test report

A written report shall be prepared summarizing all the tests carried out and the results obtained.

This report shall be signed by the responsible person(s) of the testing laboratory and shall include drawings, parts, lists, material certificates, etc.

The report shall be obtainable from the valve manufacturer on request.

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## Annex A (informative)

### Example of test sequence

One design of test valve with three different 'O' ring material specifications and three different outlet specifications.

'O' ring specifications:

- a) O<sub>2</sub> service – fluorocarbon (FKM);
- b) C<sub>2</sub>H<sub>2</sub> service – ethylene propylene (EPDM);
- c) CO<sub>2</sub> service – nitrile rubber (NBR).

**Table A.1 — Sequence of tests for type approval (basic design plus two type variants)**

Basic design O <sub>2</sub> service		Type variant a) C <sub>2</sub> H <sub>2</sub> service different outlet connection		Type variant b) CO <sub>2</sub> service different outlet connection	
Test sequence	Sample valve number	Test sequence	Sample valve number	Test sequence	Sample valve number
1	1	—	—	—	—
2	2 and 3	2	4a and 5a	2	4b and 5b
3	2 and 3	3	4a and 5a	3	4b and 5b
4	2 and 3	4	4a and 5a	4	4b and 5b
5	2 and 3	5	4a and 5a	5	4b and 5b
6	2 and 3	6	4a and 5a	6	4b and 5b
7	2 and 3	7	4a and 5a	7	4b and 5b
8	2 and 3	8	4a and 5a	8	4b and 5b
9	8 and 9	—	—	—	—
10	2	—	—	—	—
Plus oxygen pressure surge test	10, 11 and 12	Plus acetylene flashback test	10a, 11a and 12a	—	—

## Annex B (normative)

### Valve impact test

In circumstances where cylinder valves are used in cylinders of water capacity 5 l, or greater, and where valve protection is not intended to be fitted, the following test shall be carried out. The purpose of this test is to ensure that the valve has sufficient inherent strength to withstand impacts which may occur in transit.

One valve, in the closed condition (closed to the torque used in the endurance test, in accordance with 5.4.4) shall be fitted into a gas cylinder neck equipped with the corresponding screw thread or a similar specialized fixture (see Figure B.1), the tightening torque being the same as that as used in service.

The valve shall protrude from the cylinder neck or fixture by the same nominal amount as in service.

The valve shall be struck by a 13 mm diameter hardened steel ball, which has a minimum velocity of 3 m/s and an impact energy, as given in Table B.1. The impact shall be at 90° to the longitudinal axis of the valve and co-incident with a plane passing through the same axis.

The point of impact shall be two thirds the distance ( $L$ ) from the plane where the valve stem thread meets the cylinder, to the furthest point of the valve body, measured along the longitudinal axis of the valve (see Figure B.1).

The point of impact on the valve shall not be obscured by features such as outlet connecting threads, bursting disc holders, hand wheels, etc.

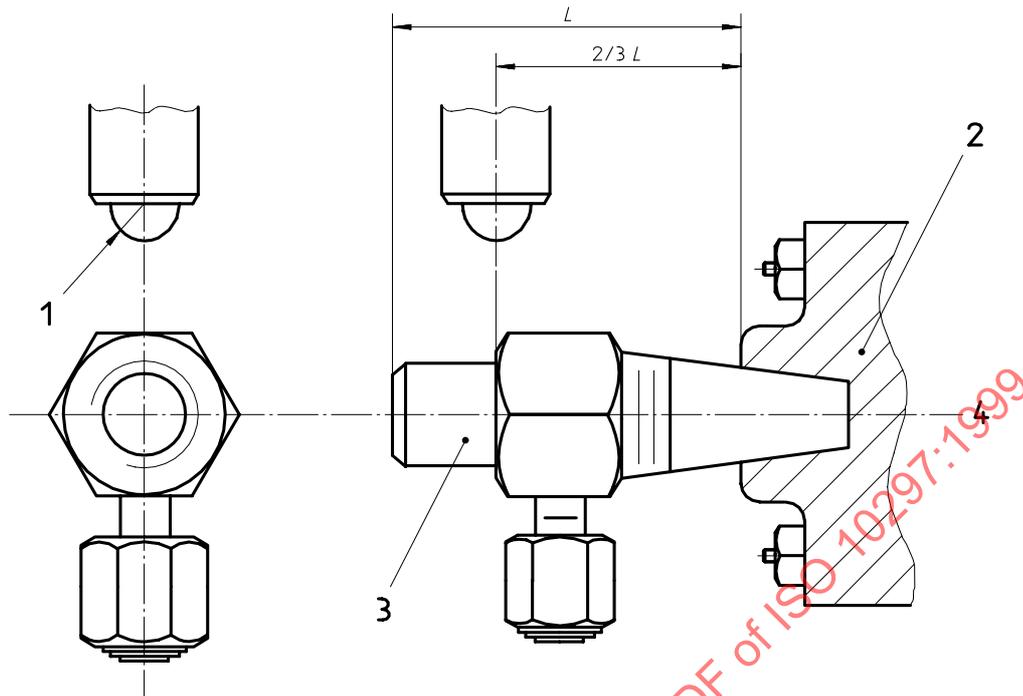
The valve shall be struck once only, and shall withstand the appropriate impact energy as given in Table B.1. Distortion due to impact is permissible.

After testing, the valve shall be removed from the test rig, it shall then be fitted into a pressure supply and closed to the previous torque.

Working pressure ( $p_w$ ) shall be applied to the valve inlet. The leakage shall conform to 4.4.5.

**Table B.1 — Impact energy**

Cylinder water capacity, $V$	Impact energy
l	J
$5 \leq V < 60$	200
$60 \leq V \leq 150$	300

**Key**

- 1 13 mm diameter hardened steel ball
- 2 Fixture or cylinder
- 3 Valve
- 4 Longitudinal axis

**Figure B.1 — Impact test**

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## Annex C (normative)

### Endurance test

#### C.1 Test valves

Valves to be tested shall be at room temperature ( $20\text{ °C} \pm 5\text{ °C}$ ). They shall have been aged in accordance with the procedure specified in Table 1.

#### C.2 Test medium

The endurance test shall be carried out with dry air or nitrogen filtered to at least  $20\text{ }\mu\text{m}$  and at a dew point of less than  $-40\text{ °C}$  at atmospheric pressure.

NOTE If nitrogen is used the risks of asphyxiation should be considered if a major leak occurs.

Tests shall not be carried out in a waterbath or other liquid medium.

#### C.3 Test machine

##### C.3.1 Equipment

Figure C.1 shows a typical arrangement of computer controlled equipment.

##### C.3.2 Speed and application of torque

The test machine shall be able to open and close the test valves at a speed between  $0,167\text{ s}^{-1}$  and  $0,5\text{ s}^{-1}$  (10 rpm and 30 rpm).

At the end of the closing part of the test cycle, overtorque due to dynamic effects shall be no more than 10 % of the set figure.

##### C.3.3 Alignment

The valve and the machine spindles shall be aligned in such a way that no significant side or axial load is put on the valve during the test.

##### C.3.4 Calibration

The calibration of the machine shall be verified before commencing and after completion of each endurance test.